

Computer-Based Compartment Fire Modeling

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Compartment Fire Modeling

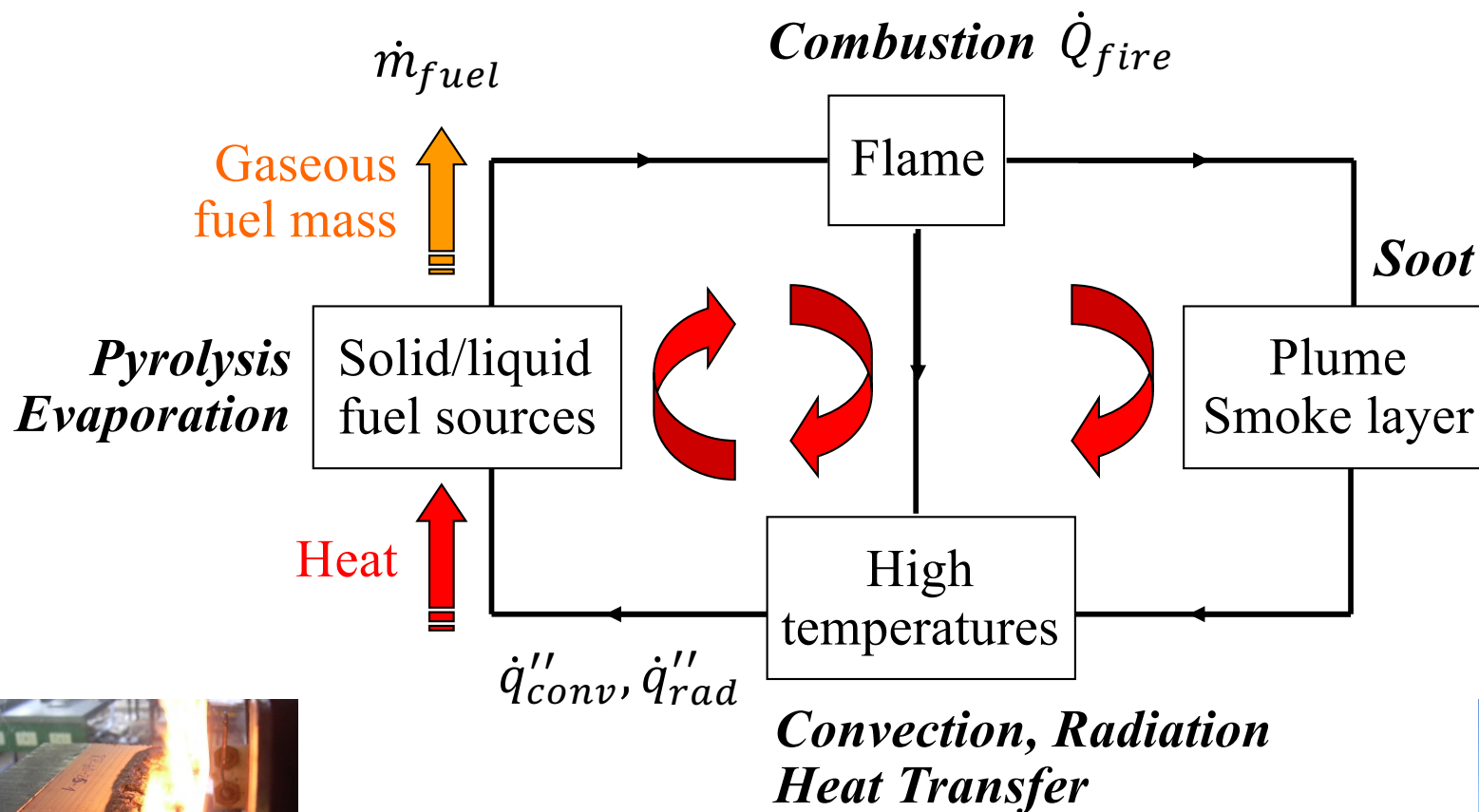


- Outline
 - **Brief Review of Compartment Fire Dynamics**
 - Fire Modeling Landscape
 - Computational Infrastructure
 - Physical Modeling
 - Examples

Fire Dynamics



- **Main features:** fire is an uncontrolled combustion process characterized by a thermal feedback loop



Fire Dynamics

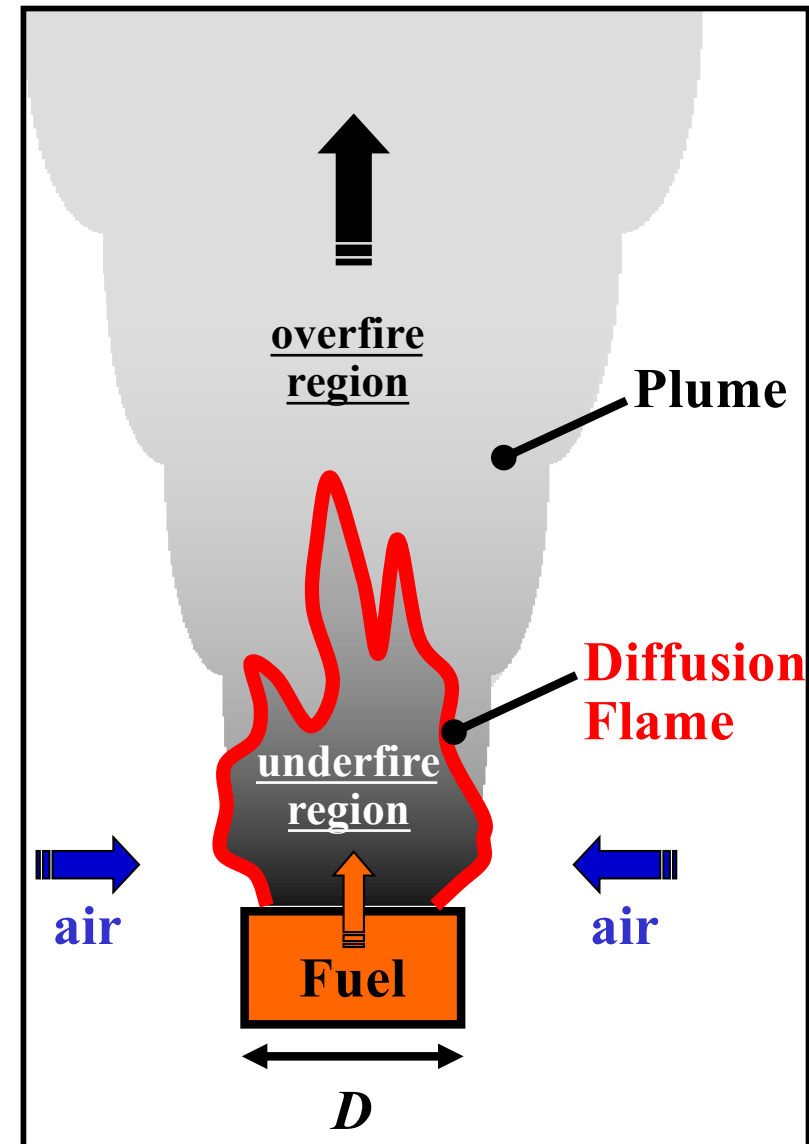


- **Main features:** fire is a buoyancy-driven, relatively-slow, non-premixed combustion process

➤ *Example:* pool fire configuration

- Fuel source velocity is small (a few cm/s)
- Buoyancy effects accelerate the flow up to several m/s; flow regime corresponds to moderate turbulence intensities
- Flame corresponds to diffusion combustion and to a thin reaction sheet where fuel and air meet in stoichiometric proportions
- Long residence times and large length scales promote soot formation and radiation losses

$$\chi_{rad} = (\dot{Q}_{rad}/\dot{Q}_{fire}) \sim 0.3$$



Fire Dynamics



- **Main features:** in compartment fires, combustion can evolve to under-ventilated (*i.e.*, fuel-rich) conditions

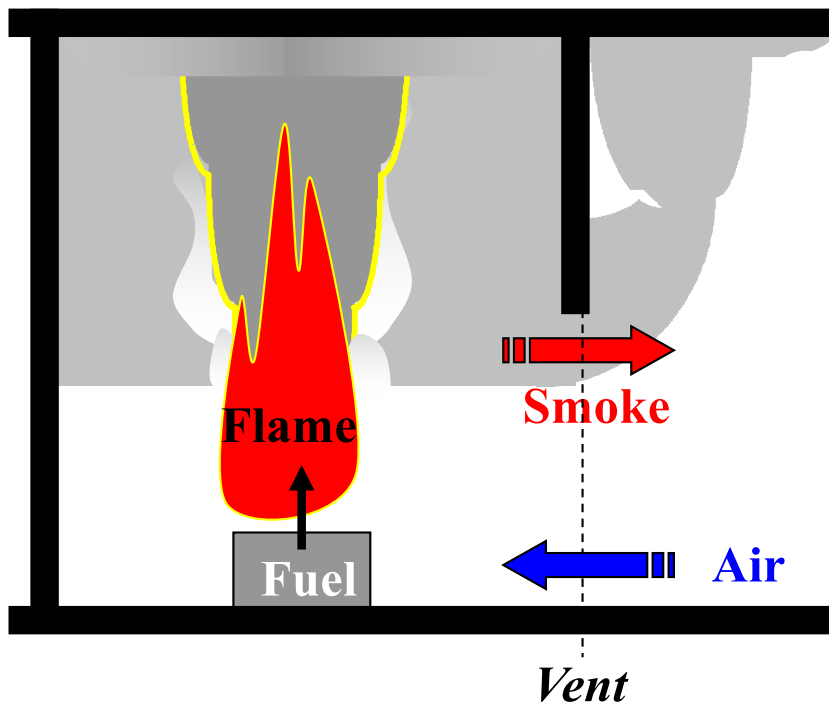


Flames extending out of the compartment of fire origin

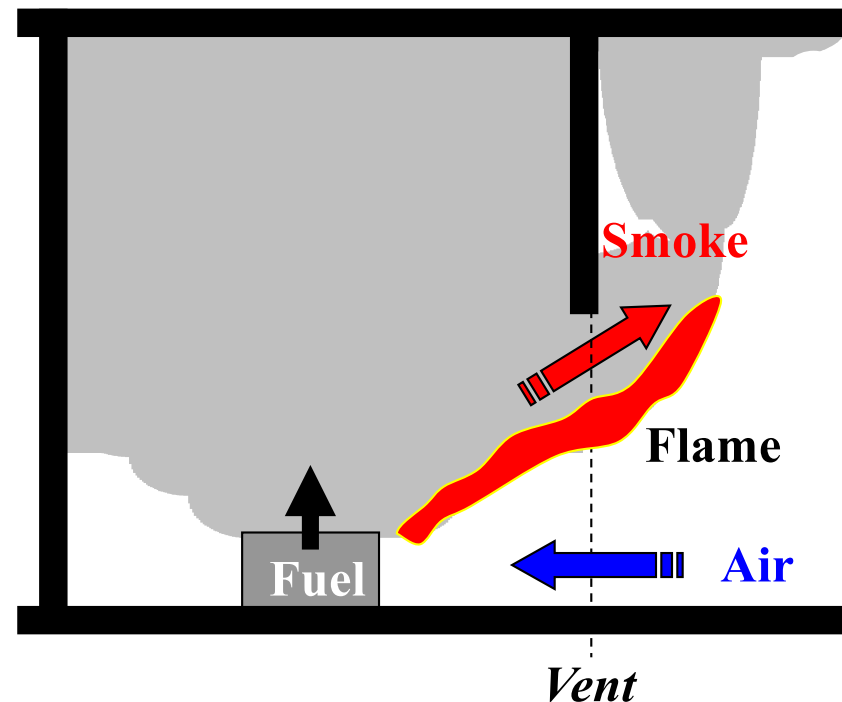
Fire Dynamics



- **Main features:** in compartment fires, combustion can evolve to under-ventilated (*i.e.*, fuel-rich) conditions
 - Flame location: (1) near the fuel source; (2) near the vents



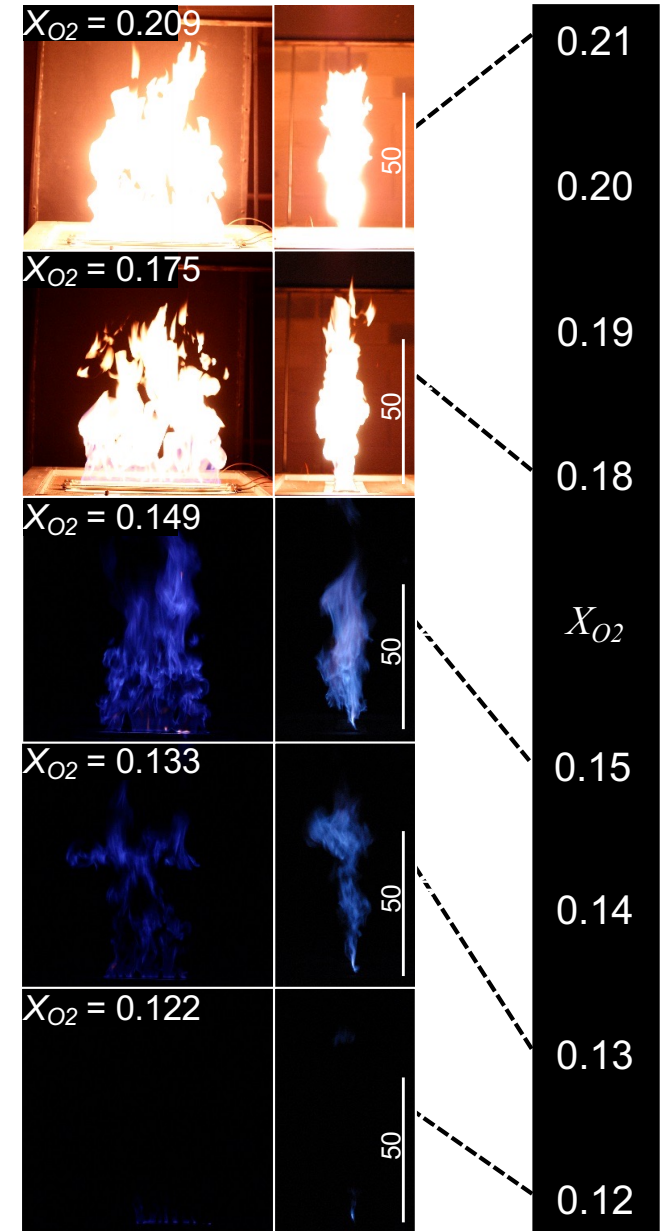
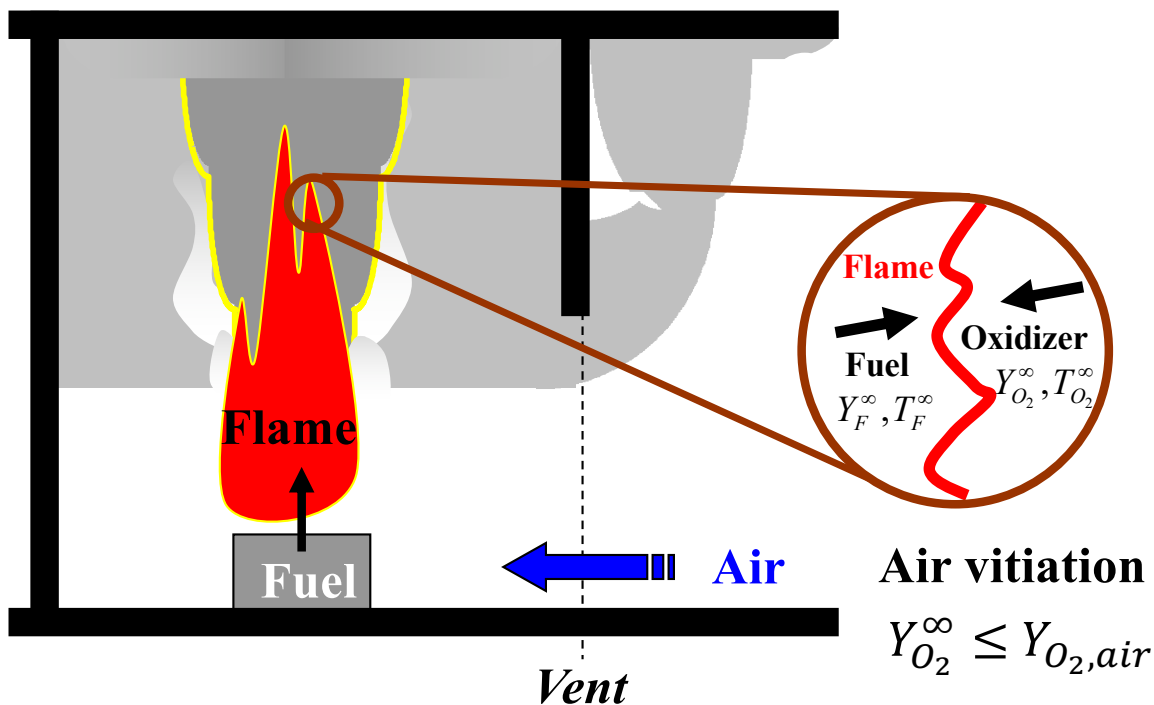
(1) Over-ventilated combustion,



(2) Under-ventilated combustion

Fire Dynamics

- **Main features:** in compartment fires, combustion can evolve to under-ventilated (*i.e.*, fuel-rich) conditions
 - Oxygen starvation reduces the flame intensity and promotes flame extinction



Fire Dynamics



- **Main features:** in compartment fires, radiation plays a dominant role in the thermal feedback to fuel sources

$$G_{UL} = \varepsilon_{UL} \times (\sigma T_{UL}^4)$$

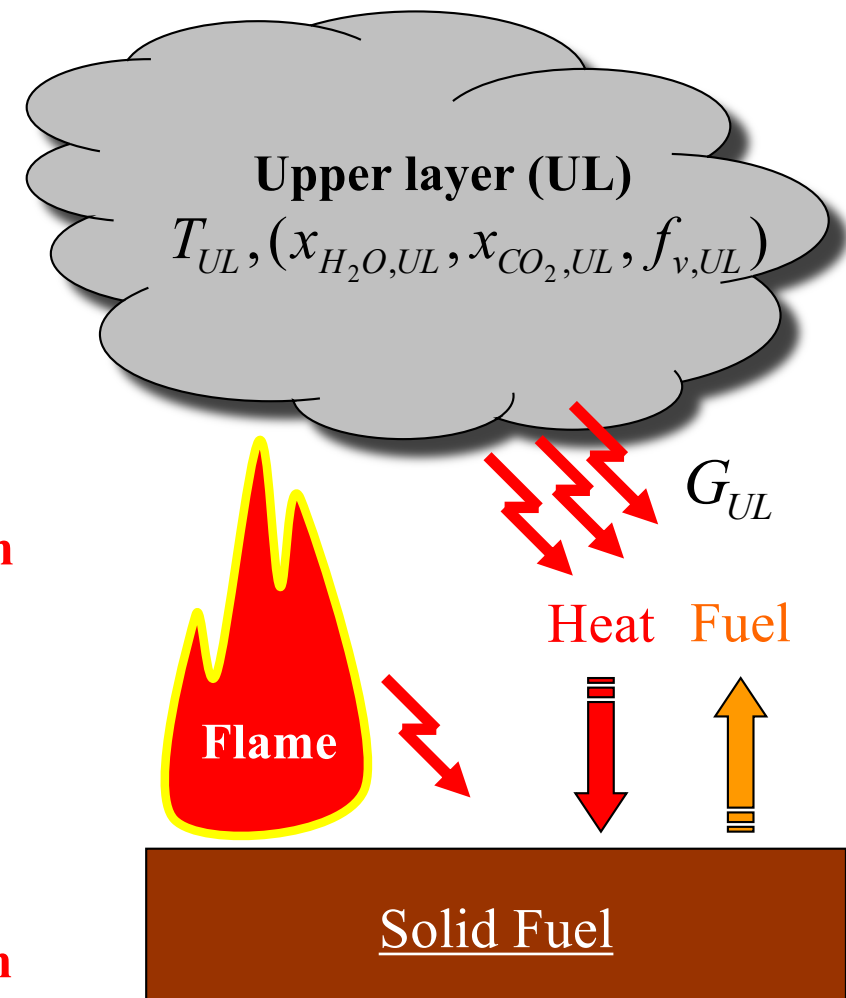
emissive power **emissivity**

$$\varepsilon_{UL} = 1 - \exp(-\kappa_{UL} \times d_{UL})$$

Planck mean absorption coefficient **mean beam length**

$$\kappa_{UL}(T_{UL}, (x_{H_2O,UL}, x_{CO_2,UL}, f_{v,UL}))$$

gas radiation **soot radiation**



Fire Dynamics



- **Main features:** in compartment fires, radiation plays a dominant role in the thermal feedback to fuel sources
 - Possible transition to *flashover* (rapid series of radiation-driven ignition events involving all flammable objects/materials present in the fire room)

Small fire



**Flammable
objects**

time
→

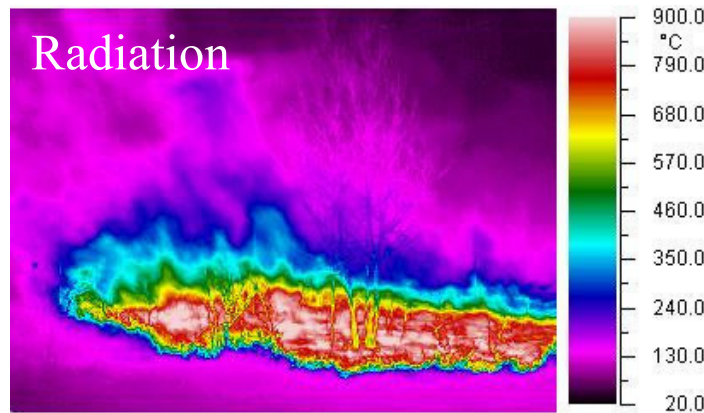
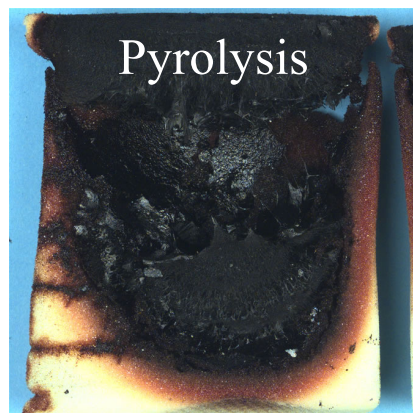
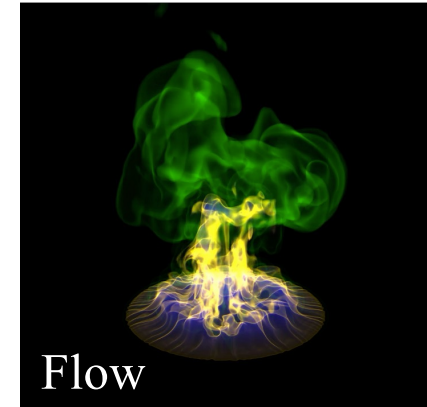
Flashover !



