

CentraleSupélec



# *Virtual chemistry for temperature and pollutant prediction in LES of non-adiabatic turbulent flames*

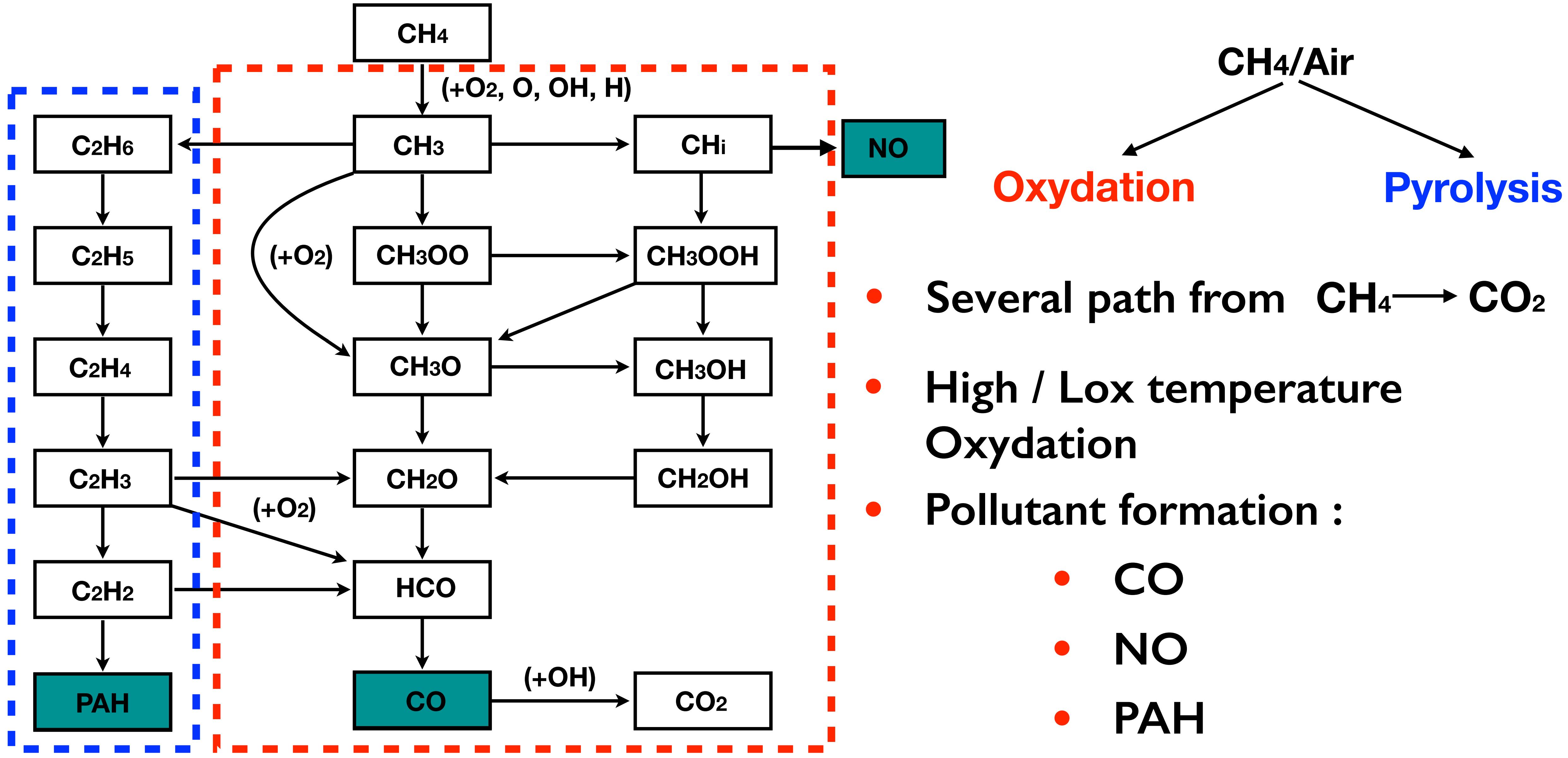
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1) Laboratoire EM2C, 3 rue Joliot Curie 91192 Gif Sur Yvette cedex, France