Fire Dynamics



- **Main features:** fire is an uncontrolled combustion process characterized by a thermal feedback loop
 - Buoyancy-driven turbulent flow
 - Non-premixed combustion
 - Including possible oxygen-limited conditions leading to flame extinction/re-ignition phenomena
 - Including soot formation
 - Thermal radiation
 - Including possible transition to flashover
 - Pyrolysis



Fire Dynamics



- Three different components in a computational model aimed at fire applications
 - Computational Fluid Dynamics (CFD) flow/combustion solver
 - Heat release by combustion and heat transfer by convection
 - Radiation solver
 - Heat transfer by radiation (electromagnetic energy)
 - Solid phase pyrolysis solver
 - Heat transfer by conduction and possible thermal degradation of materials

Compartment Fire Modeling

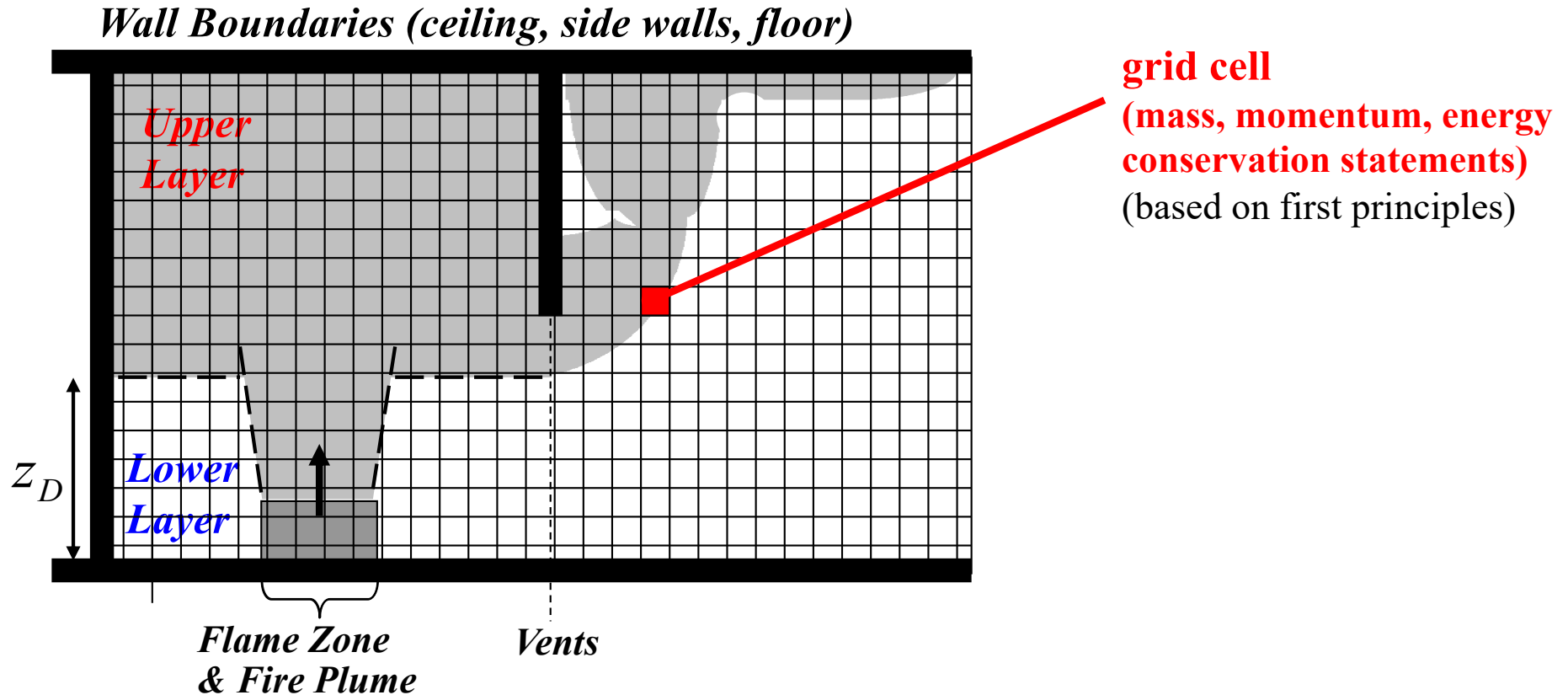


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 - Brief Review of Compartment Fire Dynamics
 - **Fire Modeling Landscape**
 - Computational Infrastructure
 - Physical Modeling
 - Examples

Fire Modeling Landscape



- CFD modeling
 - A spatially-resolved description of compartment fires



Fire Modeling Landscape



- CFD modeling

- History

- An approximately 25-years-old activity
 - Widespread use by different fire safety stakeholders (including researchers and practicing engineers)

- Landscape

- No commercial software
 - Software with limited distribution: JASMINE (Building Research Establishment, UK), KAMELEON (Norwegian University of Science and Technology/SINTEF, Norway), SMARTFIRE (University of Greenwich, UK), SOFIE (University of Cranfield, UK)
 - Open-source software: FDS (NIST, USA), FireFOAM (FM Global, USA), ISIS (IRSN, France)

Fire Modeling Landscape



- CFD modeling

- Applications

- Performance-based design (compartmentation performance, evacuation performance, smoke management, fire suppression systems, structural resistance, *etc*)
 - Forensic applications
 - Risk analysis
 - Fire-fighter training
 - Sensor-driven real-time emergency management
 - Research (scientific studies of fire dynamics)

Compartment Fire Modeling



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- CFD-based fire modeling (field modeling)
 - A branch in a wider class of simulation tools known as Computational Fluid Dynamics (CFD)
 - CFD infrastructure requires:
 - **Mathematical models** to describe relevant physics
 - ✓ First principles (conservation of mass, momentum, energy)
 - **Numerical algorithms** to solve mathematical models
 - ✓ Partial Differential Equations (PDE) solvers
 - ✓ Mesh generators
 - **Computer power** to enable numerical algorithms
 - ✓ Massively parallel computers
 - ✓ Graphics Processing Unit (GPU) computers



- Mathematical modeling (Direct Numerical Simulation – DNS)

$$\frac{\partial \rho}{\partial t} + \frac{\partial}{\partial x_j}(\rho u_j) = 0$$

$$\frac{\partial}{\partial t}(\rho Y_k) + \frac{\partial}{\partial x_j}(\rho Y_k u_j) = \frac{\partial}{\partial x_j}(\rho D_k \frac{\partial Y_k}{\partial x_j}) + \dot{\omega}_k, \quad 1 \leq k \leq N_s$$

$$\frac{\partial}{\partial t}(\rho u_i) + \frac{\partial}{\partial x_j}(\rho u_i u_j) = -\frac{\partial p}{\partial x_i} + \frac{\partial \tau_{ij}}{\partial x_j} + \rho g_i, \quad 1 \leq i \leq 3$$

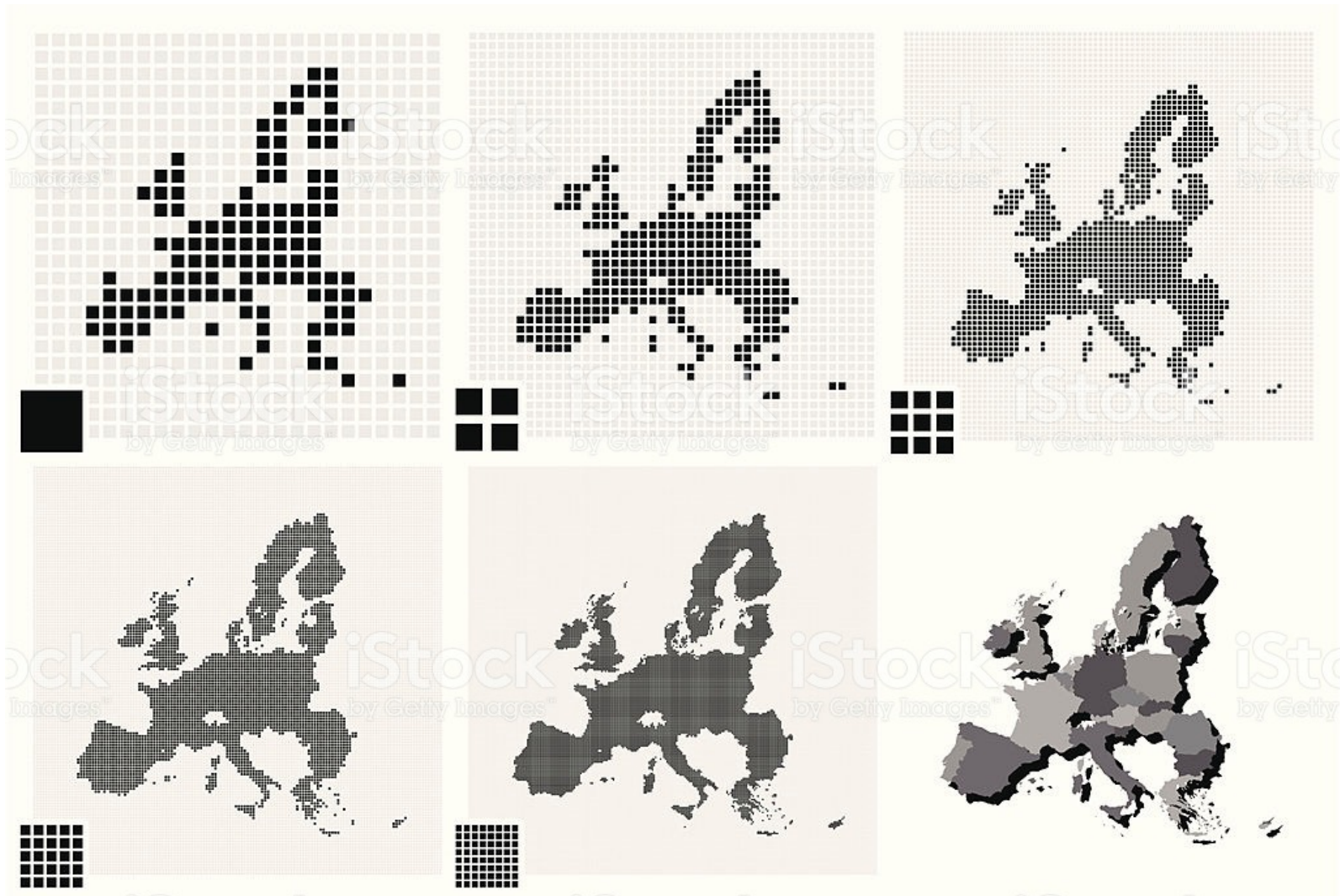
$$\frac{\partial}{\partial t}(\rho h) + \frac{\partial}{\partial x_j}(\rho h u_j) = \frac{\partial p}{\partial t} + u_j \frac{\partial p}{\partial x_j} + \tau_{ij} \frac{\partial u_i}{\partial x_j} - \frac{\partial q_j}{\partial x_j}$$

$$p = \rho R T \left(\sum_{k=1}^{N_s} \frac{Y_k}{M_k} \right)$$

Computational Infrastructure



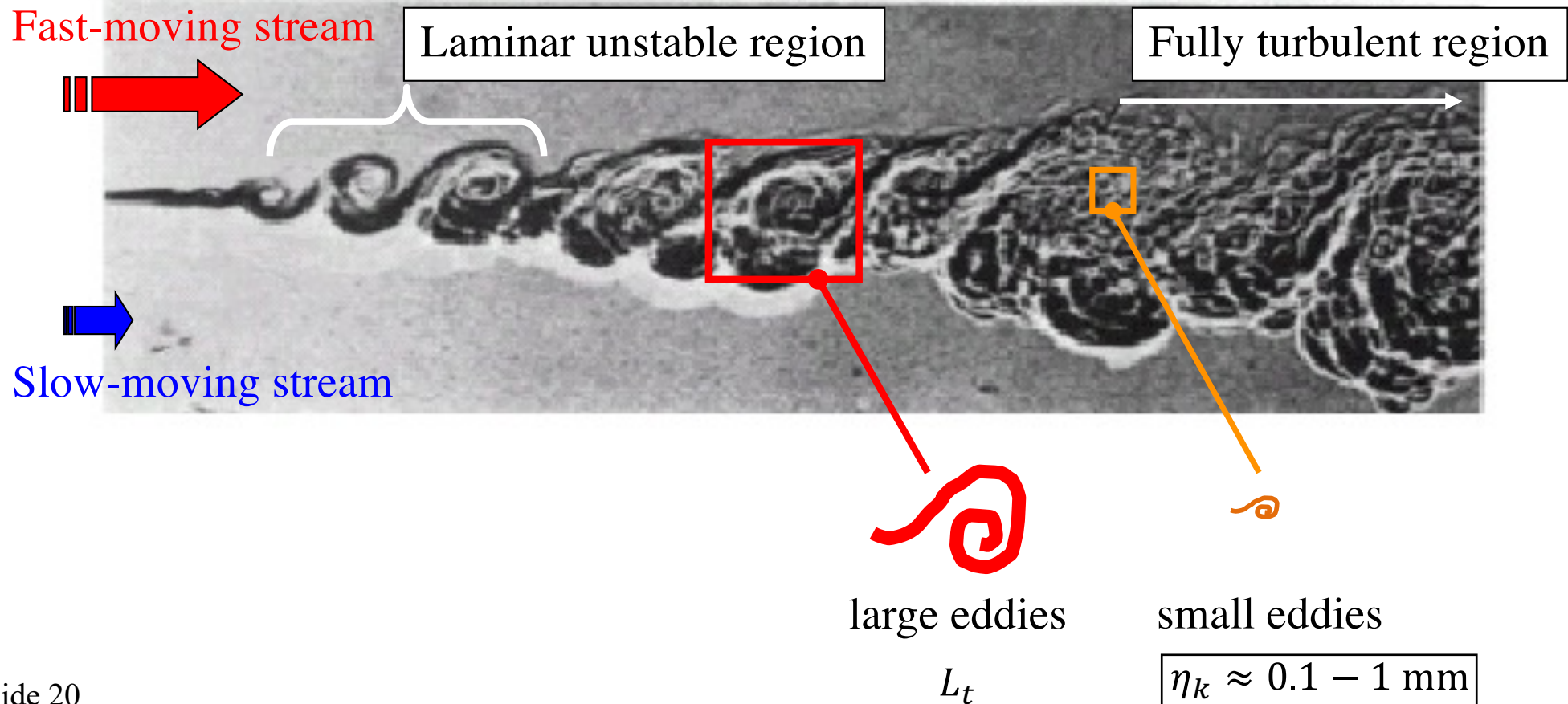
- Resolution requirements
 - How many pixels (*i.e.* computational cells)?



Computational Infrastructure



- Mathematical modeling (DNS)
 - Characteristic length scales
 - Turbulence viewed as a multi-scale problem





- Mathematical modeling (DNS)

- Computational grid requirement

- Large eddies (macro-scales)

- ✓ Turbulent rms velocity (m/s): u'

- ✓ Integral length scale (m): L_t

- ✓ Turbulent Reynolds number:

$$\text{Re}_t = \frac{\rho u' L_t}{\mu} = \frac{u' L_t}{\nu}$$

- Small eddies (micro-scales, also called Kolmogorov scales)

- ✓ Kolmogorov velocity (m/s):

$$v_K = u' \times (\text{Re}_t)^{-1/4}$$

- ✓ Kolmogorov length scale (m):

$$\eta_K = L_t \times (\text{Re}_t)^{-3/4}$$



- Mathematical modeling (DNS)

- Computational grid requirement

- Example: pool fire, $\dot{Q} = 1$ MW; $D = 1$ m

$$\left. \begin{aligned} \bar{u}_{CL,\max} &\approx 1.9 \times (\dot{Q} / 1000)^{1/5} = 7.6 \text{ m/s} \\ u' &\approx 0.3 \times \bar{u}_{CL,\max} = 2.3 \text{ m/s} \\ L_t &\approx 0.5 \times D = 0.5 \text{ m} \end{aligned} \right\} \Rightarrow \text{Re}_t = \frac{u' L_t}{\nu} \approx \frac{2.3 \times 0.5}{10^{-4}} \approx 11500$$

$$\Rightarrow \eta_K = \frac{L_t}{(\text{Re}_t)^{3/4}} = \frac{0.5}{(11500)^{3/4}} = 0.4 \text{ mm}$$

$$\Delta x_{DNS} \approx 0.4 \text{ mm}$$

Grid requirement based on flow



- Mathematical modeling (DNS)
 - Characteristic length scales
 - Combustion viewed as a stiff problem

Lecoustre *et al.* (2014) *Combust. Flame*

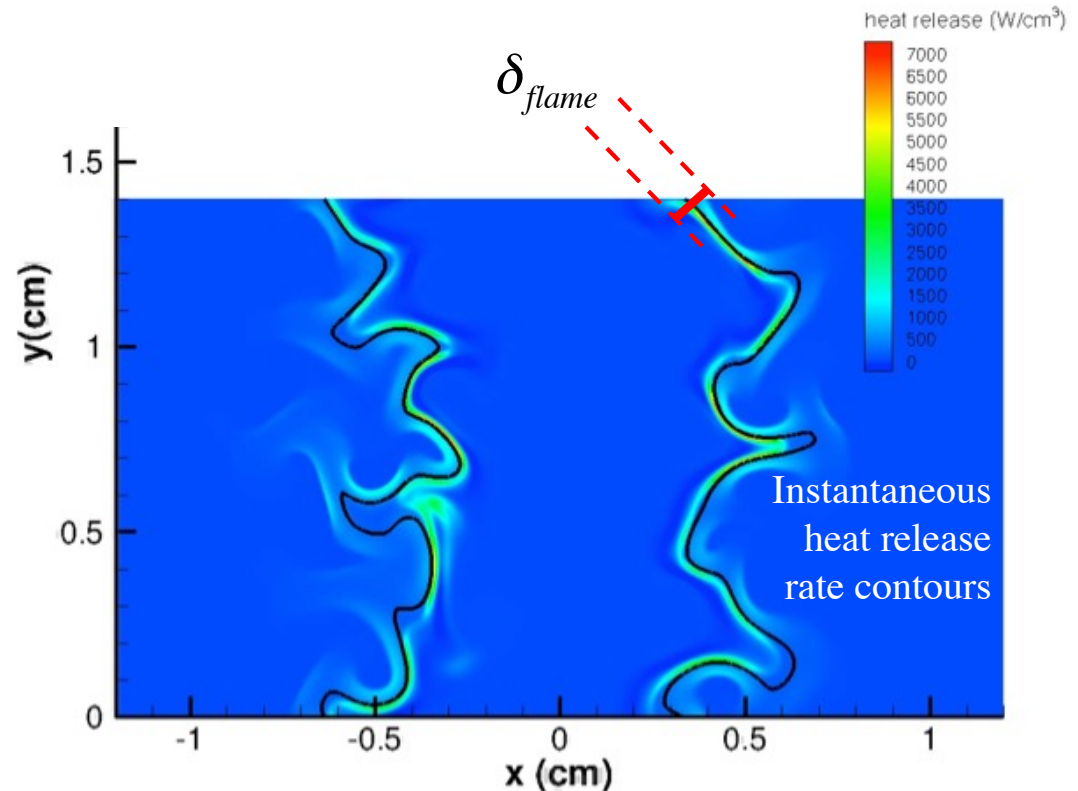
Strained laminar diffusion
flame theory

$$\delta_{flame} \sim \sqrt{D_{th,st}/\chi_{st}}$$

$$\delta_{flame} \approx 1 \text{ mm}$$

$$\Delta x_{DNS} \approx 0.1 \text{ mm}$$

**Grid requirement
based on flame**



Computational Infrastructure



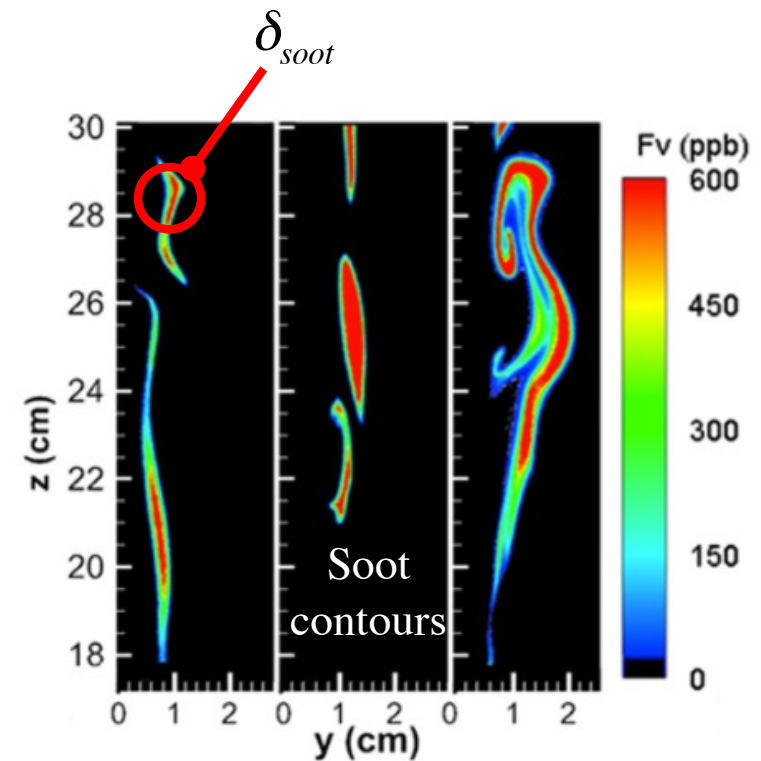
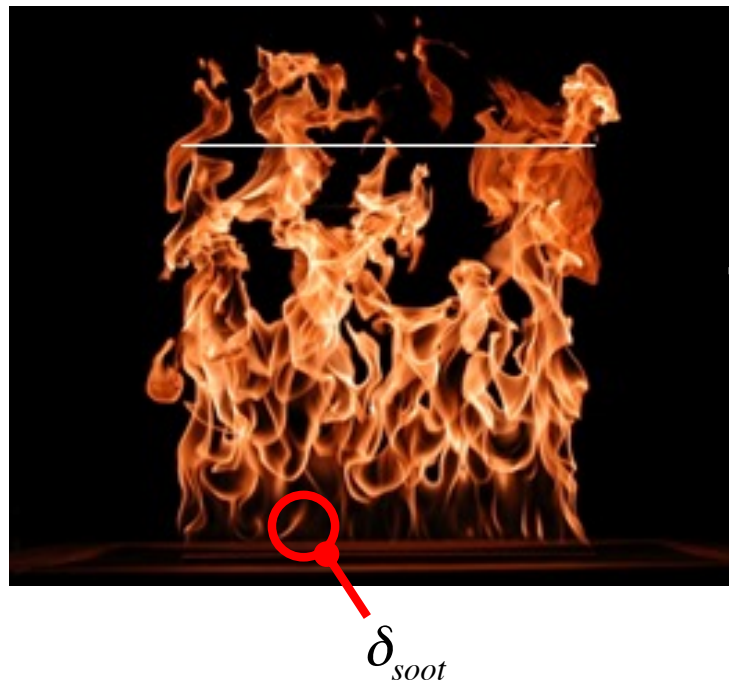
- Mathematical modeling (DNS)
 - Characteristic length scales
 - Thermal radiation viewed as a stiff problem

Experimental observations

$$\delta_{soot} \approx 1 \text{ mm}$$

$$\Delta x_{DNS} \approx 0.1 \text{ mm}$$

Grid requirement based on radiation



Valencia *et al.*
Proc. Combust. Inst. 2016



- Mathematical modeling (Direct Numerical Simulation – DNS)
 - Computational grid requirement

- Grid-resolved scales: $L_t, \eta_K, \delta_{flame}, \delta_{soot}$

$$\Delta x_{DNS} \approx \eta_K$$

$$\Delta x_{DNS} \approx (\delta_{flame}/10)$$

$$\Delta x_{DNS} \approx (\delta_{soot}/10)$$

$$\Rightarrow \Delta x_{DNS} = O(0.1 \text{ mm})$$



- Mathematical modeling (Large Eddy Simulation – LES)

$$\frac{\partial \bar{\rho}}{\partial t} + \frac{\partial}{\partial x_j} (\bar{\rho} \tilde{u}_j) = 0$$

$$\frac{\partial}{\partial t} (\bar{\rho} \tilde{Y}_k) + \frac{\partial}{\partial x_j} (\bar{\rho} \tilde{Y}_k \tilde{u}_j) = - \frac{\partial \lambda_{kj}}{\partial x_j} + \frac{\partial}{\partial x_j} \left(\overline{\rho D_k} \frac{\partial Y_k}{\partial x_j} \right) + \overline{\dot{\omega}_k'''} , \quad 1 \leq k \leq N_s$$

$$\frac{\partial}{\partial t} (\bar{\rho} \tilde{u}_i) + \frac{\partial}{\partial x_j} (\bar{\rho} \tilde{u}_i \tilde{u}_j) = - \frac{\partial T_{ij}}{\partial x_j} - \frac{\partial \bar{p}}{\partial x_i} + \frac{\partial \overline{\tau_{ij}}}{\partial x_j} + \bar{\rho} g_i, \quad 1 \leq i \leq 3$$

$$\frac{\partial}{\partial t} (\bar{\rho} \tilde{h}) + \frac{\partial}{\partial x_j} (\bar{\rho} \tilde{h} \tilde{u}_j) = - \frac{\partial Q_j}{\partial x_j} + \frac{\partial \bar{p}}{\partial t} + \overline{u_j \frac{\partial p}{\partial x_j}} + \overline{\tau_{ij} \frac{\partial u_i}{\partial x_j}} - \frac{\partial \overline{\dot{q}_j''}}{\partial x_j}$$

$$\bar{p} = \bar{\rho} R \tilde{T} \sum_{k=1}^{N_s} \frac{\tilde{Y}_k}{M_k} + R \sum_{k=1}^{N_s} \frac{(\overline{\rho T Y_k} - \bar{\rho} \tilde{T} \tilde{Y}_k)}{M_k}$$

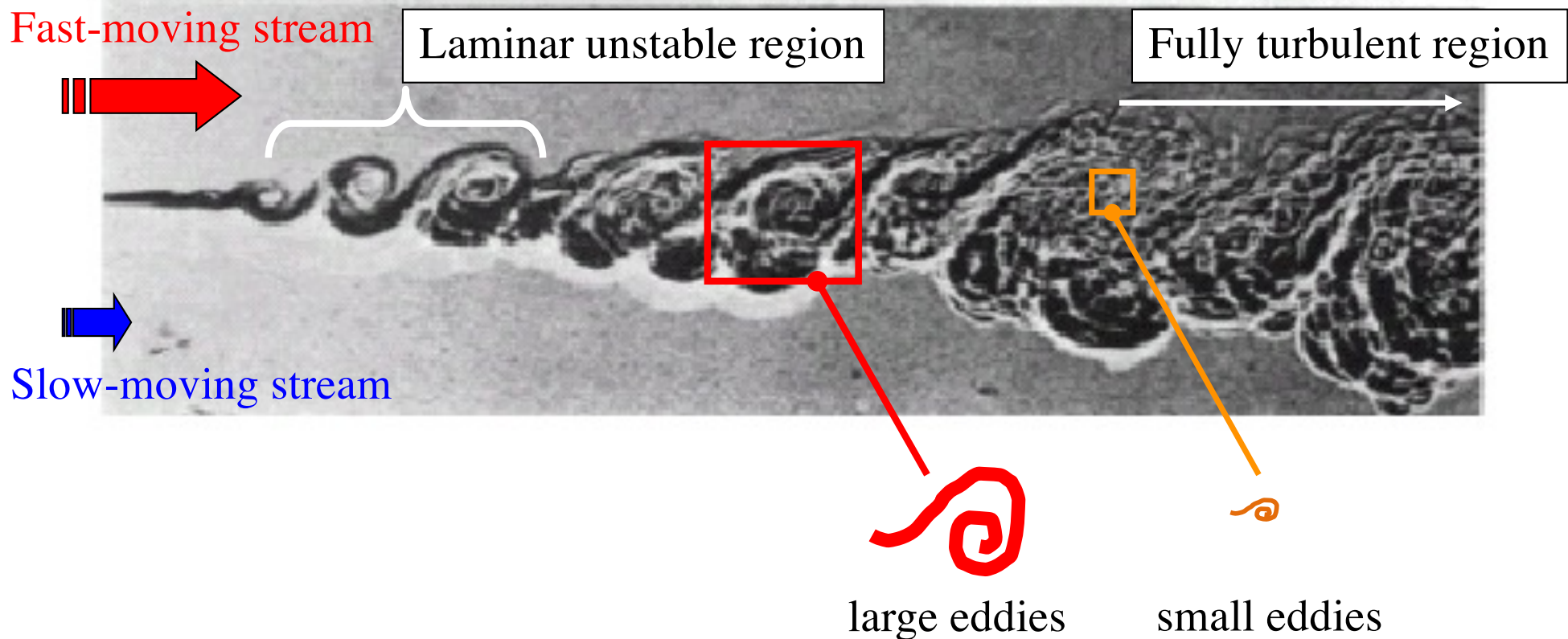


- Mathematical modeling (Large Eddy Simulation – LES)
 - Mathematical formulation applied to LES-filtered (*i.e.* computational-grid-cell-averaged) quantities; requires models to describe unresolved (subgrid-scale) physics
 - Models to describe turbulent fluxes: $\lambda_{kj}, T_{ij}, Q_j$
 - Models to describe turbulent combustion: $\bar{\omega}_k'''$
 - Models to describe thermal radiation transport: $\bar{q}_{rad}''' = -\partial/\partial x_j(\bar{q}_j'')$

Computational Infrastructure



- Mathematical modeling (LES)
 - Computational grid requirement
 - Turbulence viewed as a multi-scale problem





- Mathematical modeling (LES)
 - Computational grid requirement (fine-grained LES)
 - Example: pool fire, $\dot{Q} = 1$ MW; $D = 1$ m

$$L_t \approx 0.5 \times D = 0.5 \text{ m}$$

$$\Rightarrow \Delta x_{LES} \approx \frac{L_t}{10} = 0.05 \text{ m} \approx 100 \times \Delta x_{DNS}$$



- Mathematical modeling

- Direct Numerical Simulation (DNS)

- Grid-resolved scales: $L_t, \eta_K, \delta_{flame}, \delta_{soot}$

$$\Delta x_{DNS} \approx \eta_K$$

$$\Delta x_{DNS} \approx (\delta_{flame}/10)$$

$$\Delta x_{DNS} \approx (\delta_{soot}/10)$$

$$\Rightarrow \Delta x_{DNS} = O(0.1 \text{ mm})$$

- Large Eddy Simulation (LES)

- Grid-resolved scales: L_t

$$\Delta x_{LES} \approx (L_t/10)$$

- Unresolved scales: $\eta_K, \delta_{flame}, \delta_{soot}$



- Mathematical modeling (LES)
 - Computational grid requirement (coarse-grained LES)
 - Example: fire in a large building system

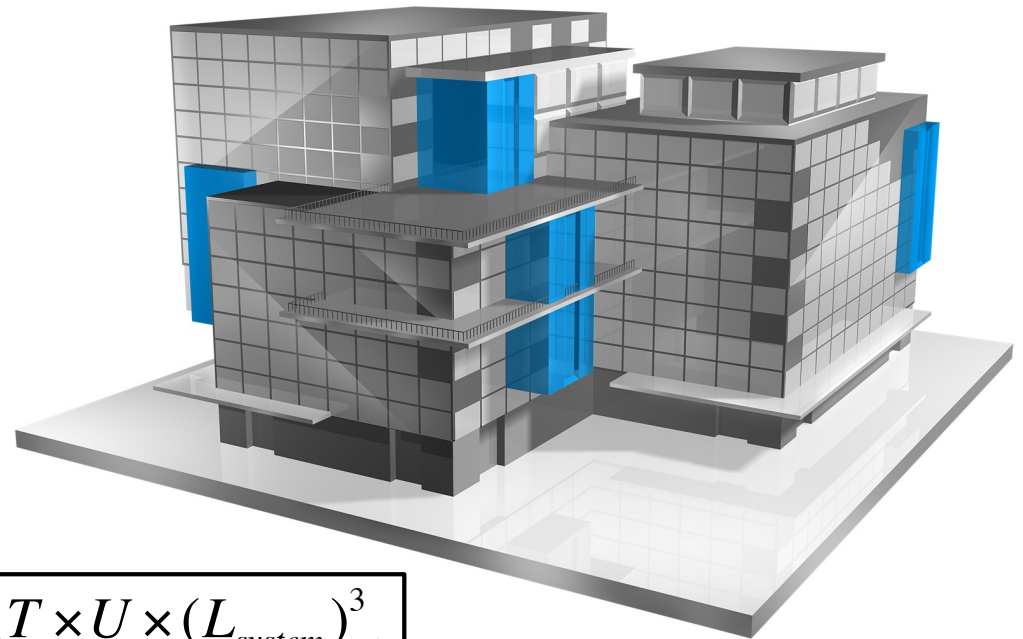
Space: $(L_{system})^3$

$$\frac{\text{CPU cost}}{N_{\Delta t} (N_{\Delta x} N_{\Delta y} N_{\Delta z})} = O(100\mu s)$$

$$N_{\Delta x} \sim N_{\Delta y} \sim N_{\Delta z} = \left(\frac{L_{system}}{\Delta x} \right)$$

$$N_{\Delta t} = \left(\frac{T}{\Delta t} \right) = \left(\frac{T}{CFL \times (\Delta x / U)} \right)$$

$$\text{CPU cost} \times (\Delta x)^4 = O(100\mu s) \times \left(\frac{T \times U \times (L_{system})^3}{CFL} \right)$$





- Mathematical modeling (LES)
 - Computational grid requirement (coarse-grained LES)
 - Example: fire in a large building system

$$L_{system} = 50 \text{ m}$$

$$U = 10 \text{ m/s}$$

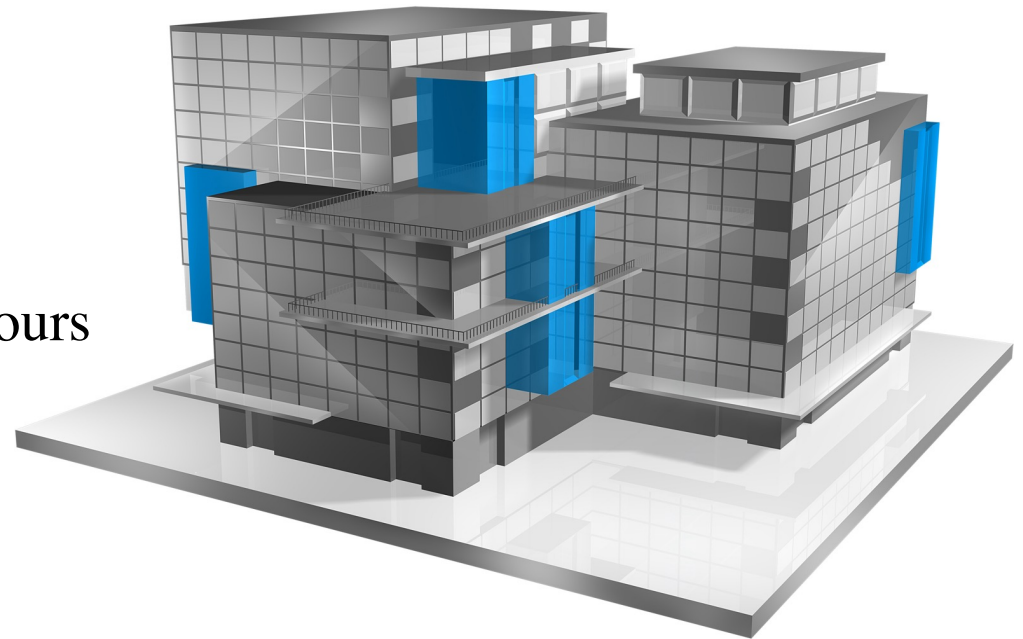
$$T = 10 \text{ minutes}$$

$$\text{CPU cost} = 24 \times 32 \text{ PEs} = 768 \text{ hours}$$

$$CFL = 0.5$$

$$\Rightarrow \Delta x \sim 0.5 \text{ m}$$

$$(N_{\Delta x} N_{\Delta y} N_{\Delta z}) \sim 1 \text{ Million}$$





- Computational modeling

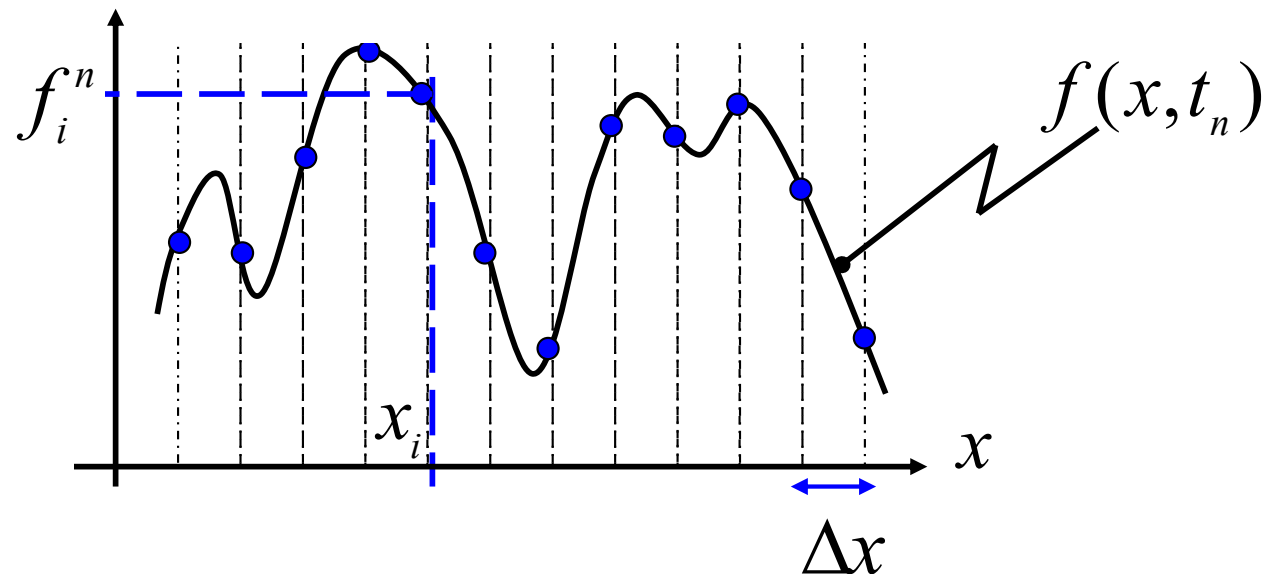
- Going from a mathematical model to a numerical algorithm

Model problem

$$\frac{\partial f}{\partial t} + u \frac{\partial f}{\partial x} = D \frac{\partial^2 f}{\partial x^2}$$

- **Discretization:** describe continuous function $f(x,t)$ as a set of discrete numbers corresponding to values taken by f at prescribed space and time locations

$$f_i^n = f(x_i, t_n)$$





- Computational modeling

- Going from a mathematical model to a numerical algorithm

Model problem

$$\boxed{\frac{\partial f}{\partial t} + u \frac{\partial f}{\partial x} = D \frac{\partial^2 f}{\partial x^2}}$$

- Formulation of discretized equations: describe original partial differential equations (PDEs) as a set of algebraic operations (additions, subtractions, multiplications, divisions) that can be performed by a computer

$$\underbrace{\frac{f_i^{n+1} - f_i^n}{\Delta t}}_{\text{unsteady}} + u \underbrace{\frac{f_{i+1}^n - f_{i-1}^n}{2\Delta x}}_{\text{convection}} = D \underbrace{\frac{f_{i+1}^n + f_{i-1}^n - 2f_i^n}{(\Delta x)^2}}_{\text{diffusion}}$$



- Computational modeling

- PDE solvers

- Model problem

$$\frac{\partial f}{\partial t} + u \frac{\partial f}{\partial x} = D \frac{\partial^2 f}{\partial x^2}$$

Computational domain

$$L_x = 20 \text{ m}$$

Initial conditions

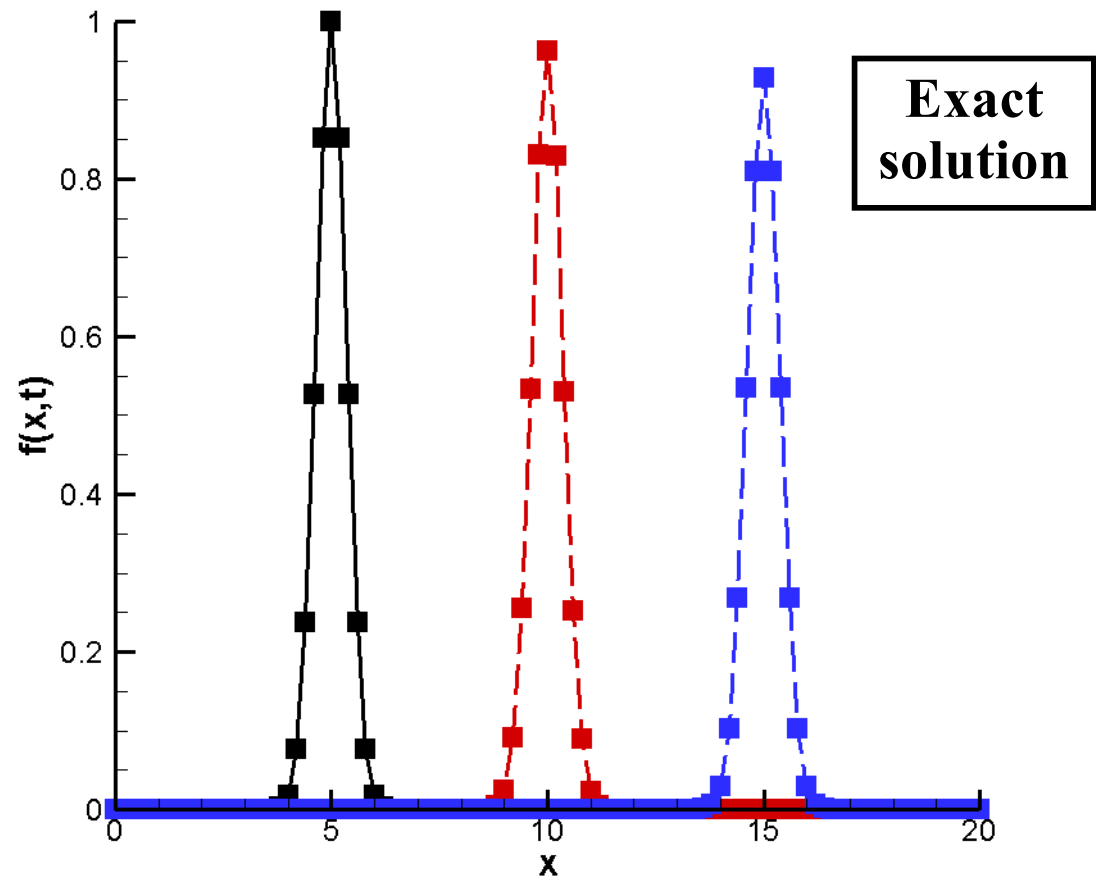
$$f(x, t = 0) = \exp\left(-\frac{(x - x_0)^2}{\sigma_0^2}\right)$$

$$x_0 = 5 \text{ m} ; \sigma_0 = 0.5 \text{ m}$$

Parameters

$$u = 1 \text{ m/s} ; \delta_0 = 2\sigma_0 = 1 \text{ m}$$

$$\text{Re} = \frac{u \times \delta_0}{D} = 1000 ; D = 10^{-3} \text{ m}^2/\text{s}$$