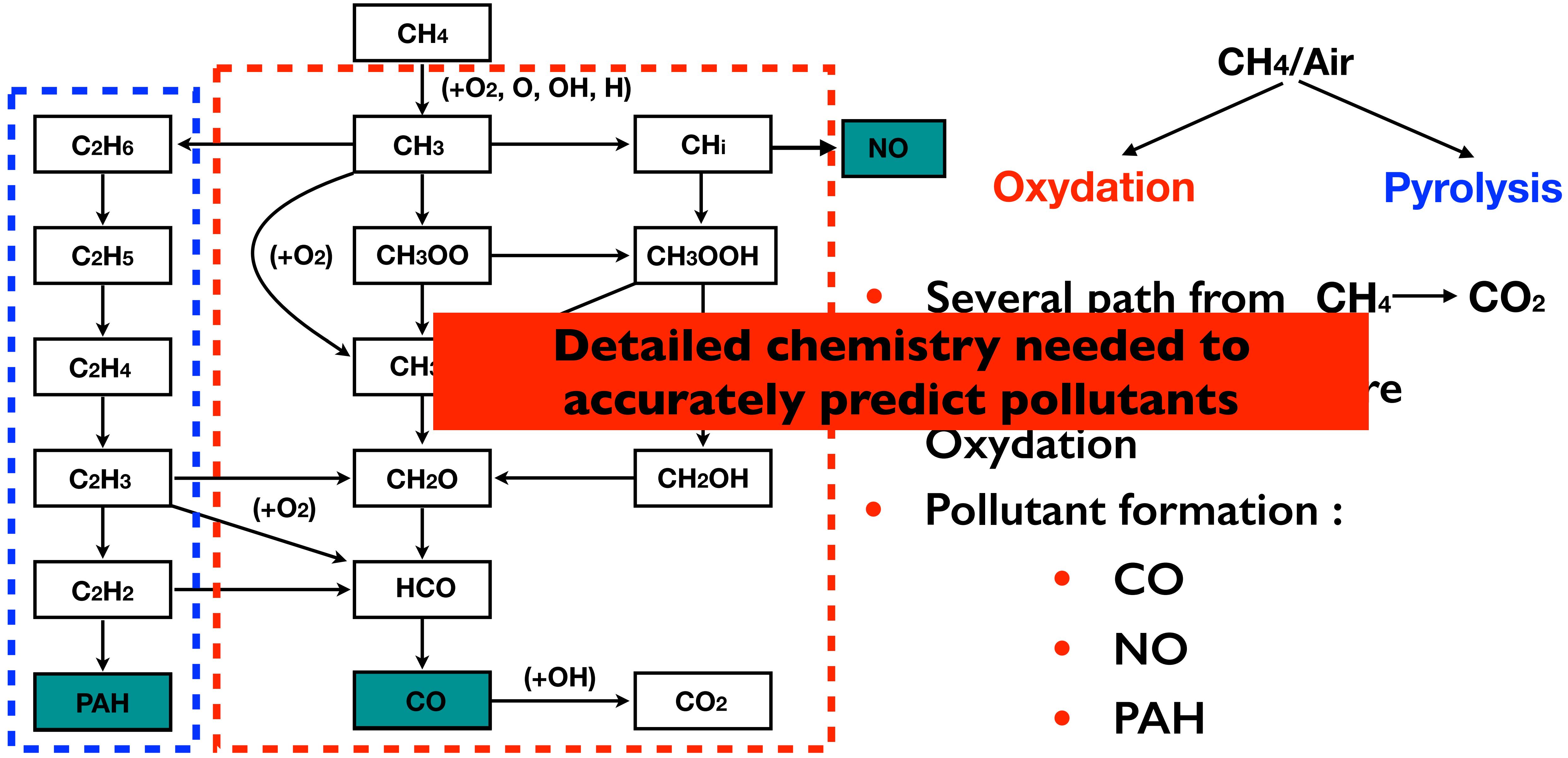
2) SAFRAN Tech, Rue des Jeunes Bois, Châteaufort - CS 80112, 78772 Magny-les-Hameaux, France

*Journées conjointes Groupement de Recherche CNRS Feux - Groupement Français de Combustion*

# Hydrocarbon/O<sub>2</sub> kinetic model - pollutant formation



# Hydrocarbon/O<sub>2</sub> kinetic model - pollutant formation



# *Pollutant formation prediction in large scale application*

- Turbulent flow field
- Air/Fuel stratification
- Multiple combustion regimes
- Heat losses etc...

**CO :**

- Intermediate / Product species

- Detailed chemistry too expensive
- Reduced order chemistry models needed to limit the simulation CPU cost

**NO :**

- Fast / Long time scales
- Slow recombination / Reburning

# *Chemistry modeling in LES*

- Strategies employed in Large Eddy Simulation to reduce CPU cost associated to combustion chemistry modeling:

## → **Global optimized mechanisms**

Westbrook et al., (1981)  
Jones et al., (1988)  
Franzelli et al., (2010)

- Low CPU cost
- Temperature, flame speed
- Limited range of operating conditions
- Pollutant...

## → **Tabulated chemistry**

Peters (1984)  
Gicquel et al., (2000)  
Van Oijien et al., (2001)  
Pierce and Moin (2005)  
Fiorina et al. (2010)

- Detailed chemistry
- Flamelet regime
- Multiple coordinates, Flame index

## → **Systematically reduced chemistry:**

### **1. *Skeletal***

Luche (2003)  
Lu and Law (2005)  
P. Pepiot and H. Pitsch(2008)  
Jaravel et al., (2016)

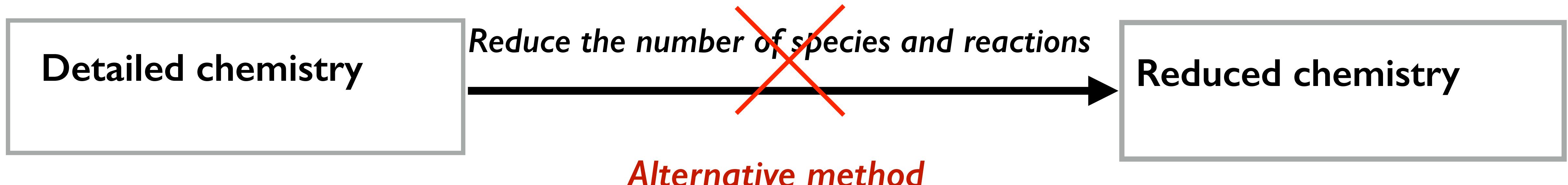
### **2. *Analytically reduced***

- Accurate flame structure description including pollutants

- High CPU cost
- Size scales with fuel complexity

# *Reduced order chemistry models in LES*

- Define a new reduced transported species method:



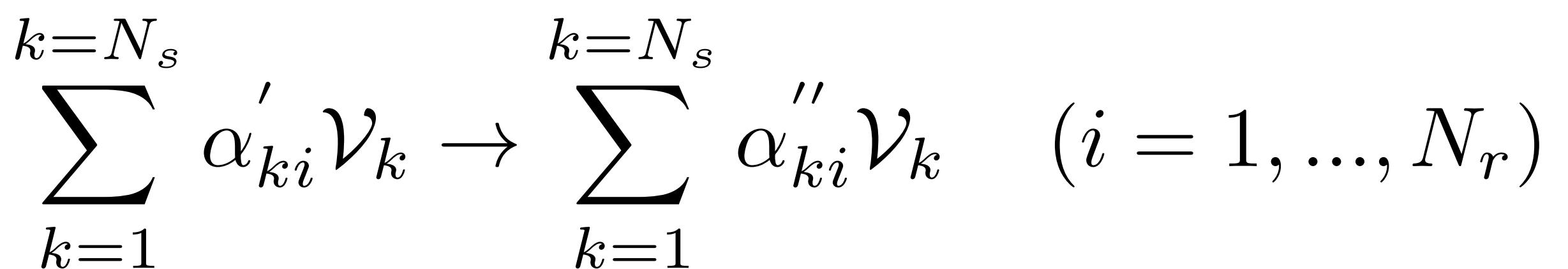
## **Virtual chemistry (M. Cailler et al. 2017)**

- Build-up reduced chemical mechanisms from scratch
- Use virtual species and virtual reactions
- Build-up dedicated sub-mechanisms to predict flame quantities of interest.