- Computational modeling

- PDE solvers

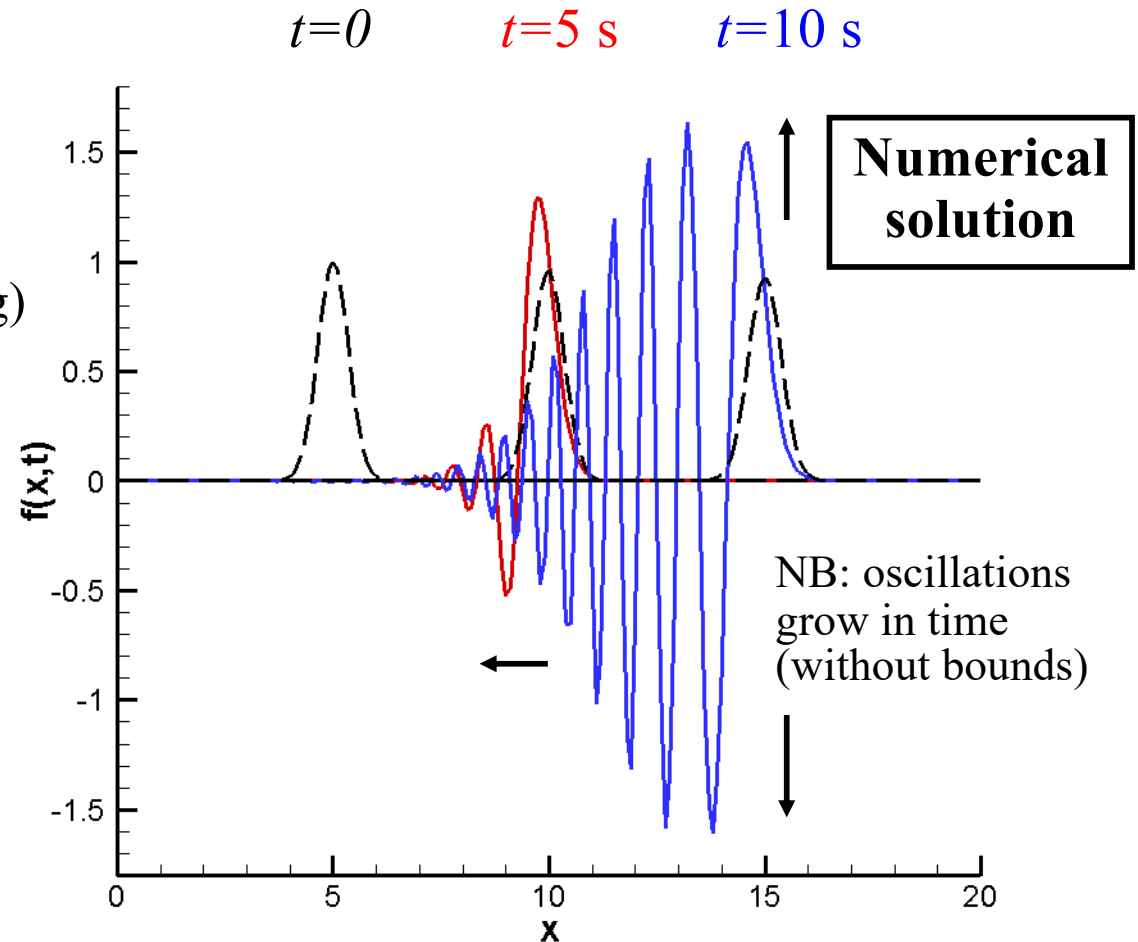
- Model problem

EECD scheme
(Euler/explicit, central-differencing)

$$\Delta x = (\delta_0 / 10) = 0.1 \text{ m}$$

$$\Delta t = 0.2 \times (\Delta x / u) = 0.02 \text{ s}$$

unstable solution!
(unphysical)





- Computational modeling

- PDE solvers

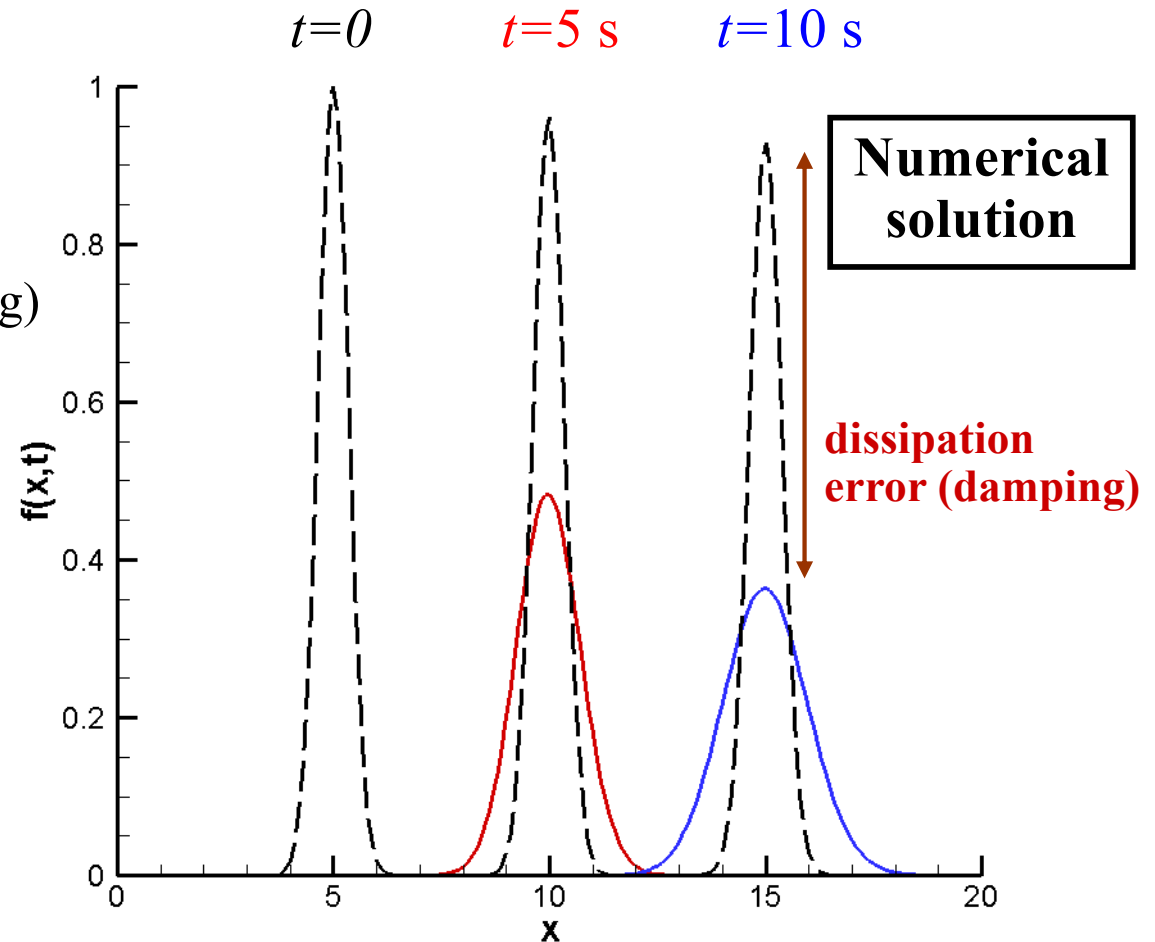
- Model problem

EEUD scheme
(Euler/explicit, upwind-differencing)

$$\Delta x = (\delta_0 / 10) = 0.1 \text{ m}$$

$$\Delta t = 0.2 \times (\Delta x / u) = 0.02 \text{ s}$$

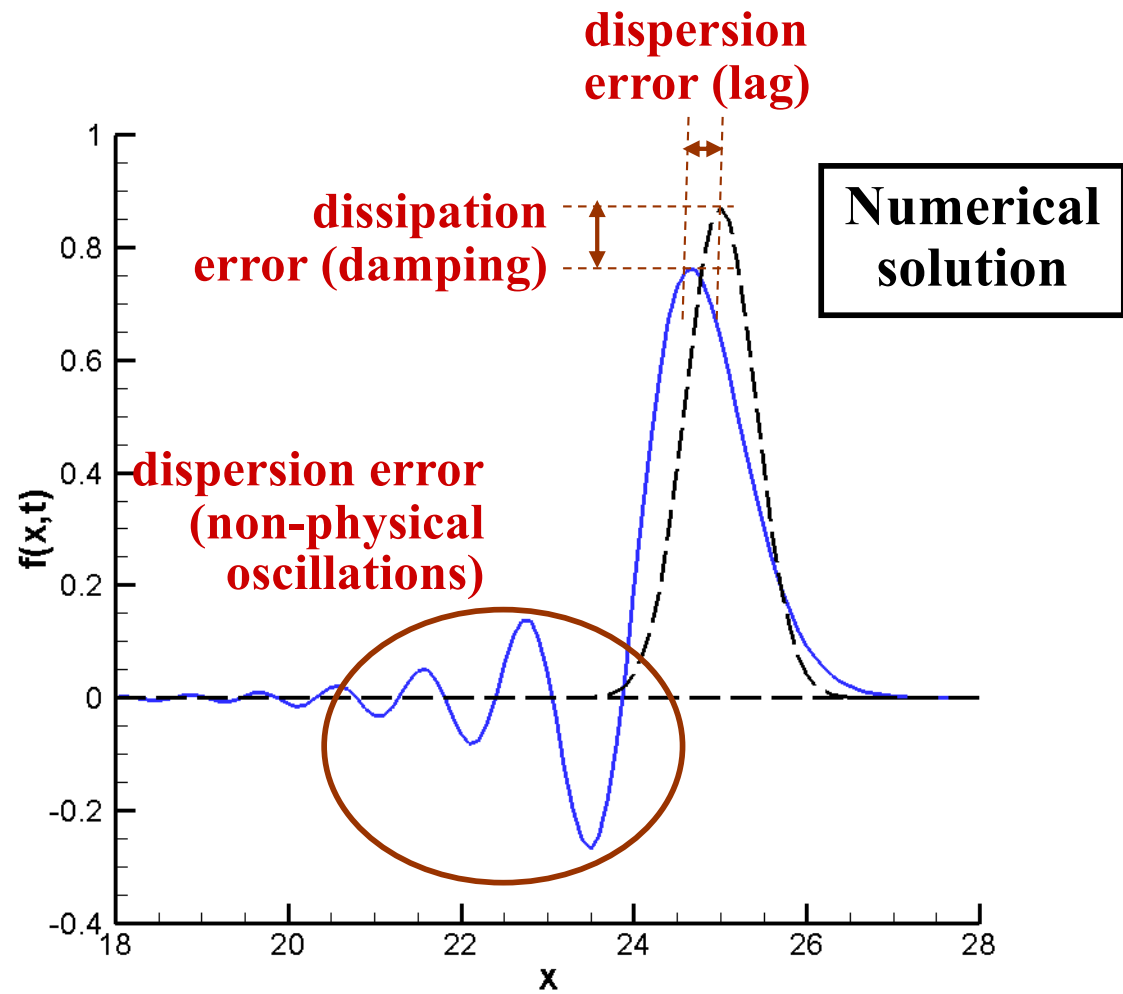
stable solution
positive solution
(physical but) inaccurate





- Computational modeling

- PDE solvers
- Numerical errors





- Computational modeling

- PDE solvers

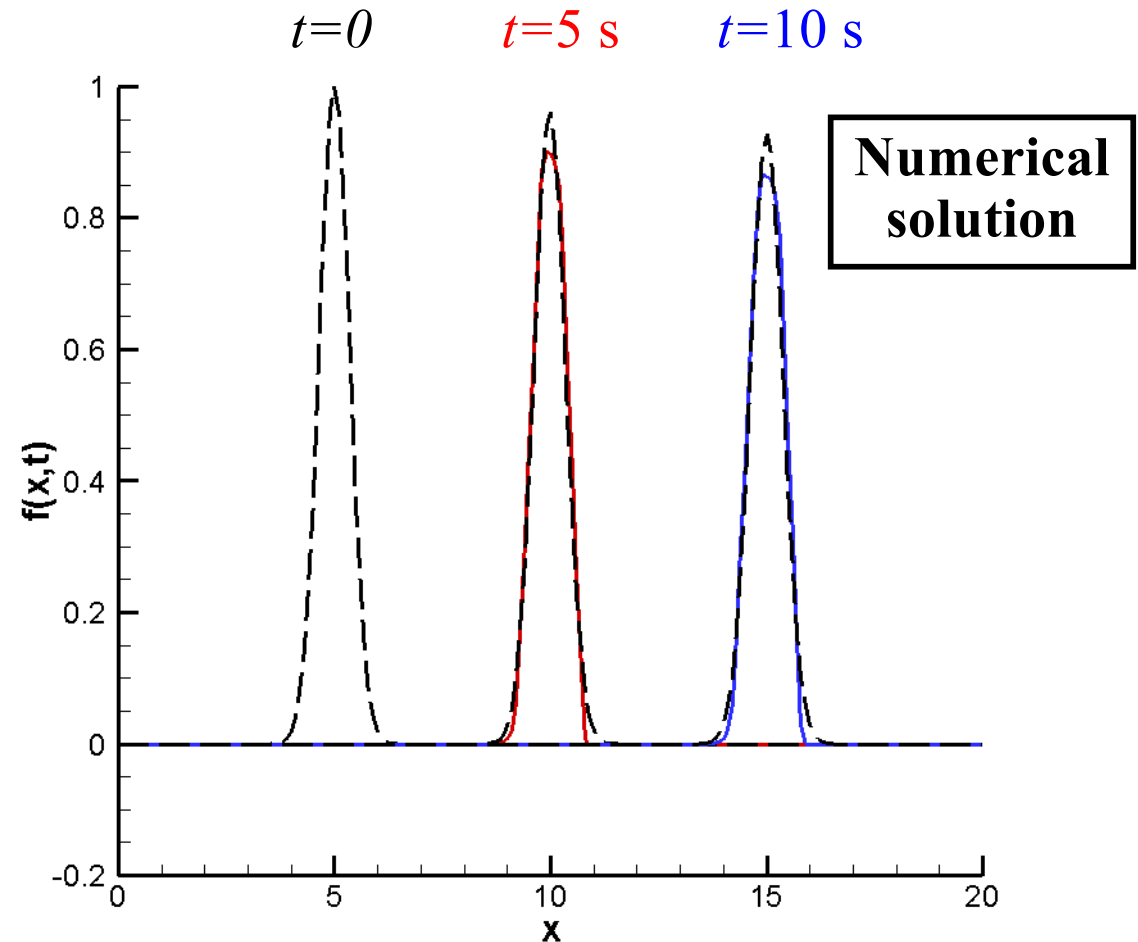
- Model problem

TVD scheme
(Total Variation Diminishing)

$$\Delta x = (\delta_0 / 10) = 0.1 \text{ m}$$

$$\Delta t = 0.2 \times (\Delta x / u) = 0.02 \text{ s}$$

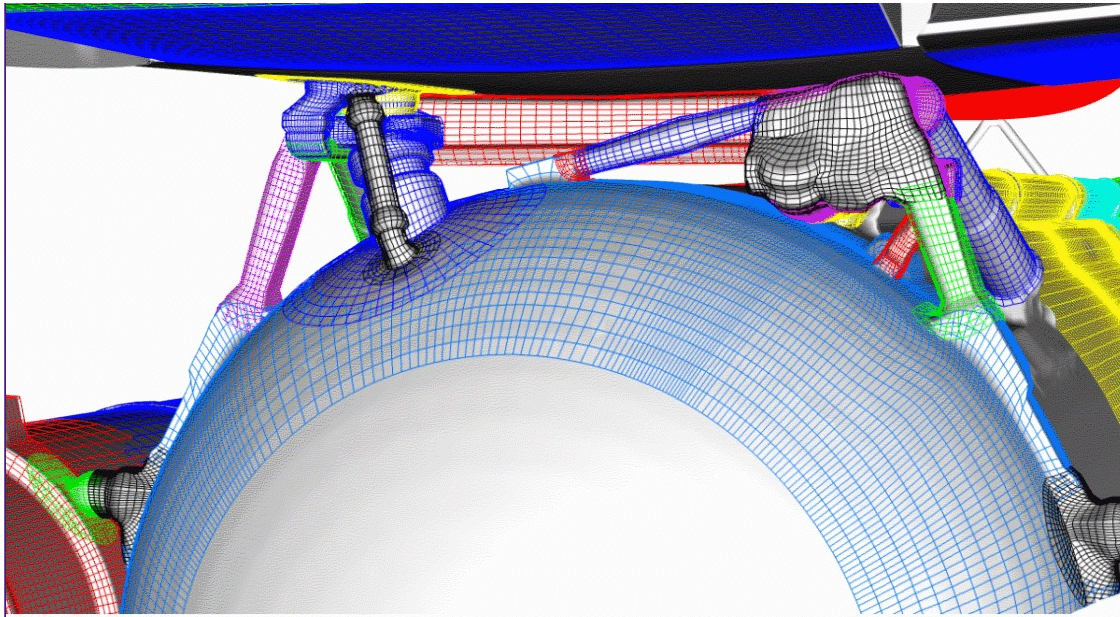
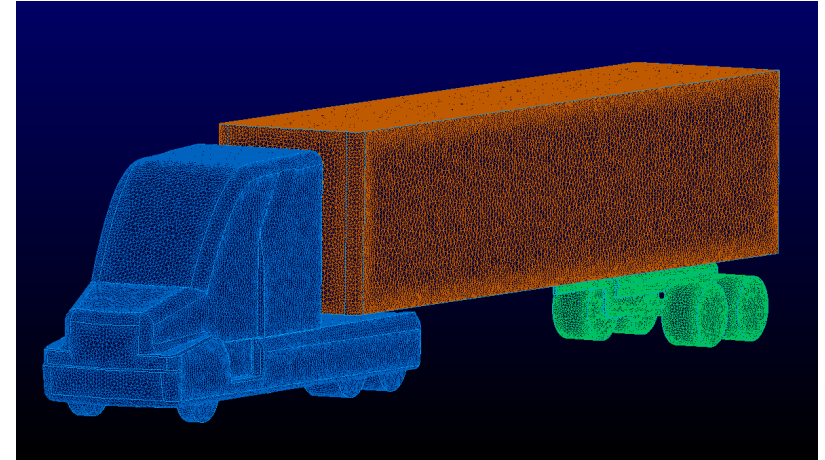
stable solution
physical and accurate
(and positive)



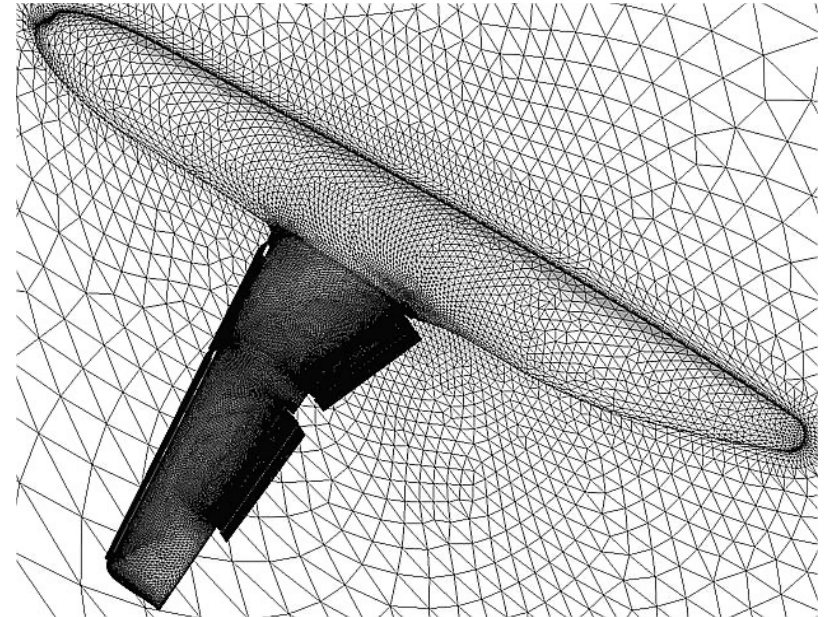
Computational Infrastructure



- Computational modeling
 - Mesh generators



Multiblock-structured grid



Unstructured grid

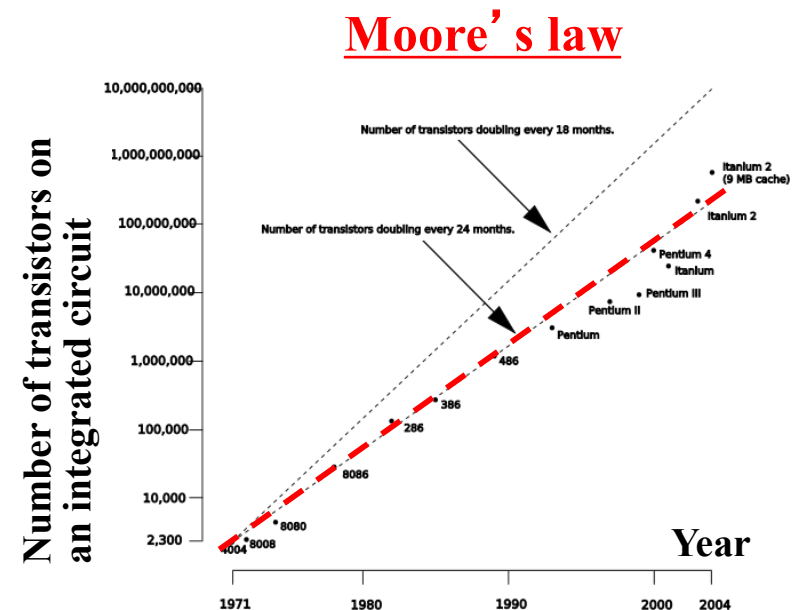
Computational Infrastructure



- Computer power

- Cyber-infrastructure (CI): Information Technologies for computation, storage, communication, and data processing services, driven by:

- Fast development of computer and network technologies
- Dissemination of these technologies on a global scale
- Rapid decrease in cost ($< \$1/\text{MFlops}$)



Computational Infrastructure



- Computer power

- Current status of CI technologies

- High-performance computing (HPC) facilities (Government Research Laboratories, Universities)
 - ✓ Massively parallel processing systems with computational rates ~ 1 Exa (10^{18}) Flops
 - Small-to-mid-scale computing facilities (Businesses)
 - ✓ Medium-scale parallel computing systems (clusters)
 - Grid infra-structure (coupling of distributed and heterogeneous computational resources and data stores via high-speed networks)
 - ✓ Cloud computing



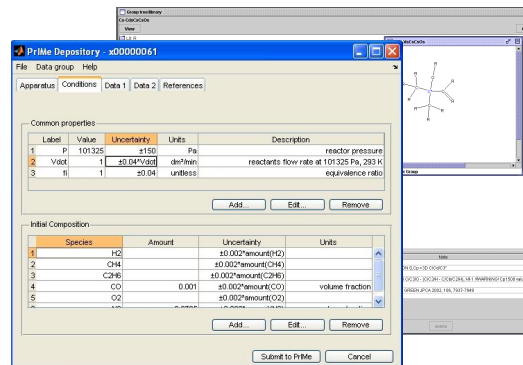
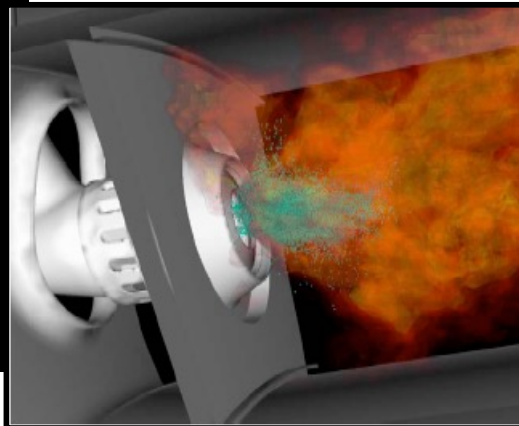
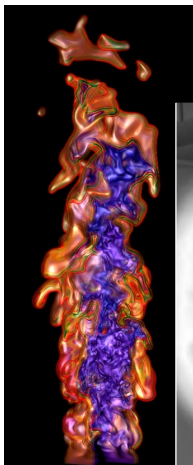
Computational Infrastructure



- Computer power

- Changes brought by CI technologies

- Development of computational research as a new scientific approach
 - Development of computational research as a new engineering approach
 - Development of open-source data and software digital libraries



Compartment Fire Modeling



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 - **Physical Modeling**
 - **Turbulence**
 - Combustion
 - Radiation
 - Pyrolysis
 - Examples

Modeling of Turbulent Flow



- Modeling of convective transport

$$\frac{\partial}{\partial t}(\bar{\rho}\tilde{u}_i) + \underbrace{\frac{\partial}{\partial x_j}(\bar{\rho}\tilde{u}_i\tilde{u}_j)}_{\text{grid-resolved convective transport}} = \underbrace{-\frac{\partial T_{ij}}{\partial x_j}}_{\text{subgrid-scale convective transport}} - \frac{\partial \bar{p}}{\partial x_i} + \frac{\partial \bar{\tau}_{ij}}{\partial x_j} + \bar{\rho}g_i$$

$$T_{ij} = \overline{\rho u_i u_j} - \bar{\rho}\tilde{u}_i\tilde{u}_j$$

requires modeling

Modeling of Turbulent Flow



- Modeling of turbulence

- Classical LES treatment: gradient transport model for turbulent fluxes featuring a turbulent viscosity μ_t

$$T_{ij} = -\boxed{\mu_t} \left(\frac{\partial \tilde{u}_i}{\partial x_j} + \frac{\partial \tilde{u}_j}{\partial x_i} - \frac{2}{3} \delta_{ij} \frac{\partial \tilde{u}_k}{\partial x_k} \right) + \frac{2}{3} \delta_{ij} \bar{\rho} k_{SGS}$$

- Closure expression for μ_t

$$\boxed{\mu_t = \bar{\rho} (C_{\mu_t} \Delta) \sqrt{k_{SGS}}} \quad \text{where } \Delta = (\Delta x_1 \Delta x_2 \Delta x_3)^{1/3}$$

- Closure expression for k_{SGS}

- Models: Smagorinsky; Deardorff (FDS); k -equation (FireFOAM); WALE; *etc*

Modeling of Turbulent Flow

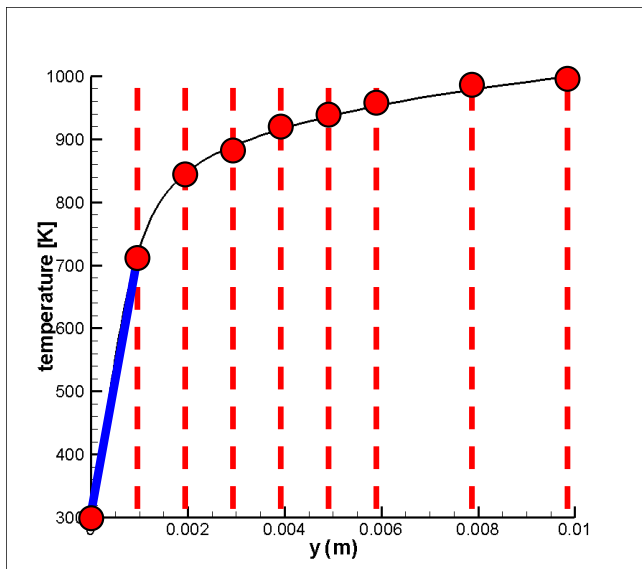


- Limitations of current subgrid-scale turbulence models

- No suitable treatment of boundary layer effects:

- Sharp gradients of temperature at the wall surface need to be evaluated in order to calculate the wall convective heat flux

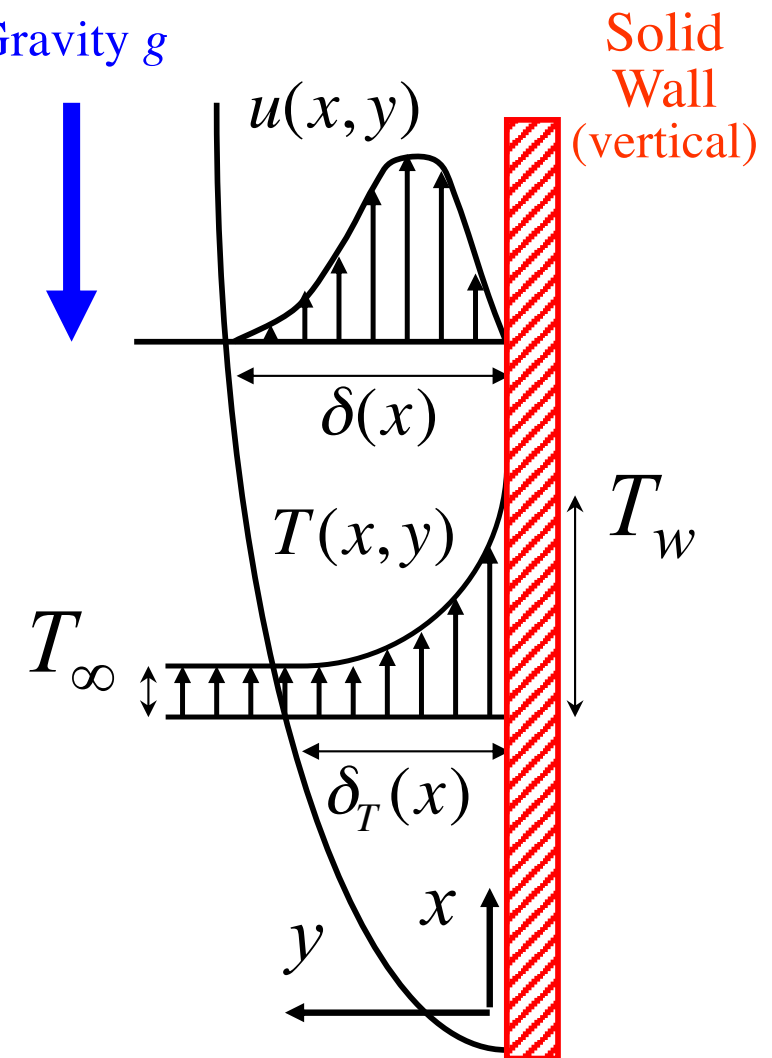
$$\dot{q}_{w,c}'' = -k \left. \frac{\partial T}{\partial y} \right|_{y=0}$$



Wall-resolved LES

$$\Delta y = O(1 \text{ mm})$$

Gravity g



Modeling of Turbulent Flow

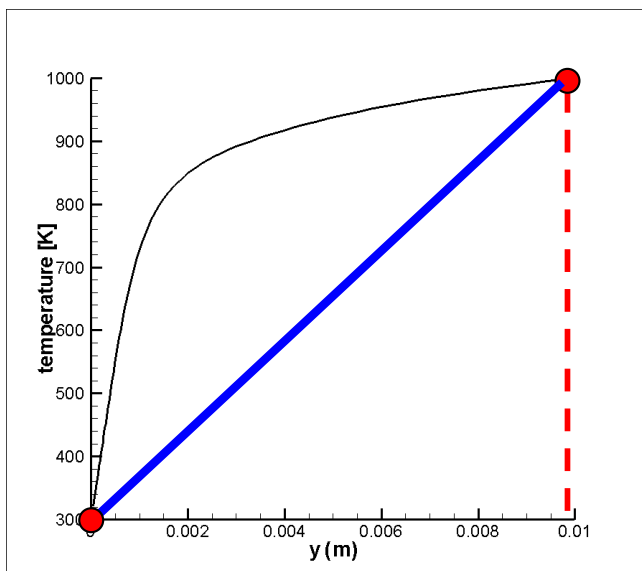


- Limitations of current subgrid-scale turbulence models

- No suitable treatment of boundary layer effects:

- Sharp gradients of temperature at the wall surface need to be evaluated in order to calculate the wall convective heat flux

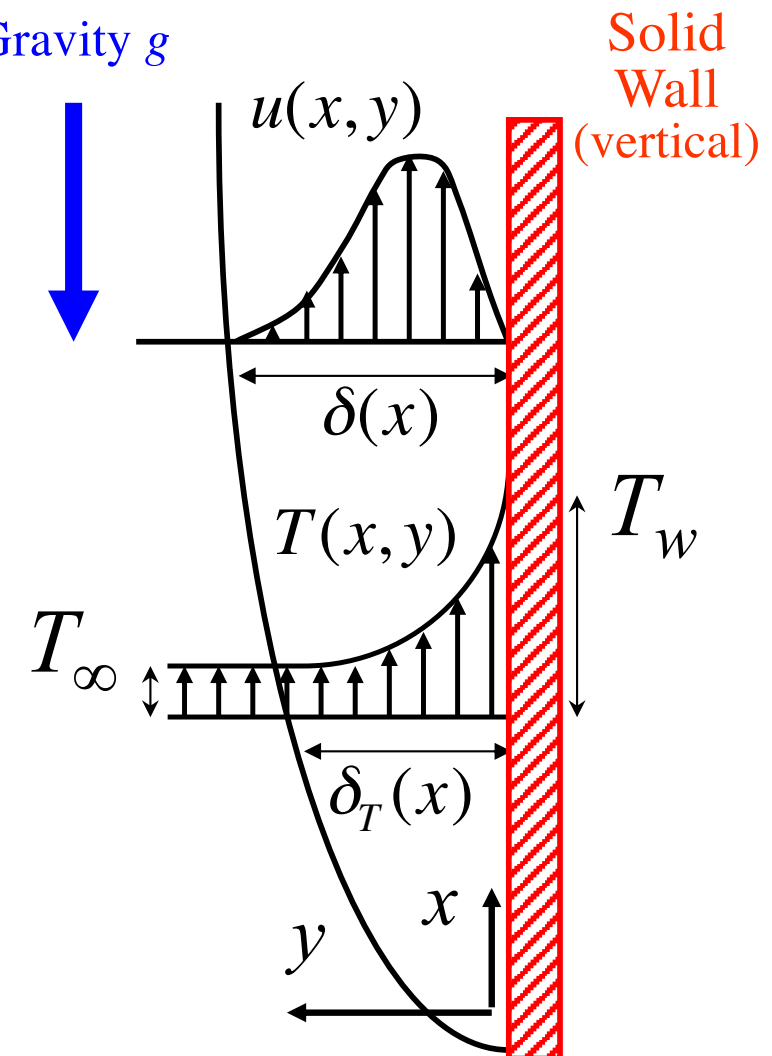
$$\dot{q}_{w,c}'' = -k \left. \frac{\partial T}{\partial y} \right|_{y=0}$$



Wall-modelled LES

$$\Delta y = O(1 \text{ cm})$$

Gravity g



Modeling of Turbulent Flow



- Limitations of current subgrid-scale turbulence models

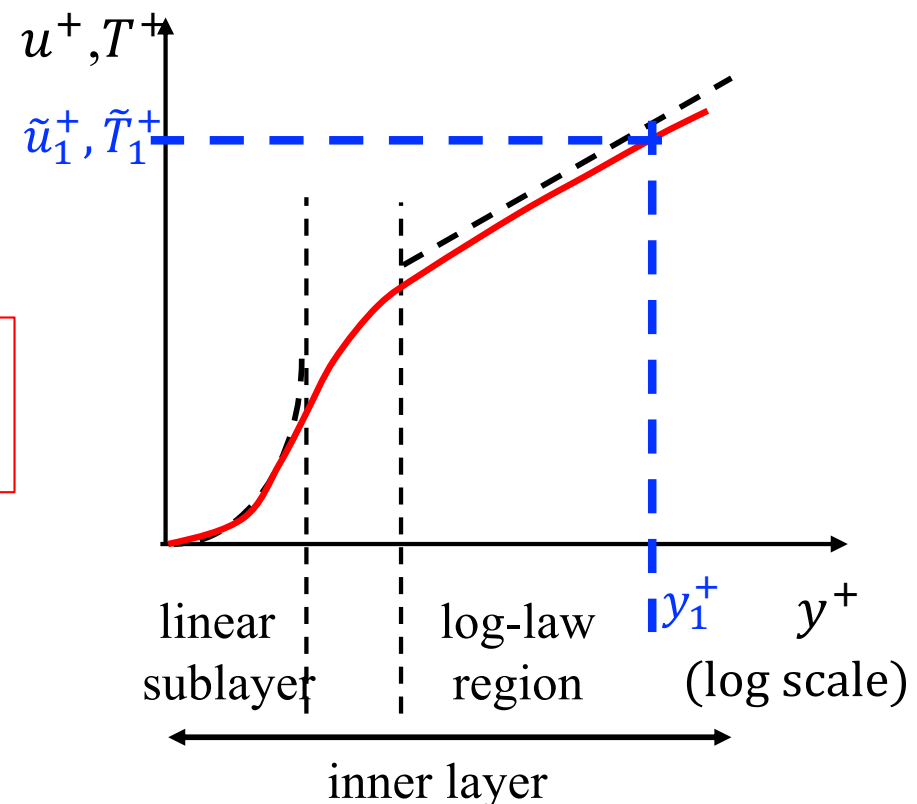
- Traditional approach to wall modeling

- ✓ First off-wall grid node located inside the log-law region
 - ✓ Implicit equation for u_τ

$$\tilde{u}_1^+ = \frac{1}{\kappa} \text{Log}(y_1^+) + C_1$$

- ✓ Equation for \dot{q}_w''

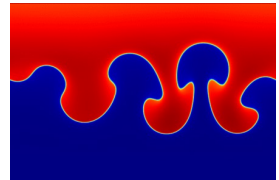
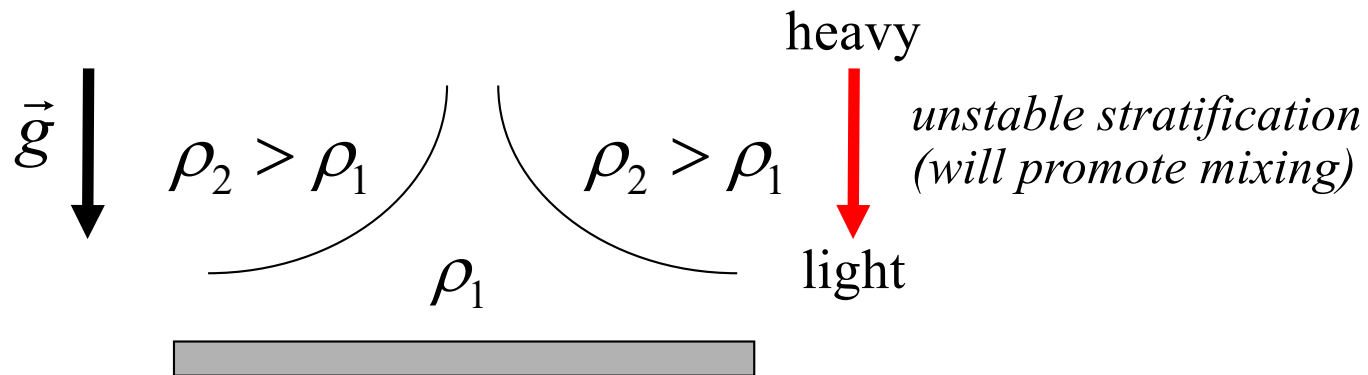
$$\frac{(\rho c_p u_\tau)(T_w - \tilde{T}_1)}{\dot{q}_w''} = \frac{1}{\kappa_\theta} \text{Log}(y_1^+) + C_\theta$$



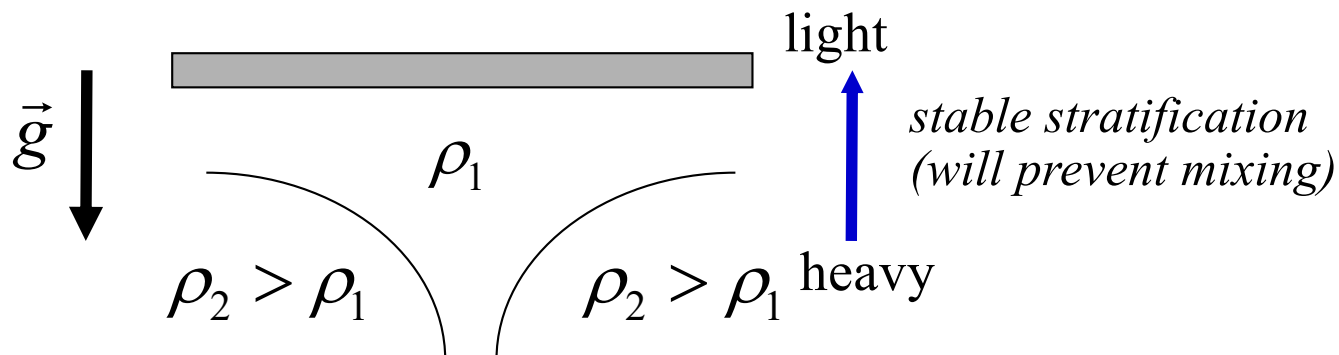
Modeling of Turbulent Flow



- Limitations of current subgrid-scale turbulence models
 - No treatment of buoyancy effects: Rayleigh-Taylor instabilities; reverse cascade of turbulent kinetic energy
 - Example of a pool fire configuration



- Example of a ceiling jet configuration



Compartment Fire Modeling



- Outline
 - Brief Review of Compartment Fire Dynamics
 - Fire Modeling Landscape
 - Computational Infrastructure
 - **Physical Modeling**
 - Turbulence
 - **Combustion**
 - Radiation
 - Pyrolysis
 - Examples

Modeling of Combustion



- Modeling of chemical reaction rates

$$\frac{\partial}{\partial t}(\bar{\rho}\tilde{Y}_k) + \frac{\partial}{\partial x_j}(\bar{\rho}\tilde{Y}_k\tilde{u}_j) = -\frac{\partial\lambda_{kj}}{\partial x_j} + \frac{\partial}{\partial x_j}\left(\overline{\rho D_k \frac{\partial Y_k}{\partial x_j}}\right) + \underbrace{\overline{\dot{\omega}_k}}_{\text{mass reaction rate (grid-scale and subgrid-scale)}}$$

requires modeling

Modeling of Combustion



- Modeling of chemical reaction rates
 - Fuel composition is often unknown in fire problems: use a representative surrogate fuel (wood, plastic, foam, fabric, *etc*)
 - Global combustion equation (no chemistry)
$$C_n H_m O_p + \{n + (m / 4) - (p / 2)\} O_2 \rightarrow n CO_2 + (m / 2) H_2 O$$
 - Closure expression for reaction rate: Eddy Dissipation Model (EDM) model

$$\overline{\dot{\omega}}_F''' = \bar{\rho} \times \frac{\min(\tilde{Y}_F; \tilde{Y}_{O_2} / r_s)}{\tau_t}$$

$$\text{where } \tau_t = C_{\tau_t} \times \left(\frac{\bar{\rho} \Delta^2}{\mu_t} \right)$$



- Limitations of current combustion models
 - No treatment of extinction effects
 - Extinction by oxygen depletion in under-ventilated compartment fires or by evaporative cooling in fires weakened by water-based suppression systems
 - No treatment of ignition effects
 - Possible re-ignition of fuel-air mixture following extinction
 - Ignition of flammable vapor-air mixtures in explosion scenarios (*e.g.*, backdraft)
 - No treatment of toxic products emission (soot, *CO*, *HCN*, *etc*)

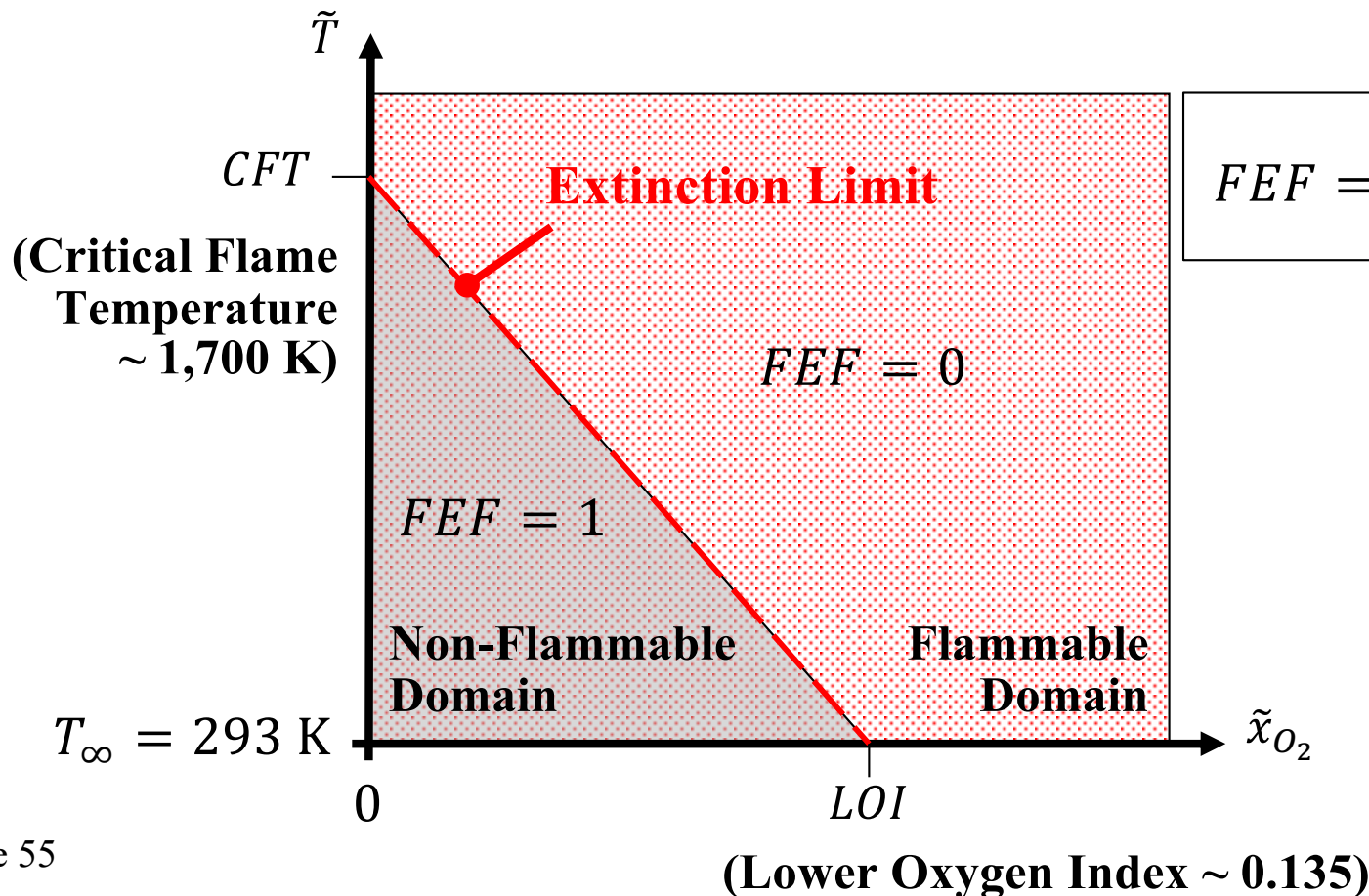
Modeling of Combustion



- Limitations of current combustion models

- Current treatment of extinction effects (FDS)