# Virtual kinetic mechanisms

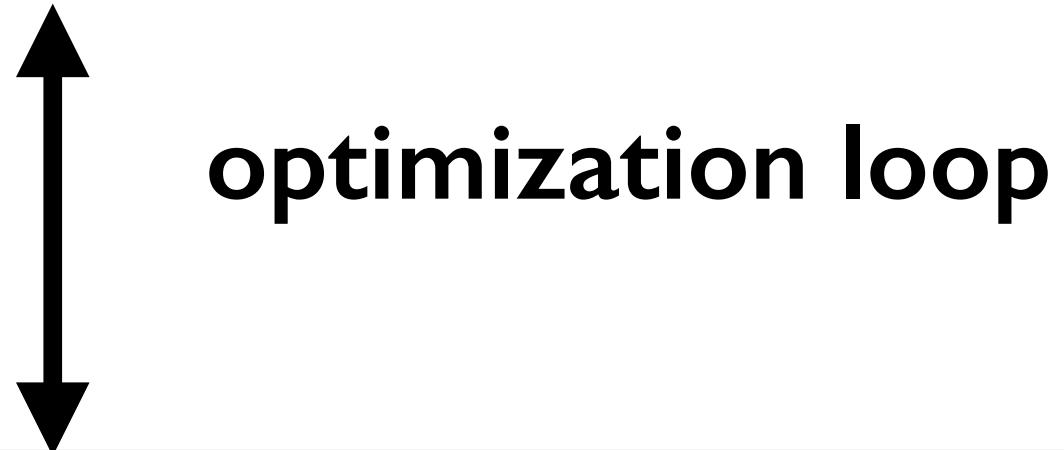
- Virtual kinetic scheme composed by:

- $N_s$  virtual species  $\mathcal{V}_k$
- $N_r$  virtual reactions



- The strategy relies on optimizing:
  - the **number of virtual species and virtual reactions**
  - **thermodynamic properties** of virtual species
  - **reaction rate parameters** of the virtual scheme

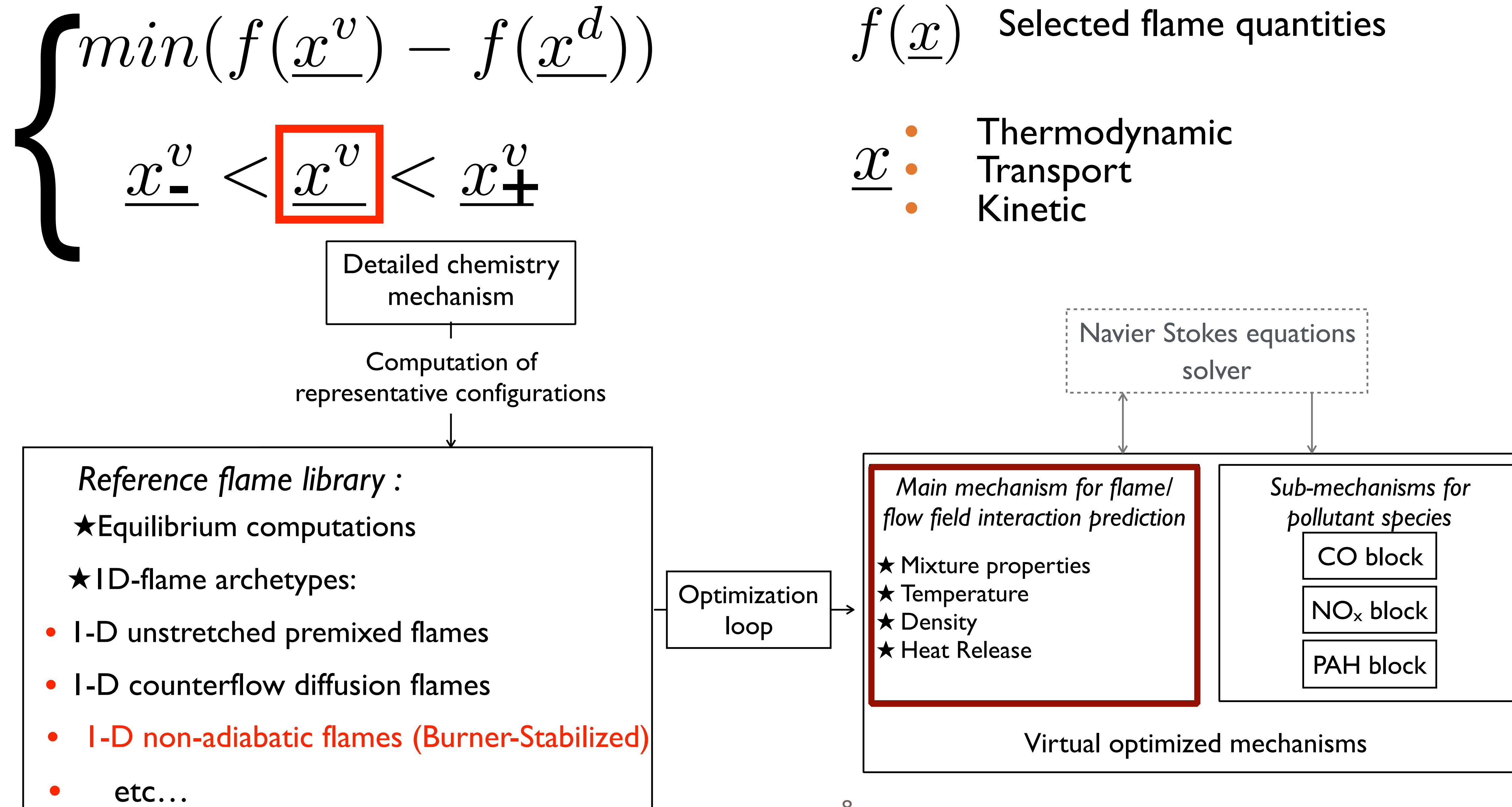
► Species properties  
 $c_{pk}, h_k, \lambda_k, D_k$   
► Kinetic rate parameters  
 $A_i, E_{ai}, F_i^k$