- Flammability map based on a critical value of the flame temperature for extinction, $FEF = function(\tilde{x}_{O_2}, \tilde{T})$



$$FEF = H\left(\frac{CFT - \tilde{T}}{CFT - T_{\infty}} - \frac{\tilde{x}_{O_2}}{LOI}\right)$$

Compartment Fire Modeling

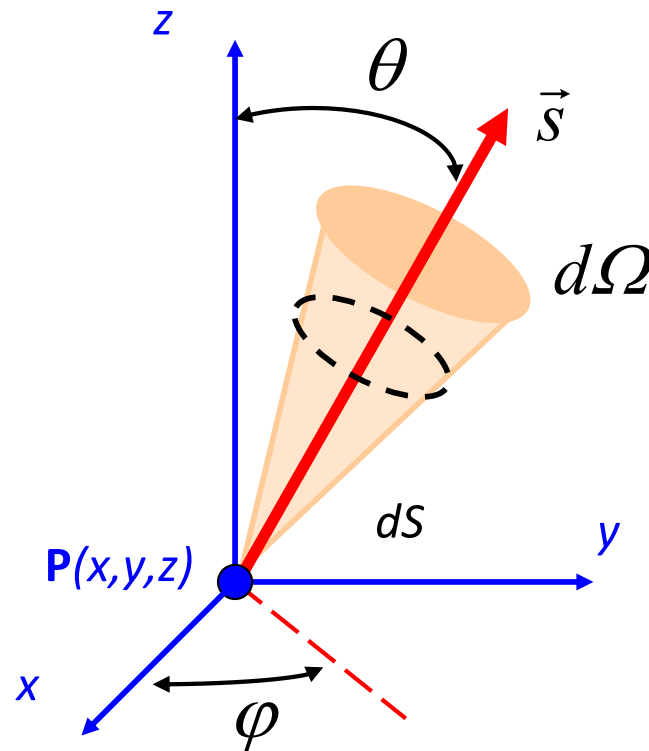


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Modeling of Thermal Radiation



- Modeling of radiative cooling/heating
 - Theoretical framework
 - Decomposition of angular space (hemisphere) into discrete viewing directions and elementary viewing areas (solid angles)



Viewing direction:

$$\vec{s} = \begin{cases} \sin \theta \cos \varphi \\ \sin \theta \sin \varphi \\ \cos \theta \end{cases}$$

Elementary solid angle:

$$d\Omega = \sin \theta \, d\theta \, d\varphi$$

Modeling of Thermal Radiation



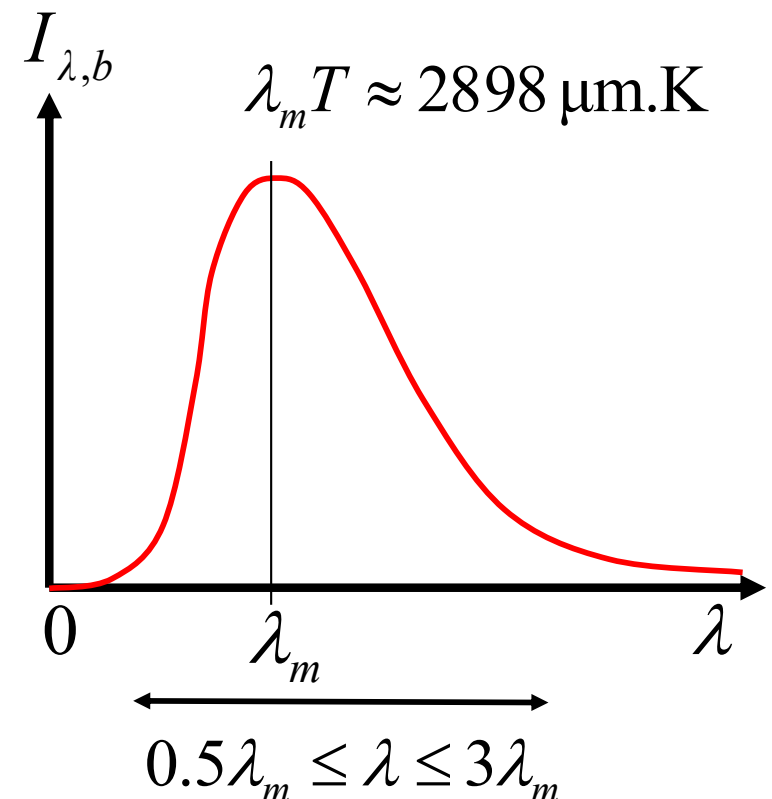
- Modeling of radiative cooling/heating

- Theoretical framework

- Description of radiation energy as electromagnetic wave energy with spectral-dependent properties
 - Black body radiation

$$I_{\lambda,b}(\lambda, T) = \frac{2hc_0^2 T^5}{(\lambda T)^5 \left[\exp\left(\frac{hc_0}{k(\lambda T)}\right) - 1 \right]}$$

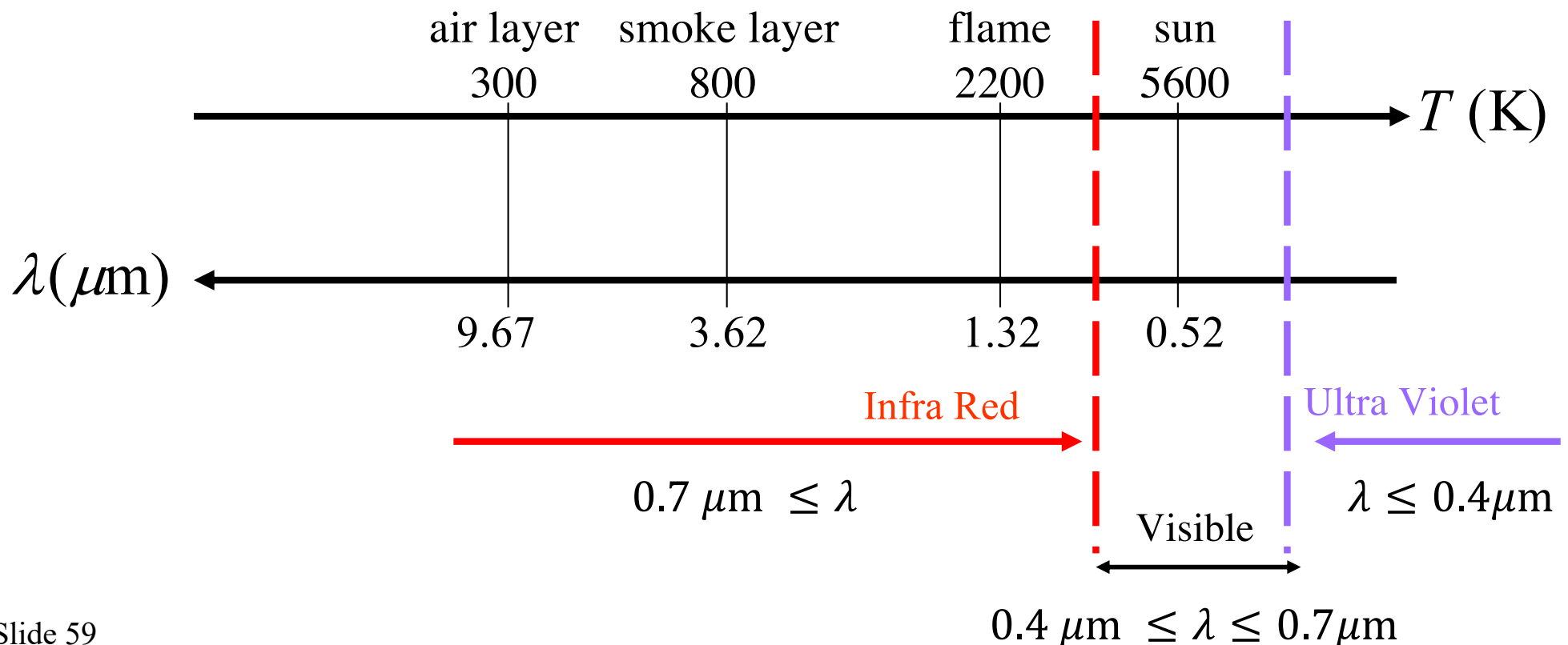
$$I_b = \int_0^\infty I_{\lambda,b} d\lambda = \frac{\sigma T^4}{\pi}$$



Modeling of Thermal Radiation



- Modeling of radiative cooling/heating
 - Theoretical framework
 - Description of radiation energy as electromagnetic wave energy with spectral-dependent properties



Modeling of Thermal Radiation



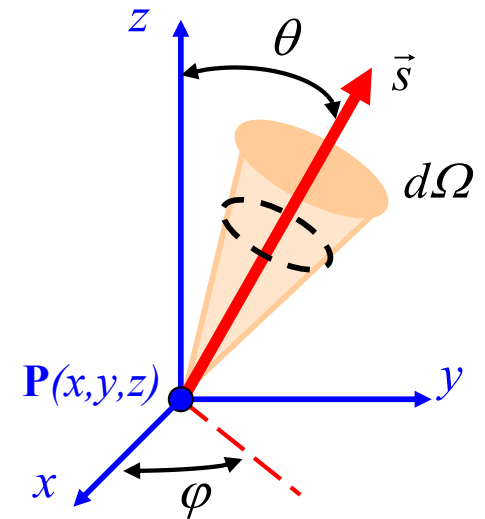
- Modeling of radiative cooling/heating
 - Radiative transfer equation (RTE) (assumed grey medium)

$$I(P, \vec{s}): \quad \nabla I \cdot \vec{s} = \underbrace{\kappa \left(\frac{\sigma T^4}{\pi} \right)}_{\text{emission}} - \underbrace{\kappa I}_{\text{absorption}}$$

emission

absorption

$$\kappa(T, (x_{H_2O}, x_{CO_2}, f_v))$$



$$\dot{q}_{rad}''' = - \int_{4\pi \text{ sr}} (\nabla I \cdot \vec{s}) d\Omega = -4\kappa(\sigma T^4) + \kappa \times \int_{4\pi \text{ sr}} I d\Omega$$

- Solution methods for the RTE: Discrete Ordinate Method (DOM); Discrete Transfer Method (DTM); Monte Carlo Method (MCM)

Modeling of Thermal Radiation

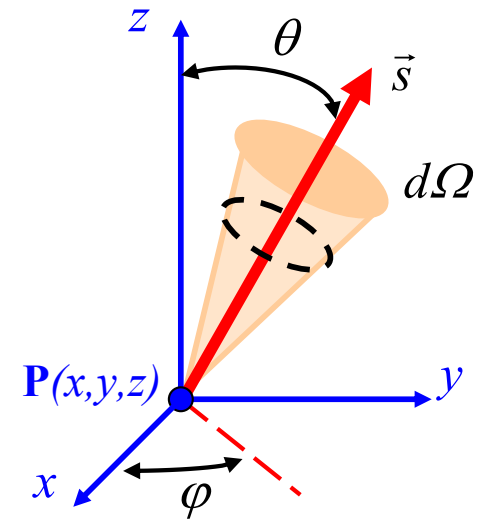


- Modeling of radiative cooling/heating
 - Radiative transfer equation (LES framework)

$$\nabla \bar{I} \cdot \vec{s} = \overbrace{\kappa \left(\frac{\sigma T^4}{\pi} \right)}^{\text{Emission (grid-scale \& subgrid-scale)}} - \overbrace{\kappa \bar{I}}^{\text{Absorption}}$$

requires modeling

$$\bar{\dot{q}}_{rad}''' = -4\overline{\kappa(\sigma T^4)} + \kappa \times \int_{4\pi \text{ sr}} \bar{I} d\Omega$$



Modeling of Thermal Radiation



- Modeling of radiative cooling/heating
 - Closure model for the RTE (LES framework): the prescribed global radiant fraction (PGRF) approach

$$\nabla \bar{I} \cdot \vec{s} = \boxed{\chi_{rad}^g} \times \left(\frac{\dot{q}_{comb}'''}{4\pi} \right), \text{ if } \dot{q}_{comb}''' > 0$$

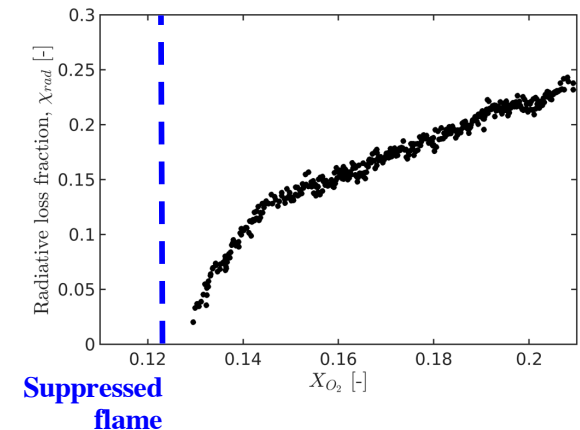
$$\nabla \bar{I} \cdot \vec{s} = 0, \text{ if } \dot{q}_{comb}''' = 0$$

- Global radiant fraction χ_{rad}^g is treated as an input quantity to the fire model and is user-prescribed

Modeling of Thermal Radiation



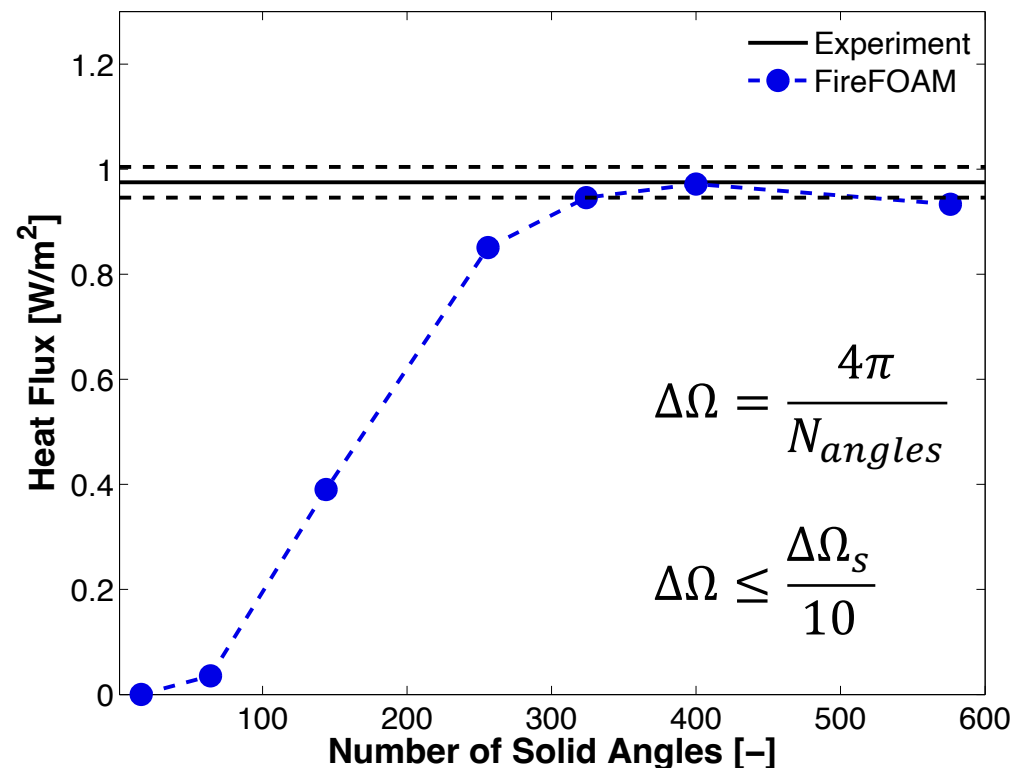
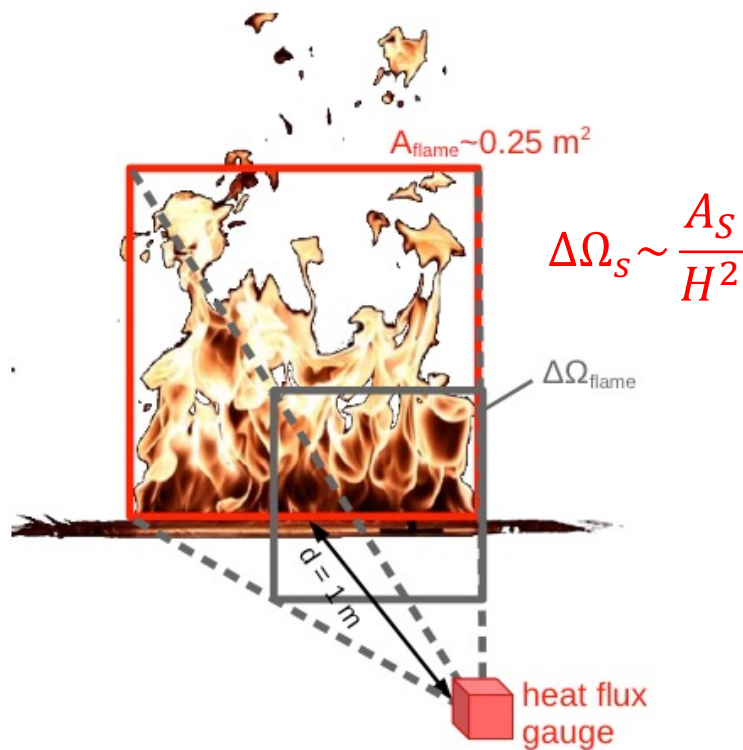
- Limitations of current radiation models
 - The PGRF approach remains approximate (does not work in scenarios in which the flame structure changes and the radiant fraction is not constant)
 - Radiation properties depend on soot volume fraction: the radiation model requires a soot model
 - No treatment of spectral effects (assumed grey medium)
 - Accuracy of DOM and DTM controlled by discretization of angular space and typically decreases in the far-field (ray effect)
 - Solution methods for the RTE are computationally expensive (typically multiplies the cost of CFD by a factor of at least 2)



Modeling of Thermal Radiation



- Control of numerical accuracy (DOM, DTM)
 - The solution of the RTE is a function of spatial location and angular direction; the accuracy of the RTE solution is controlled by the number of angles used in the decomposition of angular space



Compartment Fire Modeling

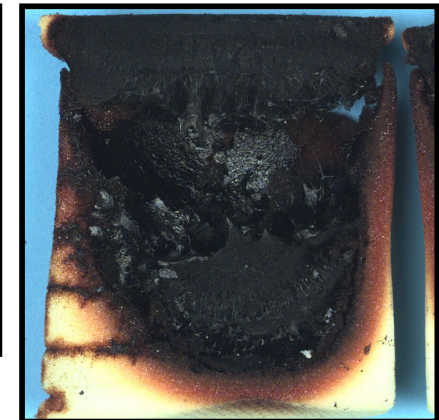
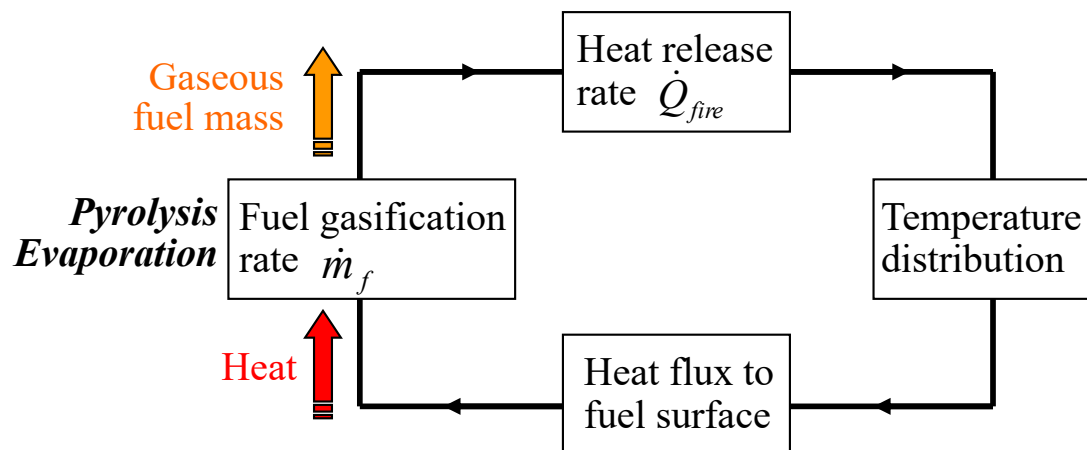


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Modeling of Pyrolysis



- Modeling of the fuel mass loss rate (MLR)
 - Two different approaches
 - *Empirical* approach: prescribed MLR; predicted ignition time
 - *Advanced* approach: MLR predicted from gas-to-solid thermal feedback and finite rate decomposition kinetics



Modeling of Pyrolysis



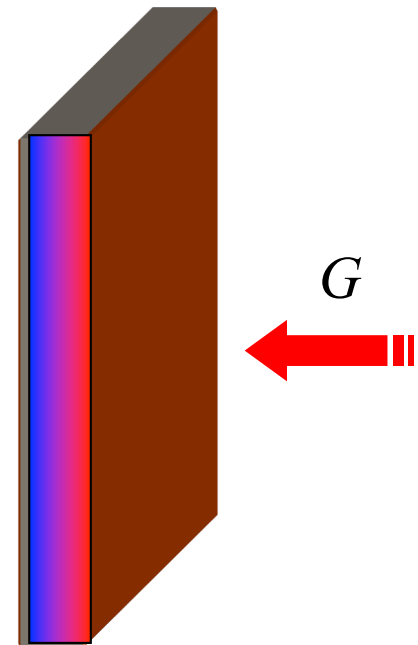
- Modeling of the fuel mass loss rate (MLR)

- Advanced approach

- Finite-rate chemical kinetic model: explicit treatment of thermal decomposition chemistry
 - Thermal degradation across flammable solid described by a local 1D problem in the direction normal to the exposed solid surface

$$\underbrace{-k_s \frac{\partial T_s}{\partial x}(0,t)}_{\text{heat flux to solid interior (conduction)}} = \underbrace{-\varepsilon G + \varepsilon \sigma (T_s(0,t)^4 - T_\infty^4)}_{\text{radiation}} + \underbrace{h(T_s(0,t) - T_\infty)}_{\text{convection}}$$

thermal feedback
(challenge: need to be evaluated accurately)



Modeling of Pyrolysis



- Modeling of the fuel mass loss rate (MLR)

- Advanced approach

- Finite-rate chemical kinetic model: explicit treatment of thermal decomposition chemistry
 - Sequence of 4 chemical reactions (example)

(Drying) (1 *kg wet solid*)

→ ($\eta_{H_2O,Rd}$ *kg water vapor*) + ($\eta_{ds,Rd}$ *kg dry solid*)

(Thermal pyrolysis) (1 *kg dry solid*)

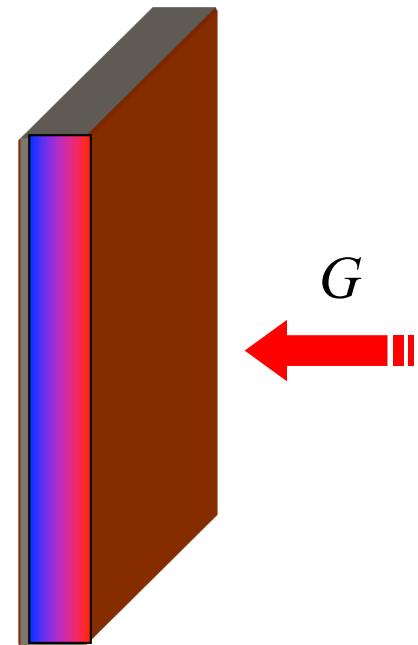
→ ($\eta_{f,Rp}$ *kg fuel*) + ($\eta_{c,Rp}$ *kg char*)

(Oxidative pyrolysis) (1 *kg dry solid*) + ($\eta_{O_2,Rop}$ *kg O₂*)

→ ($\eta_{f,Rop}$ *kg fuel*) + ($\eta_{c,Rop}$ *kg char*)

(Char oxidation) (1 *kg char*) + ($\eta_{O_2,Rco}$ *kg O₂*)

→ ($\eta_{p,Rco}$ *kg products*) + ($\eta_{a,Rco}$ *kg ash*)



Modeling of Pyrolysis



- Modeling of the fuel mass loss rate (MLR)

- Advanced approach

- Arrhenius reaction rates (Lautenberger & Fernandez-Pello, *Combust. Flame* **156**:1503-1513, 2009)

$$\dot{m}_{Rd}''' \Delta V = \left(\frac{\rho_{s,ws}(1 - \psi_{ws})x_{ws}\Delta V}{(\rho_{s,ws}(1 - \psi_{ws})x_{ws}\Delta V)_{\Sigma}} \right)^{n_{Rd}} (\rho_{s,ws}(1 - \psi_{ws})x_{ws}\Delta V)_{\Sigma} \times A_{Rd} \exp\left(-\frac{E_{Rd}}{RT_s}\right)$$

$$\dot{m}_{Rp}''' \Delta V = \left(\frac{\rho_{s,ds}(1 - \psi_{ds})x_{ds}\Delta V}{(\rho_{s,ds}(1 - \psi_{ds})x_{ds}\Delta V)_{\Sigma}} \right)^{n_{Rp}} (\rho_{s,ds}(1 - \psi_{ds})x_{ds}\Delta V)_{\Sigma} \times A_{Rp} \exp\left(-\frac{E_{Rp}}{RT_s}\right)$$

Modeling of Pyrolysis



- Modeling of the fuel mass loss rate (MLR)

- Advanced approach

- Arrhenius reaction rates (Lautenberger & Fernandez-Pello, *Combust. Flame* **156**:1503-1513, 2009)

$$\dot{m}_{Rop}''' \Delta V = \left(\frac{\rho_{s,ds}(1 - \psi_{ds})x_{ds}\Delta V}{(\rho_{s,ds}(1 - \psi_{ds})x_{ds}\Delta V)_{\Sigma}} \right)^{n_{Rop}} (\rho_{s,ds}(1 - \psi_{ds})x_{ds}\Delta V)_{\Sigma} \times \left(\frac{Y_{g,O_2}}{Y_{g,O_2,air}} \right)^{n_{O_2,Rop}} \times A_{Rop} \exp\left(-\frac{E_{Rop}}{RT_s}\right)$$

$$\dot{m}_{Rco}''' \Delta V = \left(\frac{\rho_{s,c}(1 - \psi_c)x_c\Delta V}{(\rho_{s,c}(1 - \psi_c)x_c\Delta V)_{\Sigma}} \right)^{n_{Rco}} (\rho_{s,c}(1 - \psi_c)x_c\Delta V)_{\Sigma} \times \left(\frac{Y_{g,O_2}}{Y_{g,O_2,air}} \right)^{n_{O_2,Rco}} \times A_{Rco} \exp\left(-\frac{E_{Rco}}{RT_s}\right)$$

Modeling of Pyrolysis



- Modeling of the fuel mass loss rate (MLR)

- Advanced approach

- Governing equations (Lautenberger & Fernandez-Pello, *Combust. Flame* **156**:1503-1513, 2009)

$$\frac{\partial}{\partial t} (\rho_{s,ws} (1 - \psi_{ws}) x_{ws} \Delta V) = -\dot{m}_{Rds}''' \Delta V$$

$$\frac{\partial}{\partial t} (\rho_{s,ds} (1 - \psi_{ds}) x_{ds} \Delta V) = +\eta_{ds,Rd} \dot{m}_{Rd}''' \Delta V - \dot{m}_{Rp}''' \Delta V - \dot{m}_{Rop}''' \Delta V$$

$$\frac{\partial}{\partial t} (\rho_{s,c} (1 - \psi_c) x_c \Delta V) = +\eta_{c,Rp} \dot{m}_{Rp}''' \Delta V + \eta_{c,Rop} \dot{m}_{Rop}''' \Delta V - \dot{m}_{Rco}''' \Delta V$$

$$\frac{\partial}{\partial t} (\rho_{s,a} (1 - \psi_a) x_a \Delta V) = +\eta_{a,Rco} \dot{m}_{Rco}''' \Delta V$$

$$\begin{aligned} \frac{\partial}{\partial t} (\bar{\rho}_g Y_{g,O_2} \bar{\psi} \Delta V) + \frac{1}{\phi} \frac{\partial}{\partial \zeta} (\phi \dot{m}_{\zeta}'' Y_{g,O_2}) \Delta V \\ = \frac{1}{\phi} \frac{\partial}{\partial \zeta} (\phi \bar{\psi} \bar{\rho}_g \bar{D}_g \frac{\partial Y_{g,O_2}}{\partial \zeta}) \Delta V - \eta_{O_2,Rop} \dot{m}_{Rop}''' \Delta V - \eta_{O_2,Rco} \dot{m}_{Rco}''' \Delta V \end{aligned}$$

Modeling of Pyrolysis



- Modeling of the fuel mass loss rate (MLR)
 - Advanced approach
 - Governing equations (Lautenberger & Fernandez-Pello, *Combust. Flame* **156**:1503-1513, 2009)

$$\dot{m}_{\zeta}'' = -\frac{\bar{K}}{v_g} \times \frac{\partial p}{\partial \zeta}$$

$$\frac{\partial}{\partial t} \left(\frac{M_g}{RT_g} p \bar{\psi} \Delta V \right) = \frac{1}{\phi} \frac{\partial}{\partial \zeta} \left(\phi \frac{\bar{K}}{v_g} \frac{\partial p}{\partial \zeta} \right) \Delta V + \dot{m}_{sg}''' \Delta V$$

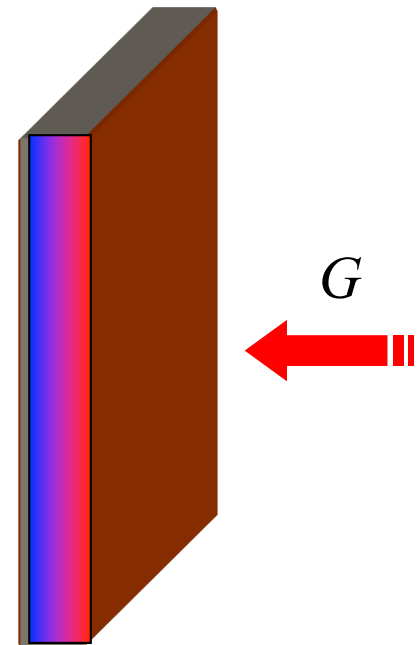
$$\begin{aligned} (\bar{\rho}_s \bar{c}_s (1 - \bar{\psi}) + \bar{\rho}_g \bar{c}_{p,g} \bar{\psi}) \frac{\partial T_s}{\partial t} \Delta V + \dot{m}_{\zeta}'' \bar{c}_{p,g} \frac{\partial T_s}{\partial \zeta} \Delta V \\ = \frac{1}{\phi} \frac{\partial}{\partial \zeta} \left(((1 - \bar{\psi}) \bar{k}_s + \bar{\psi} \bar{k}_g) \phi \frac{\partial T_s}{\partial \zeta} \right) \Delta V + \dot{q}_{hrr}''' \Delta V \end{aligned}$$

Modeling of Pyrolysis



- Modeling of the fuel mass loss rate (MLR)
 - Advanced approach
 - Fuel mass loss rate

$$\dot{m}_f'' = \int_{depth} (\eta_{f,Rp} \dot{m}_{Rp}''' + \eta_{f,Rop} \dot{m}_{Rop}''') d\zeta$$



Modeling of Pyrolysis



• Modeling of the fuel mass loss rate (MLR)

➤ Advanced approach

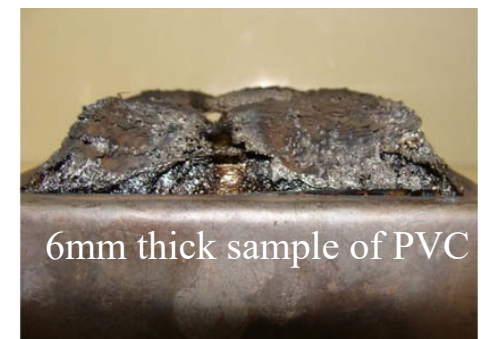
▪ Input data

Thermal properties of wet solid	$\rho_{s,ws}, k_{s,ws}, c_{s,ws}, \epsilon_{ws}$
Thermal properties of dry solid	$\rho_{s,ds}, k_{s,ds}, c_{s,ds}, \epsilon_{ds}$
Thermal properties of char	$\rho_{s,c}, k_{s,c}, c_{s,c}, \epsilon_c$
Thermal properties of ash	$\rho_{s,a}, k_{s,a}, c_{s,a}, \epsilon_a$
Porosity and permeability of wet solid	ψ_{ws}, K_{ws}
Porosity and permeability of dry solid	ψ_{ds}, K_{ds}
Porosity and permeability of char	ψ_c, K_c
Porosity and permeability of ash	ψ_a, K_a
Drying reaction	$A_{Rd}, E_{Rd}, n_{Rd}, \Delta H_{Rd}, \eta_{ds,Rd}$
Thermal pyrolysis reaction	$A_{Rp}, E_{Rp}, n_{Rp}, \Delta H_{Rp}, \eta_{c,Rp}$
Oxidative pyrolysis reaction	$A_{Rop}, E_{Rop}, n_{Rop}, n_{O_2,Rop}, \Delta H_{Rop}, \eta_{c,Rop}, \eta_{O_2,Rcp}$
Char oxidation reaction	$A_{Rco}, E_{Rco}, n_{Rco}, n_{O_2,Rco}, \Delta H_{Rco}, \eta_{a,Rco}, \eta_{O_2,Rco}$

Modeling of Pyrolysis



- Modeling of the fuel mass loss rate (MLR)
 - Advanced approach
 - Many of the model parameters cannot be measured and remain unknown
 - Model parameters determined by comparison between model predictions and experimental results from micro-scale tests (*e.g.* thermogravimetric analysis) and bench-scale tests (*e.g.*, cone calorimeter, Fire Propagation Apparatus)
 - This comparison often uses optimization methods



6mm thick sample of PVC

Modeling of Pyrolysis



- Modeling of the fuel mass loss rate (MLR)

- Advanced approach

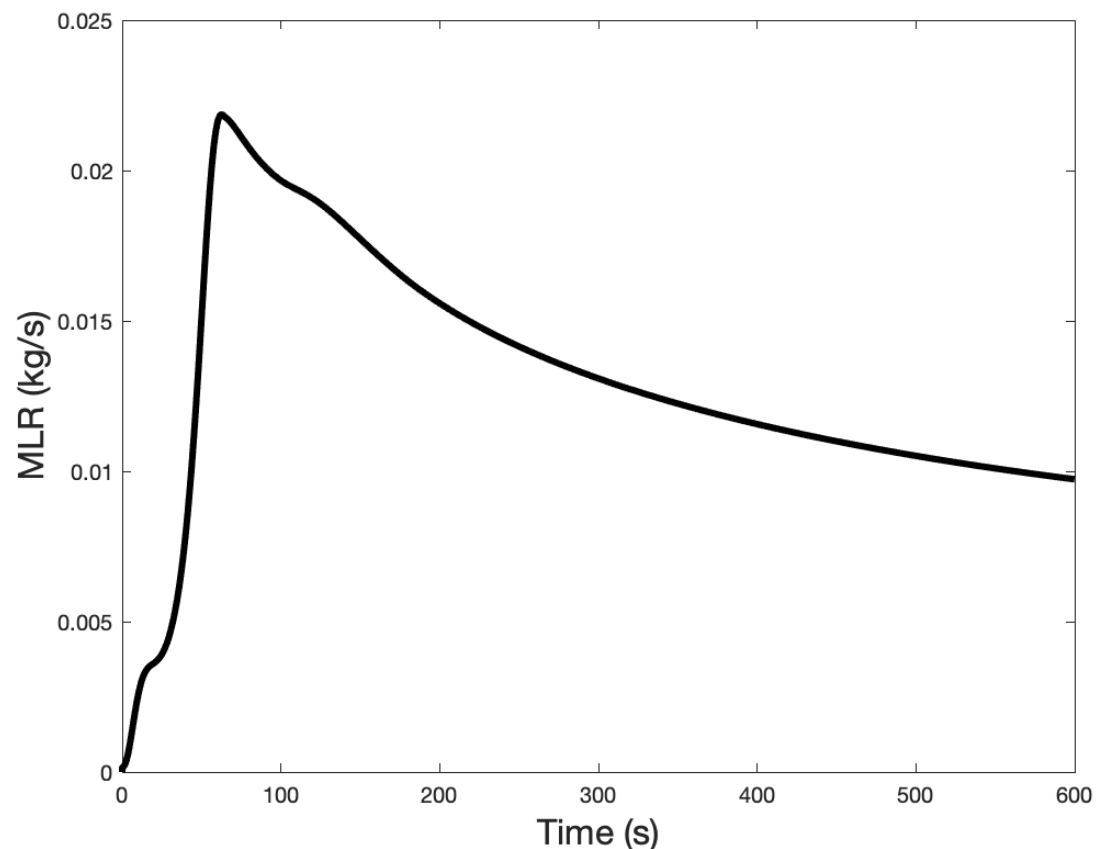
- *Example:* white pine (Lautenberger & Fernandez-Pello, *Combust. Flame* **156**:1503-1513 (2009))

$$\Delta = 3.8 \text{ cm}$$

$$G = 40 \text{ kW/m}^2$$

$$x_{O_2,g} = 0.21$$

$$t = 100 \text{ s}$$



Modeling of Pyrolysis



- Modeling of the fuel mass loss rate (MLR)

- Advanced approach

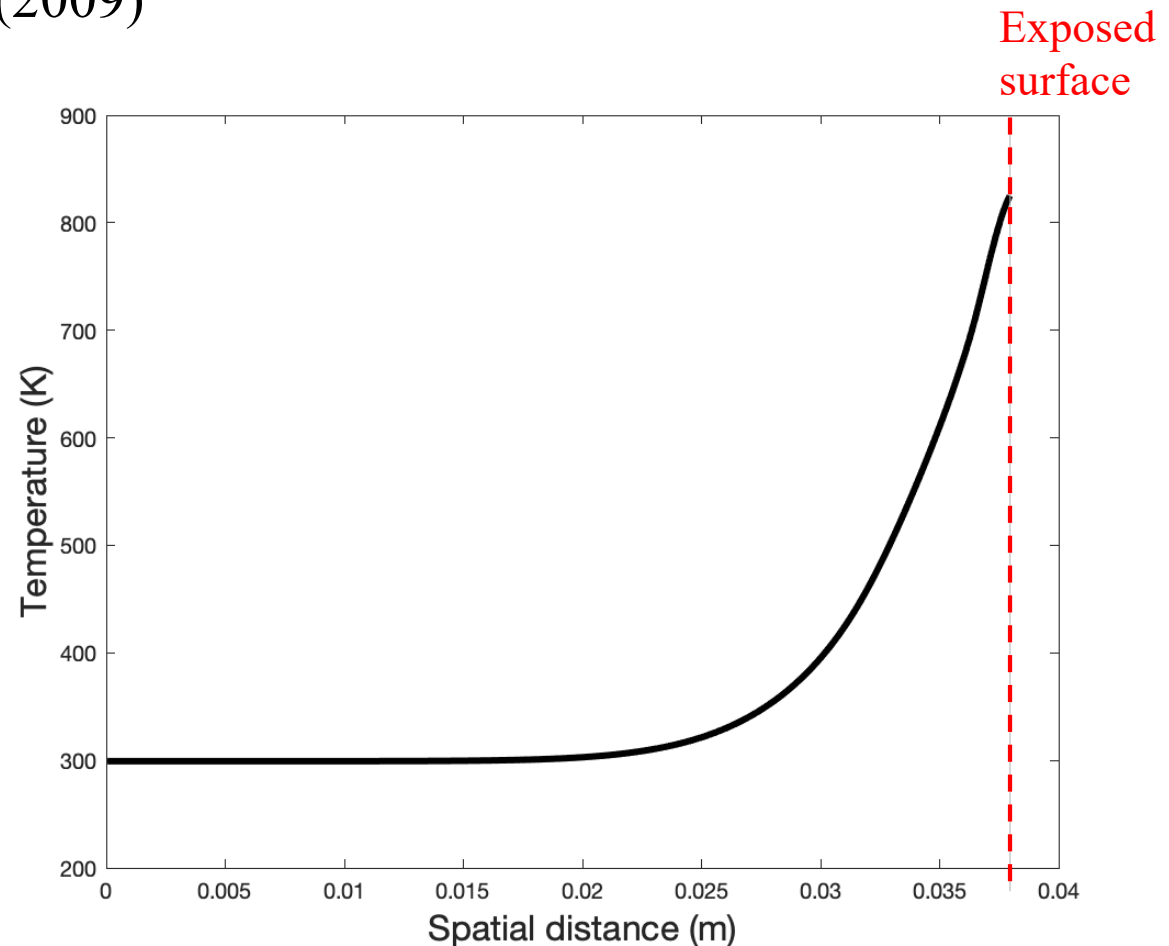
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Modeling of Pyrolysis



- Modeling of the fuel mass loss rate (MLR)

- Advanced approach

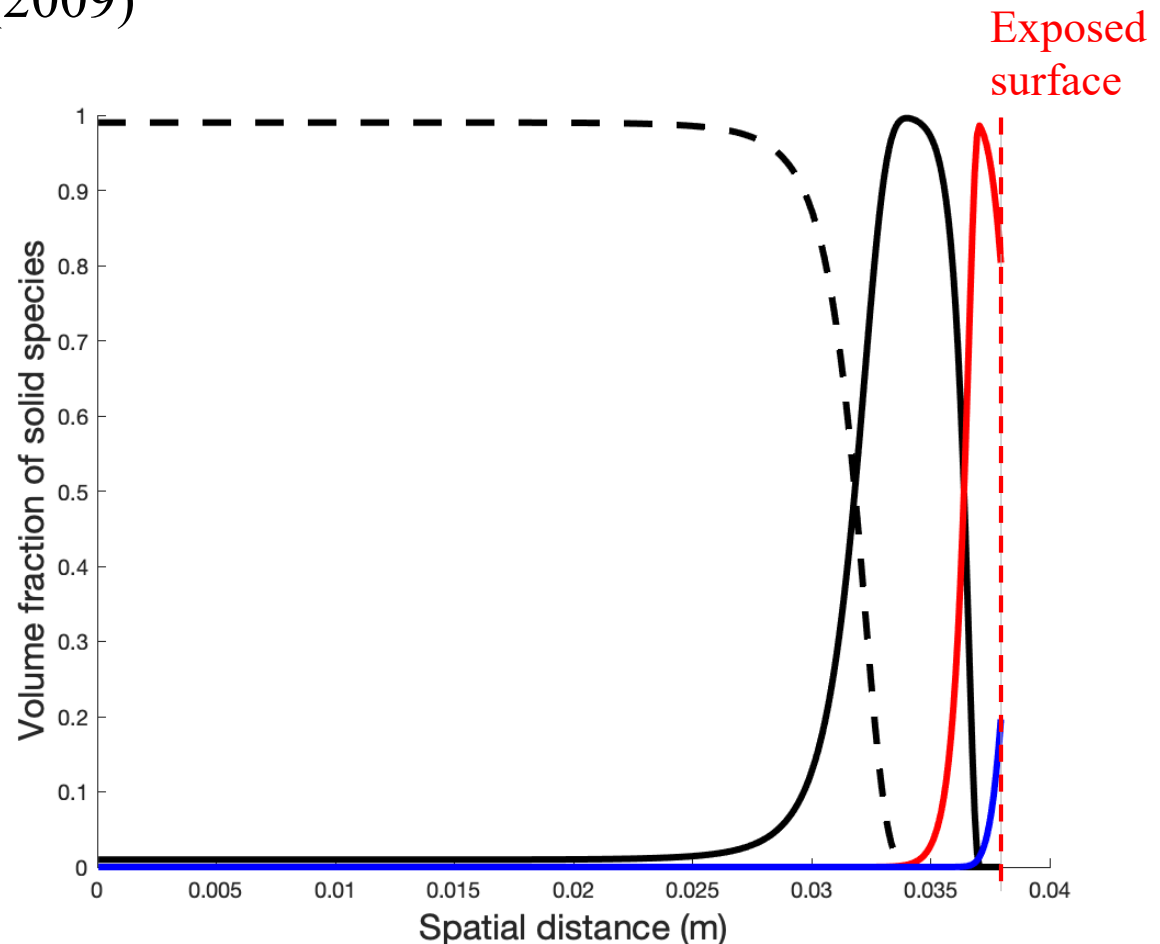
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Modeling of Pyrolysis