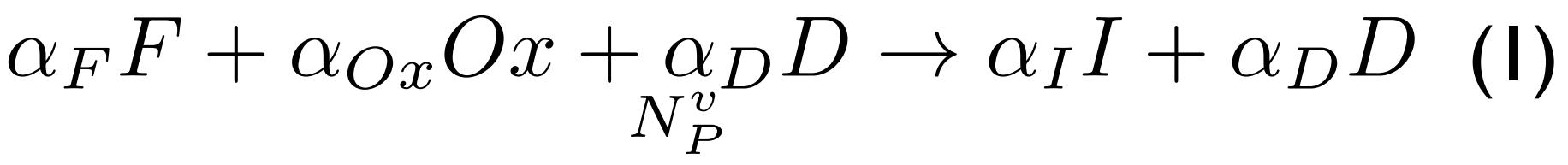
Reference database computed with detailed chemistry:

- 0D Reactors
- Equilibrium calculations
- 1D laminar calculations (premixed and/or non premixed, non-adibatic, etc...)

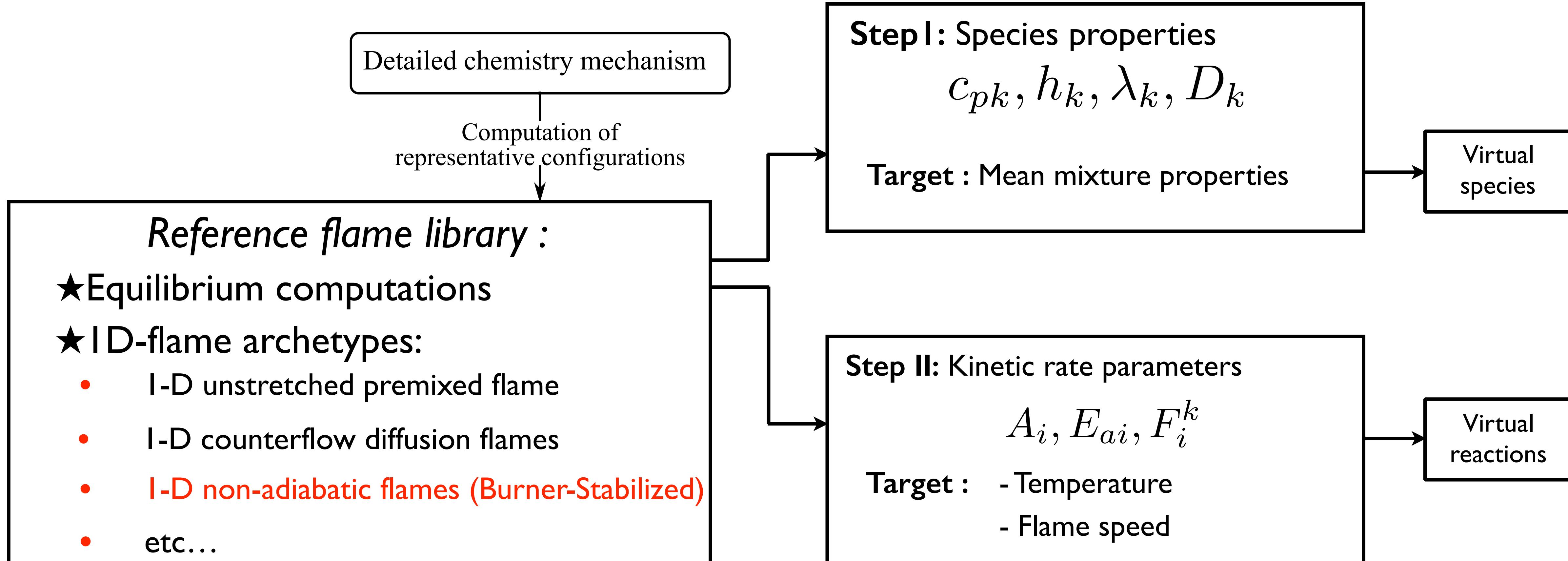
# Virtual optimized mechanisms methodology



# Main virtual mechanism generation procedure



$$I \rightarrow \sum_{k=1} \alpha_{P_k} P_k \quad (2)$$



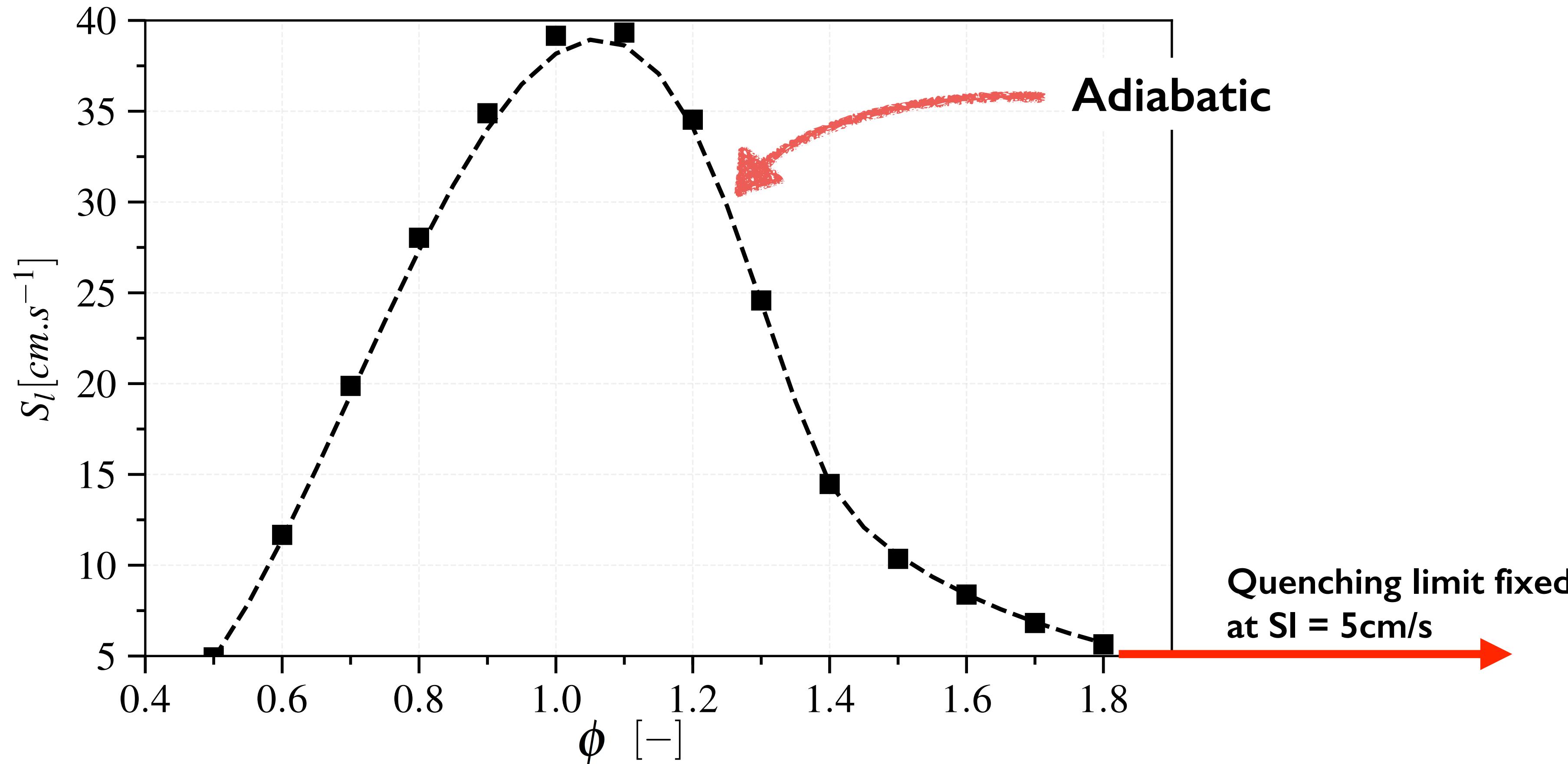
# Virtual optimized chemistry: Adiabtic/Non-adiabatic conditions

Validation versus the reference solution (GRI3.0)

→ Laminar flame consumption speed

→ Freely propagating

- - - Reference AD solution
- Reference NAD solution
- NAD Virtual - mechanism



# Virtual optimized chemistry: Adiabtic/Non-adiabatic conditions

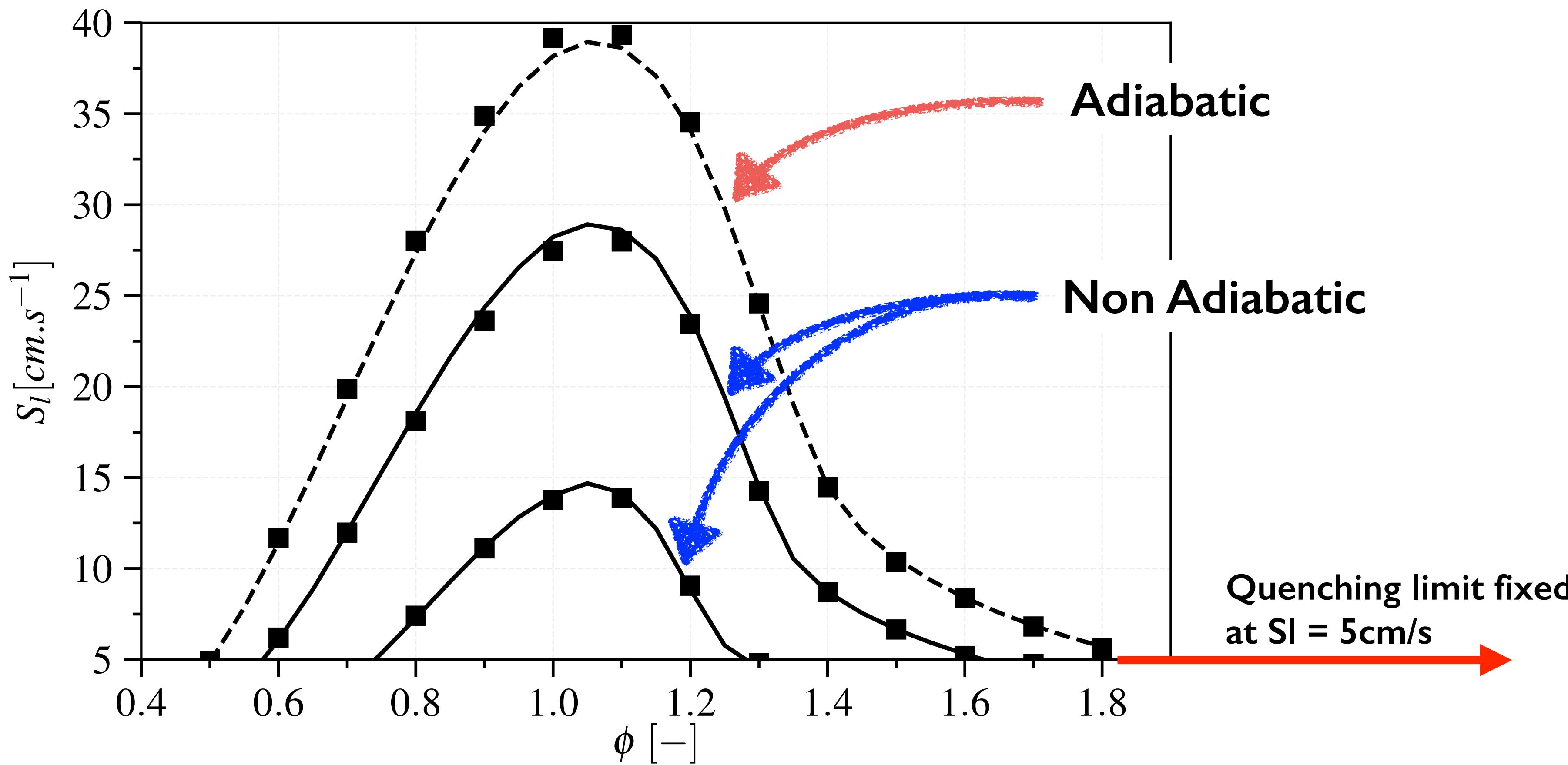
Validation versus the reference solution (GRI3.0)