- Modeling of the fuel mass loss rate (MLR)

- Advanced approach

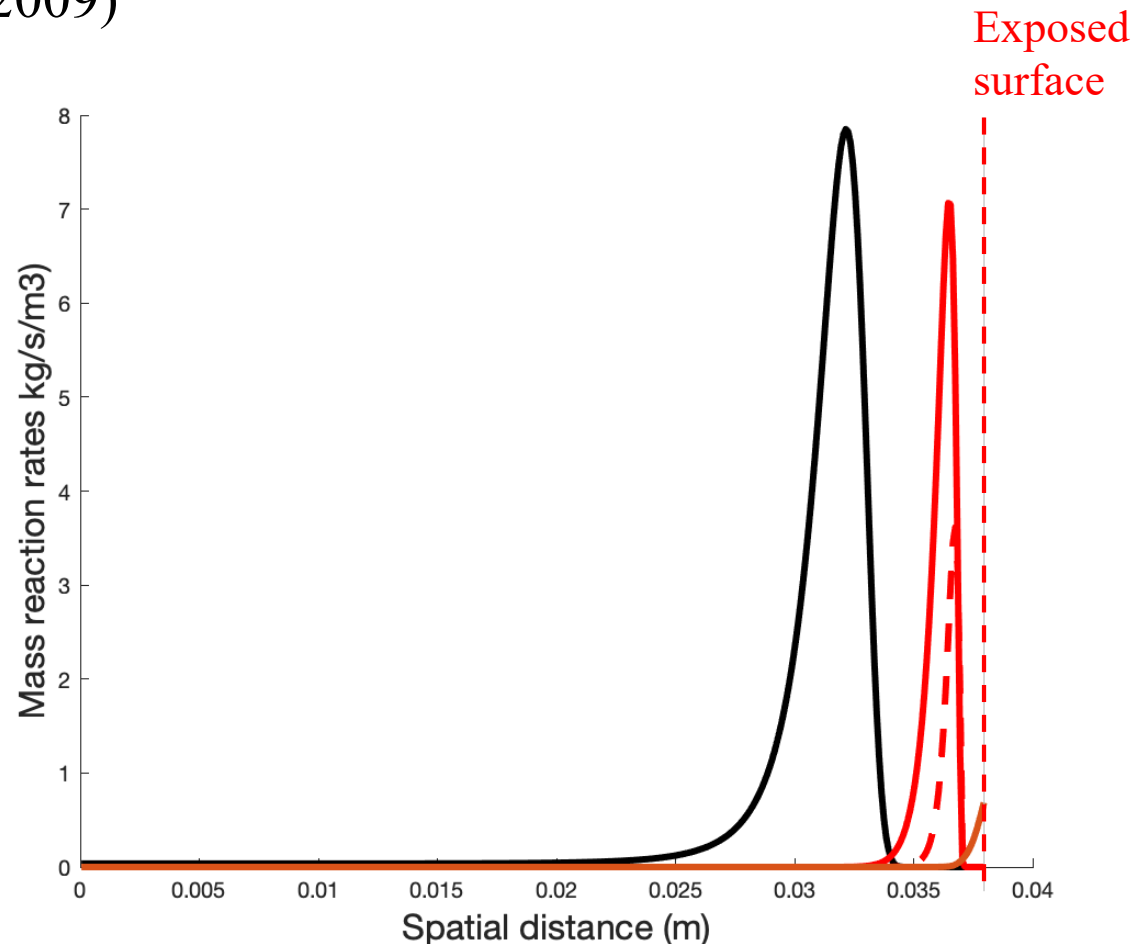
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$$\Delta = 3.8 \text{ cm}$$

$$G = 40 \text{ kW/m}^2$$

$$x_{O_2,g} = 0.21$$

$$t = 100 \text{ s}$$



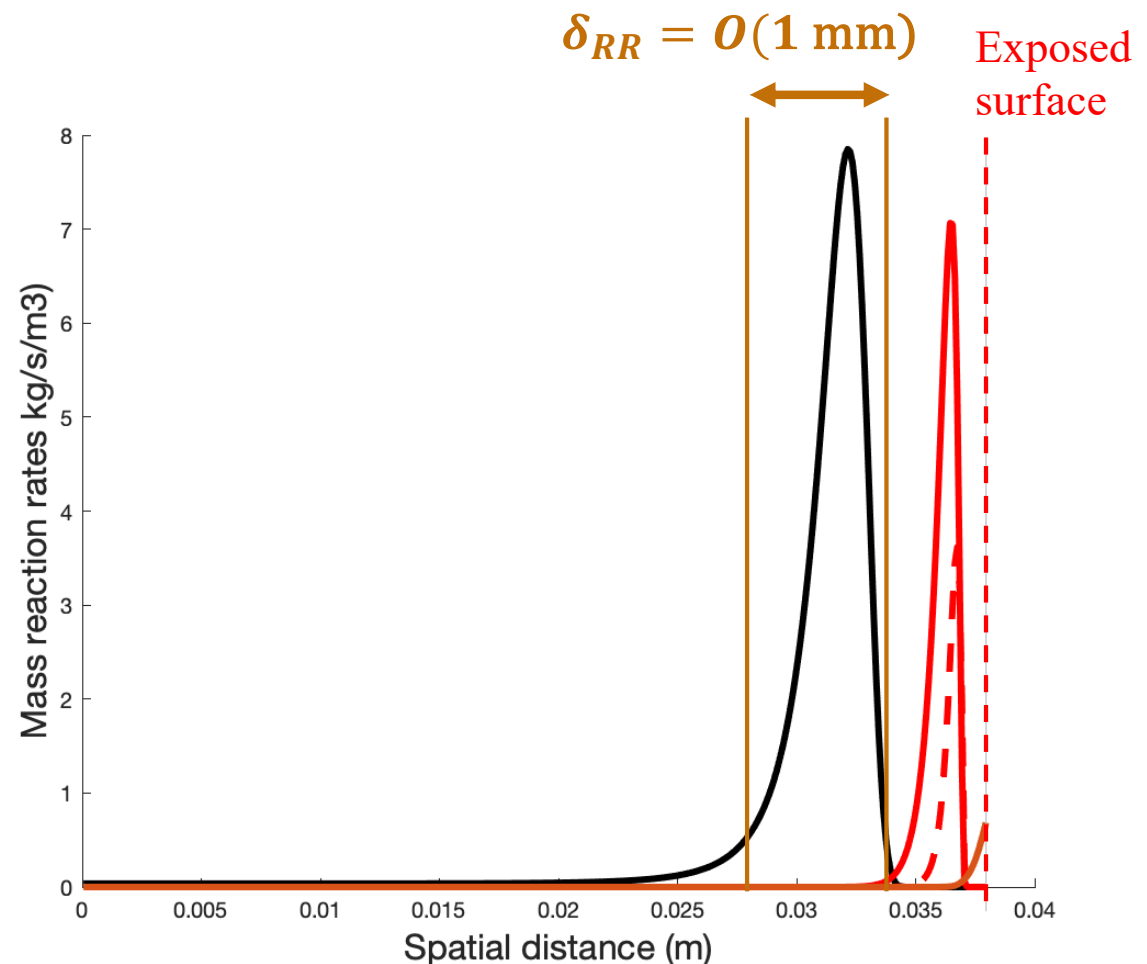
Modeling of Pyrolysis



- Control of numerical accuracy

- The accuracy of the solution of the solid phase equations is controlled by the spatial resolution

$$\Delta x_s \leq 100 - 200 \mu m$$



Compartment Fire Modeling



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Examples



• Solvers

	FireFOAM ¹	FDS ²
Scheme	Second-order accurate, finite volume solver with implicit time integration	Second-order accurate, finite difference solver with explicit time integration
Turbulence	<i>k</i> -eqn model (default), dynamic Smagorinsky, WALE, Deardorff	Deardorff (default), dynamic Smagorinsky
Combustion	Global combustion eqn, Eddy Dissipation Model (EDM)	Global combustion eqn, Eddy Dissipation Model (EDM)
Radiation	DOM-FVM (prescribed radiant fraction, grey medium model, WSGG model)	DOM-FVM (prescribed radiant fraction, grey medium model, wide band model)
Soot	Flamelet-based model	Soot yield model
Pyrolysis	1D solid phase model	1D solid phase model
Mesh	Structured and unstructured grid	Structured (Cartesian) grid

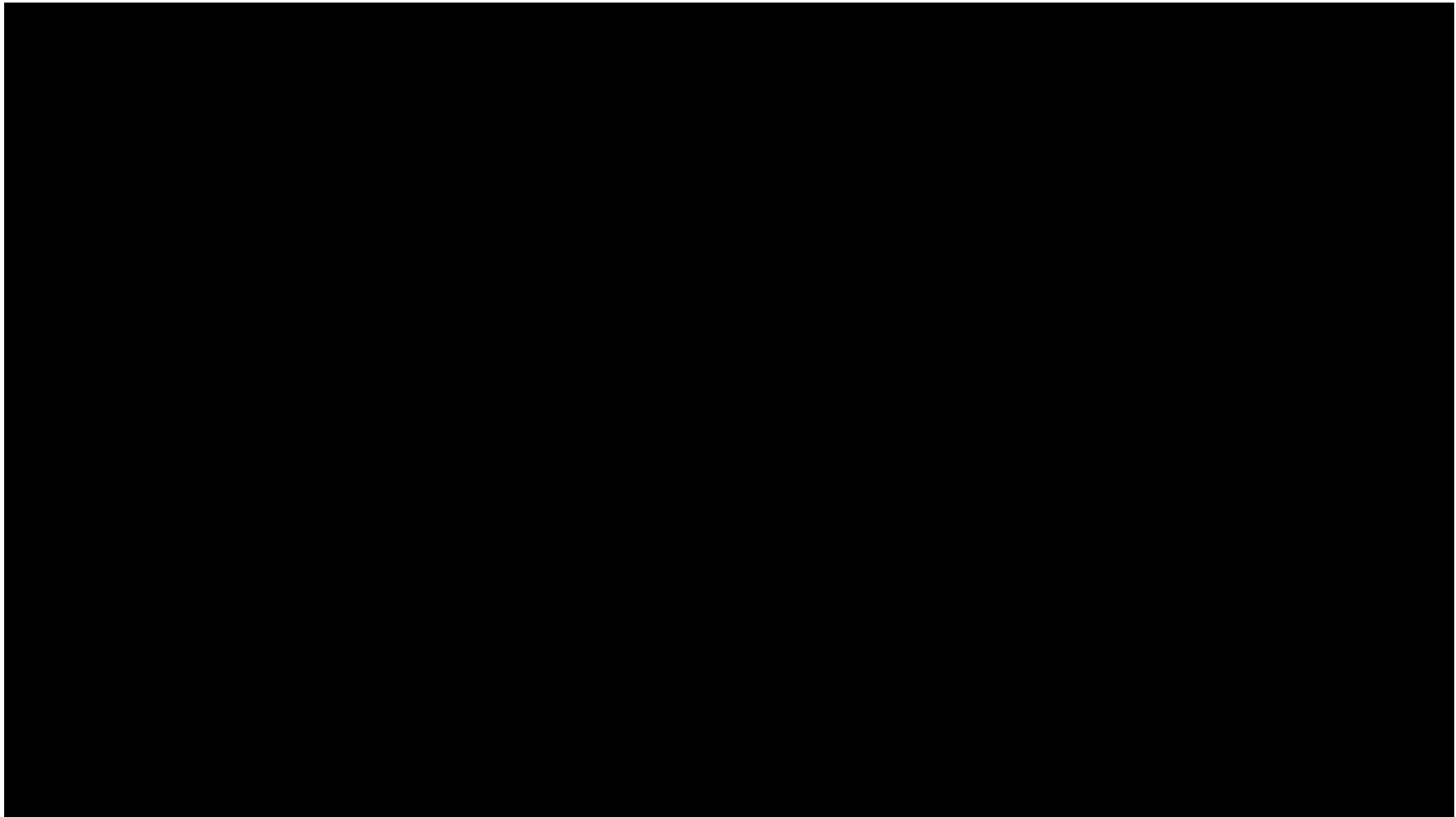
¹ FM Global (USA), FireFOAM, Available from: <https://github.com/fireFoam-dev>

² NIST (USA), FDS, Available from: <https://pages.nist.gov/fds-smv/>

Examples



- Fine-grained LES (research-level) (FireFOAM)
 - UMD experiment on flame suppression by inert gas
 - Flame structure: $L_f \sim 0.5$ m; $L_{eddy} \sim O(1-10$ cm); $U_{eddy} \sim O(1$ m/s)
(Vilfayeau, White, Sunderland, Marshall, Trouvé, *Fire Safety J.*, 2016)



Examples



- Fine-grained LES (research-level) (FireFOAM)
 - Univ. Waterloo/NIST experiment on structure of pool fires
 - Flame structure: $L_f \sim 0.5$ m; $L_{eddy} \sim O(1-30 \text{ cm})$; $U_{eddy} \sim O(1 \text{ m/s})$

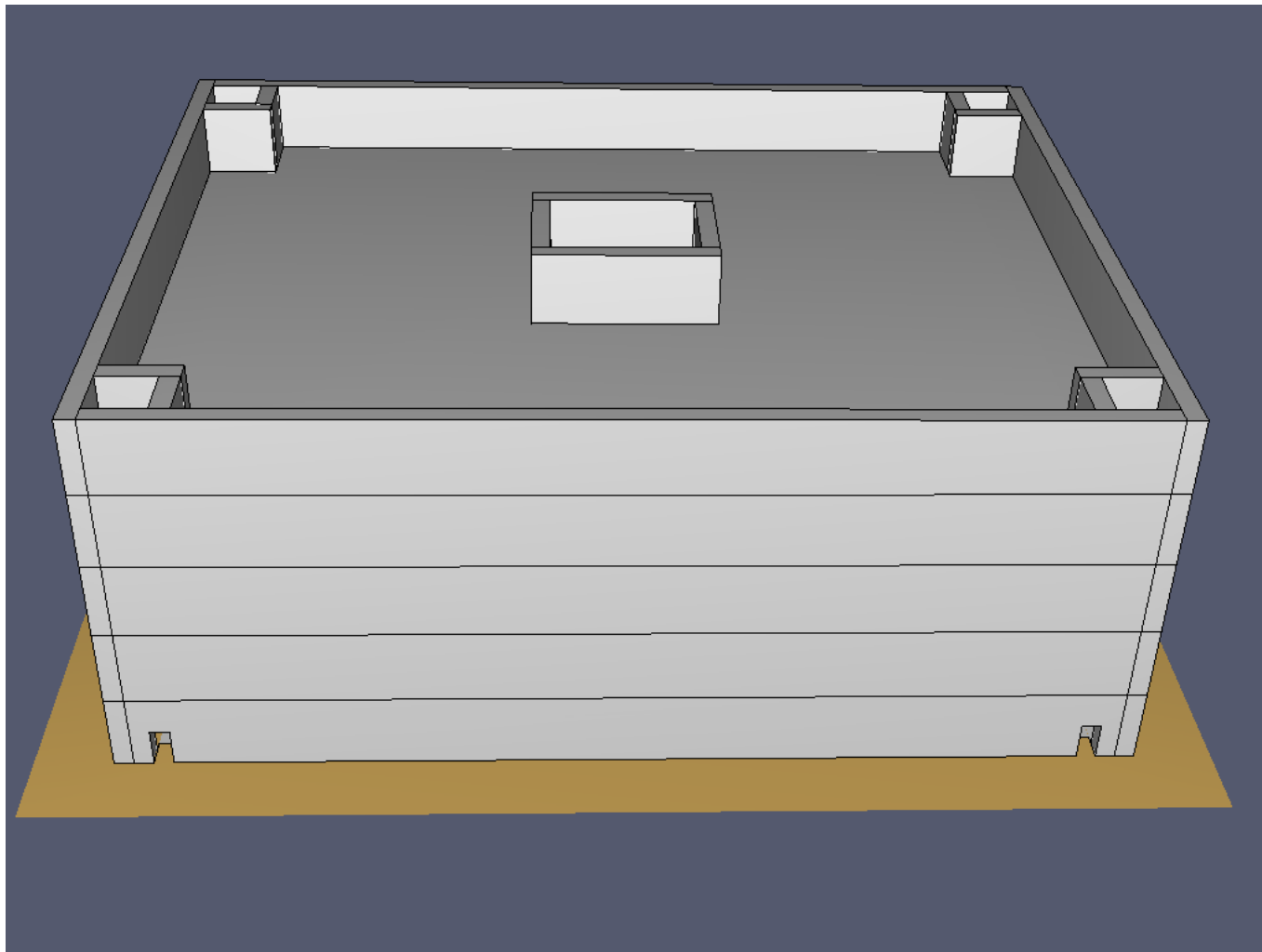
(Ahmed & Trouvé, submitted to *Combust. Flame*, 2020)



Examples



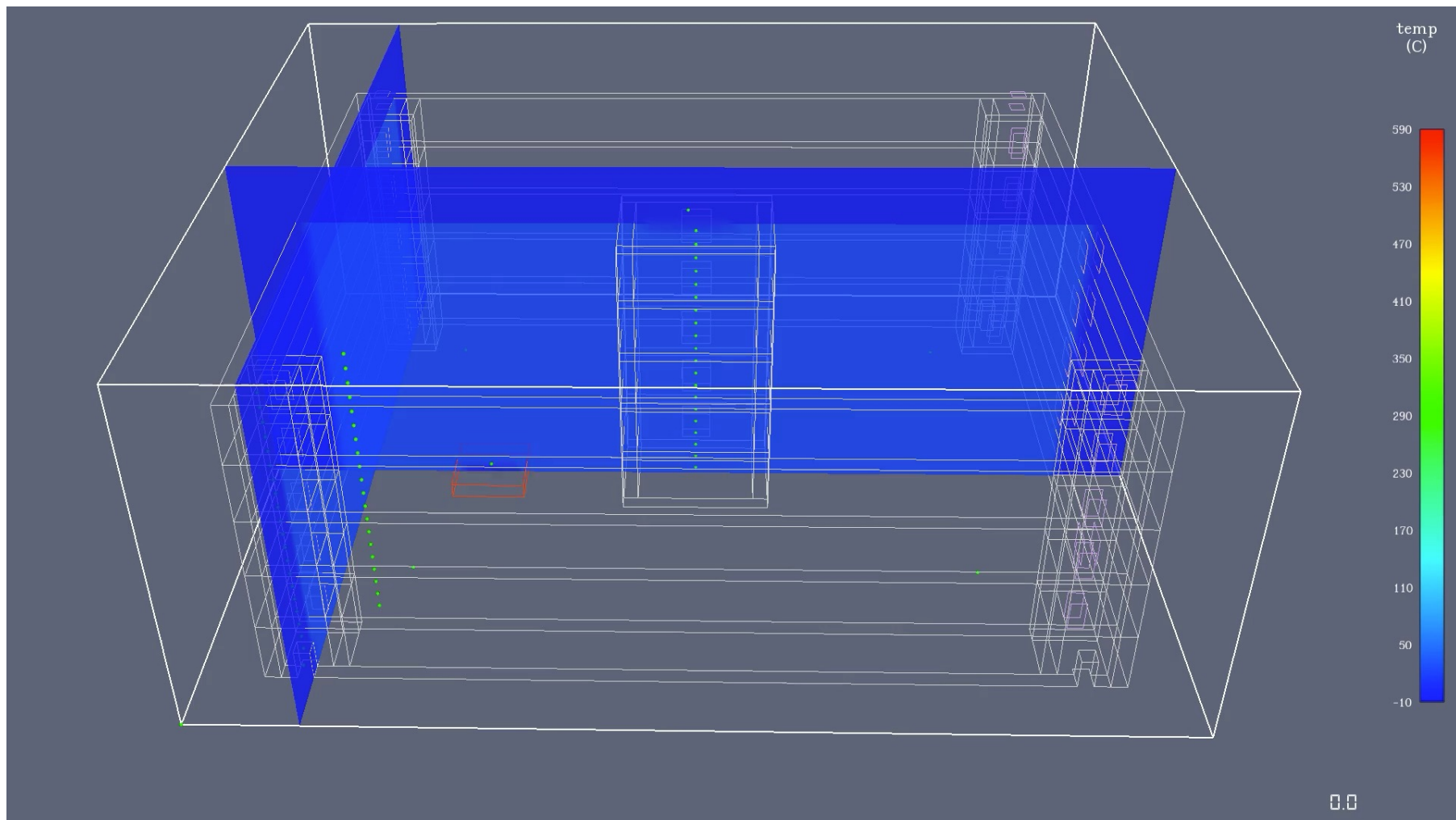
- Coarse-grained LES (engineering-level) (PyroSim/FDS)
 - Study of fire smoke dynamics in commercial 5-story building
 - Building: $L \sim O(10\text{s m})$; pressure effects; leakage paths



Examples



- Coarse-grained LES (engineering-level) (PyroSim/FDS)
 - Study of fire smoke dynamics in commercial 5-story building
 - Building: $L \sim O(10\text{s m})$; pressure effects; leakage paths



Conclusion



- Fire modeling has experienced a remarkable growth in the past two decades
 - Fueled in particular by the development of FDS (and also now FireFOAM) and the development of open-source CFD software
 - Fire models have become routine fire safety engineering tools
- Fire modeling features several technical challenges
 - Modeling of complex (solid) fuel sources (pyrolysis processes)
 - Relatively slow, buoyancy-driven flow
 - Combustion with flame extinction/ignition
 - Boundary layer flames
 - Soot formation
 - Spectrally-resolved radiation and turbulence-radiation interactions
 - Water spray
 - Flame and smoke chemistry effects (toxicity)

Conclusion



- Organizational challenge

- The fire modeling community is small, fragmented, geographically dispersed, without a history of well-defined standards and without a consensus on well-defined objectives
- There is a need for a coordinated effort to organize and strengthen the fire modeling community

- MaCFP

- The IAFSS Working Group on Measurement and Computation of Fire Phenomena (the MaCFP Working Group) (<http://www.iafss.org/macfp/>)
 - A recent initiative endorsed by the International Association for Fire Safety Science (IAFSS, <http://www.iafss.org>)
 - A forum between experimentalists and modelers to establish a common framework around the topic of CFD validation
 - A regular series of workshops
 - A list of community-approved experiments
 - A data repository (<https://github.com/MaCFP>)