→ Laminar flame consumption speed

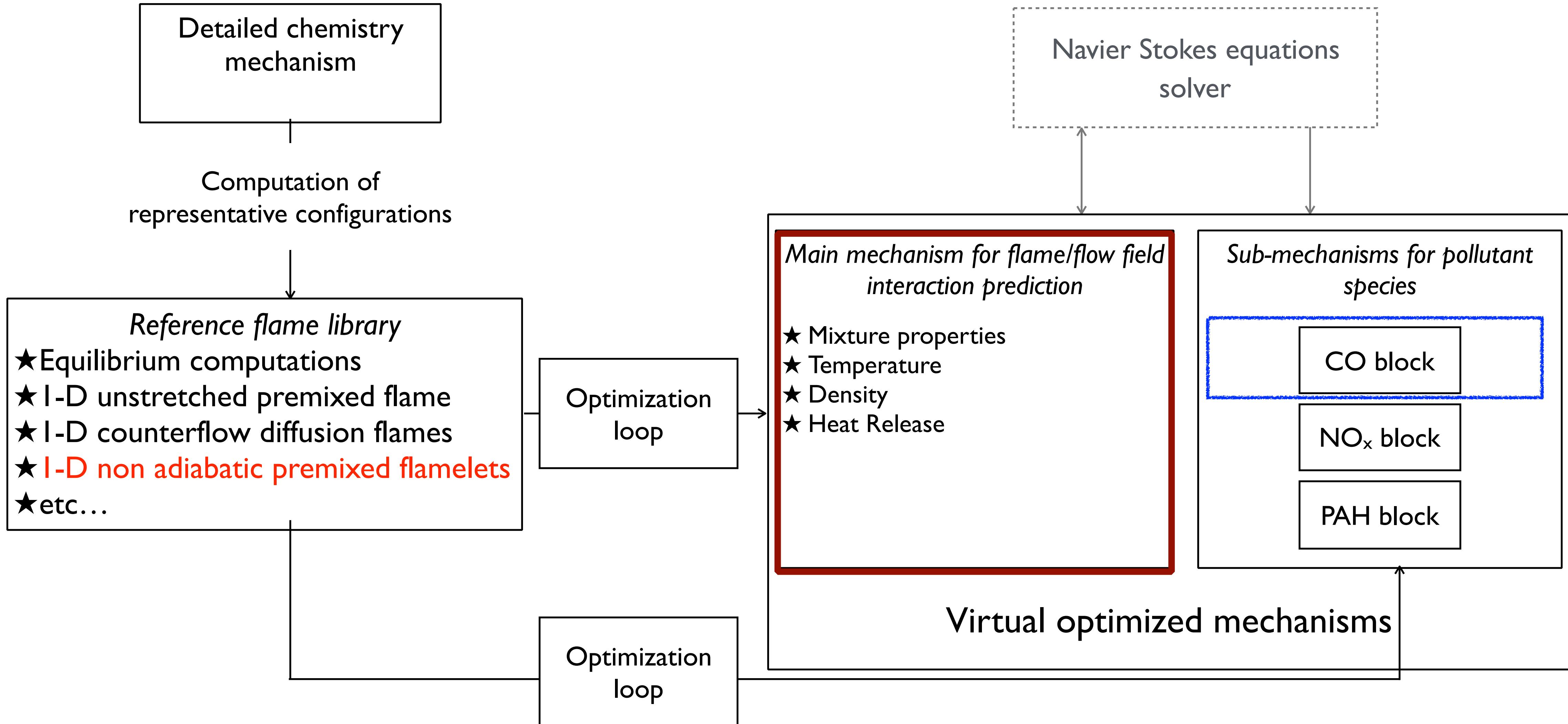
→ Freely propagating

→ Burner-stabilized

- - - Reference AD solution
- Reference NAD solution
- NAD Virtual - mechanism

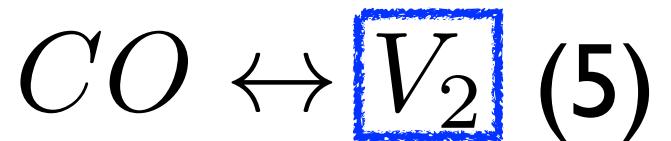
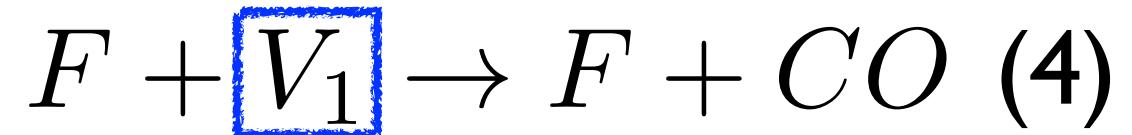


# Virtual optimized mechanisms: pollutants (CO)



# ***CO pollutant sub-mechanism***

- **3 step mechanism :**



- **Rate of progress**

$$q_3 = A_3 f_3(\phi, \Delta h) \exp\left(\frac{-E_{a,3}}{RT}\right) [F]^{F_F^3} [Ox]^{F_{Ox}^3}$$

$$q_4 = A_4 f_4(\phi, \Delta h) \exp\left(\frac{-E_{a,4}}{RT}\right) [F]^{F_F^4} [V_1]^{F_{V_1}^4}$$

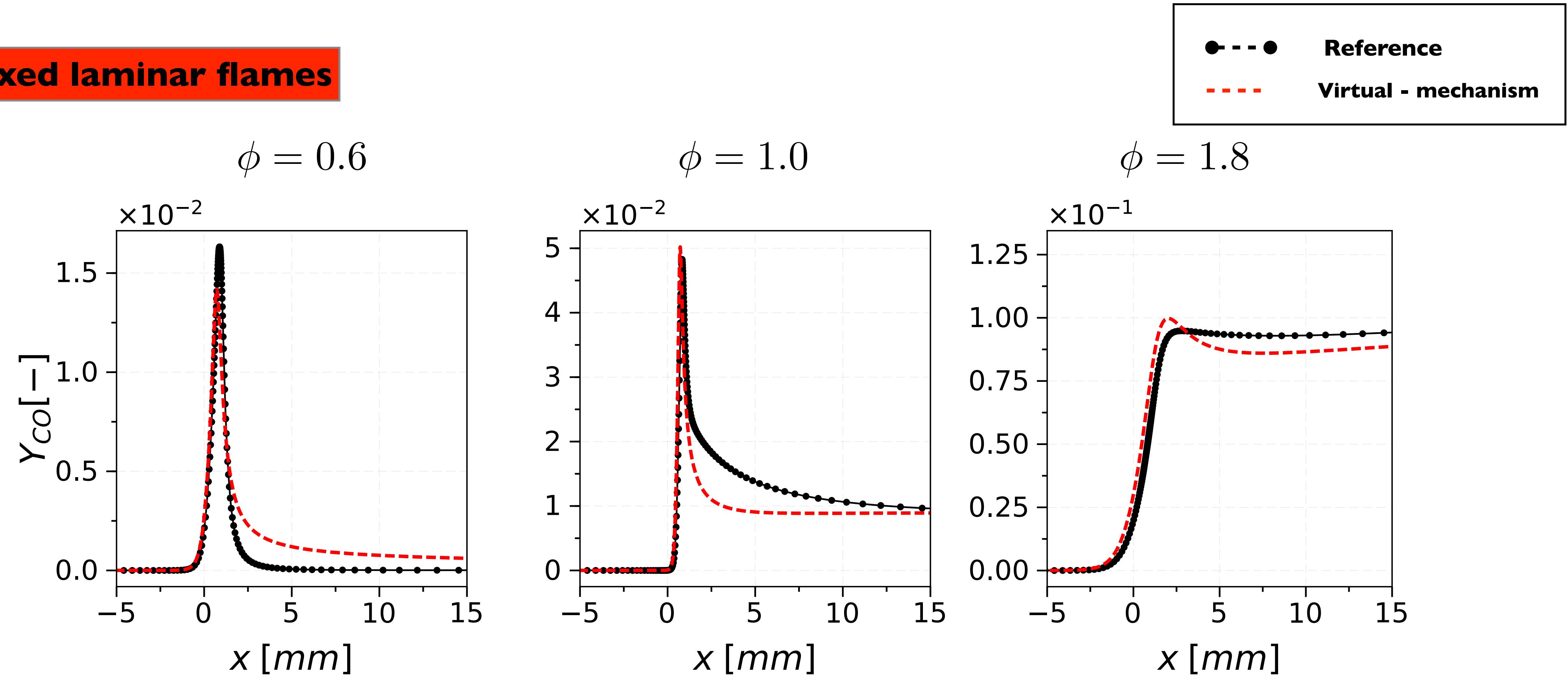
$$q_5 = A_5 f_5(\phi, \Delta h) \exp\left(\frac{-E_{a,5}}{RT}\right) \left( [CO]^{F_{CO}^5} [V_2]^{F_{V_2}^5} - \frac{[CO]^{R_{CO}^5} [V_2]^{R_{V_2}^5}}{K_{c,5}^v(\phi, \Delta h)} \right)$$

capture Y<sub>CO</sub> at equilibrium

# Validation in adiabatic conditions

CH4/air combustion at atmospheric pressure and T fresh of 300K

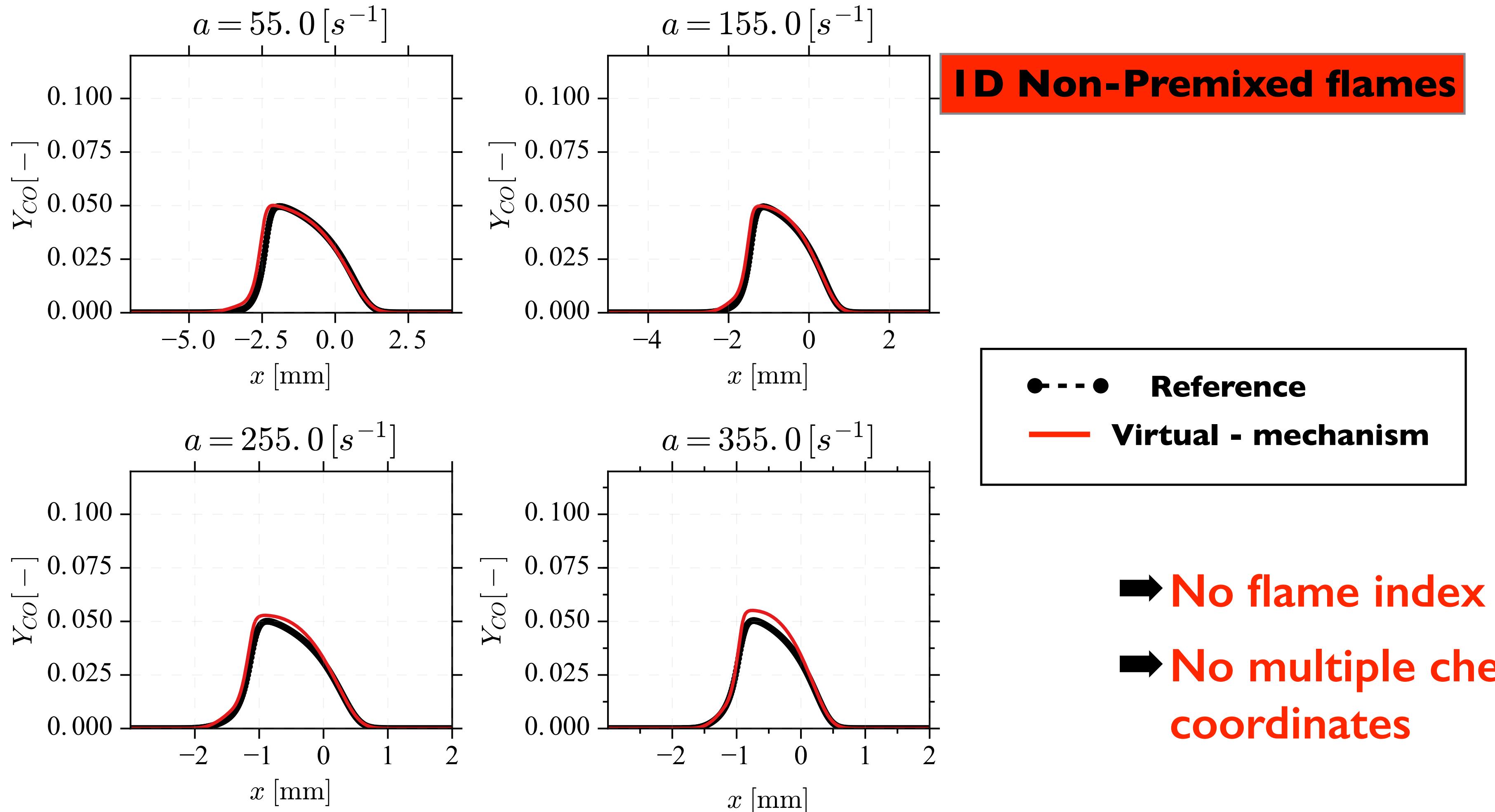
## ID Premixed laminar flames



- M. Cailler, N. Darabiha, D. Veynante and B. Fiorina. Building-up virtual optimized mechanism for flame modeling. Proceeding of the Combustion Institute (2017)
- M. Cailler, N. Darabiha and B. Fiorina. Virtual chemistry for pollutant emissions prediction. Submitted to Combustion and Flame

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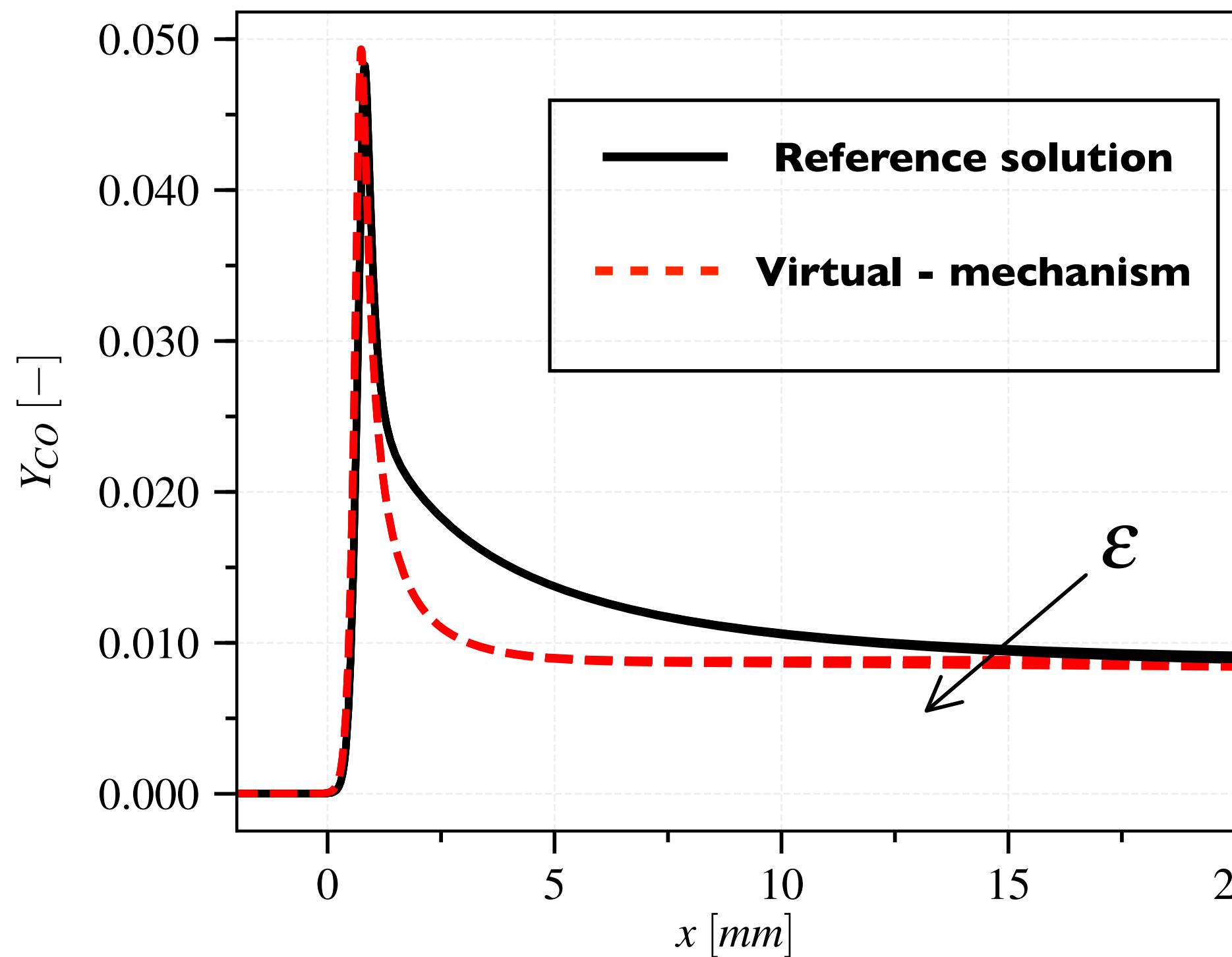
# 1-D premixed flame submitted to radiative heat losses

- A radiation sink is added to the energy balance equation of the 1-D flame solver

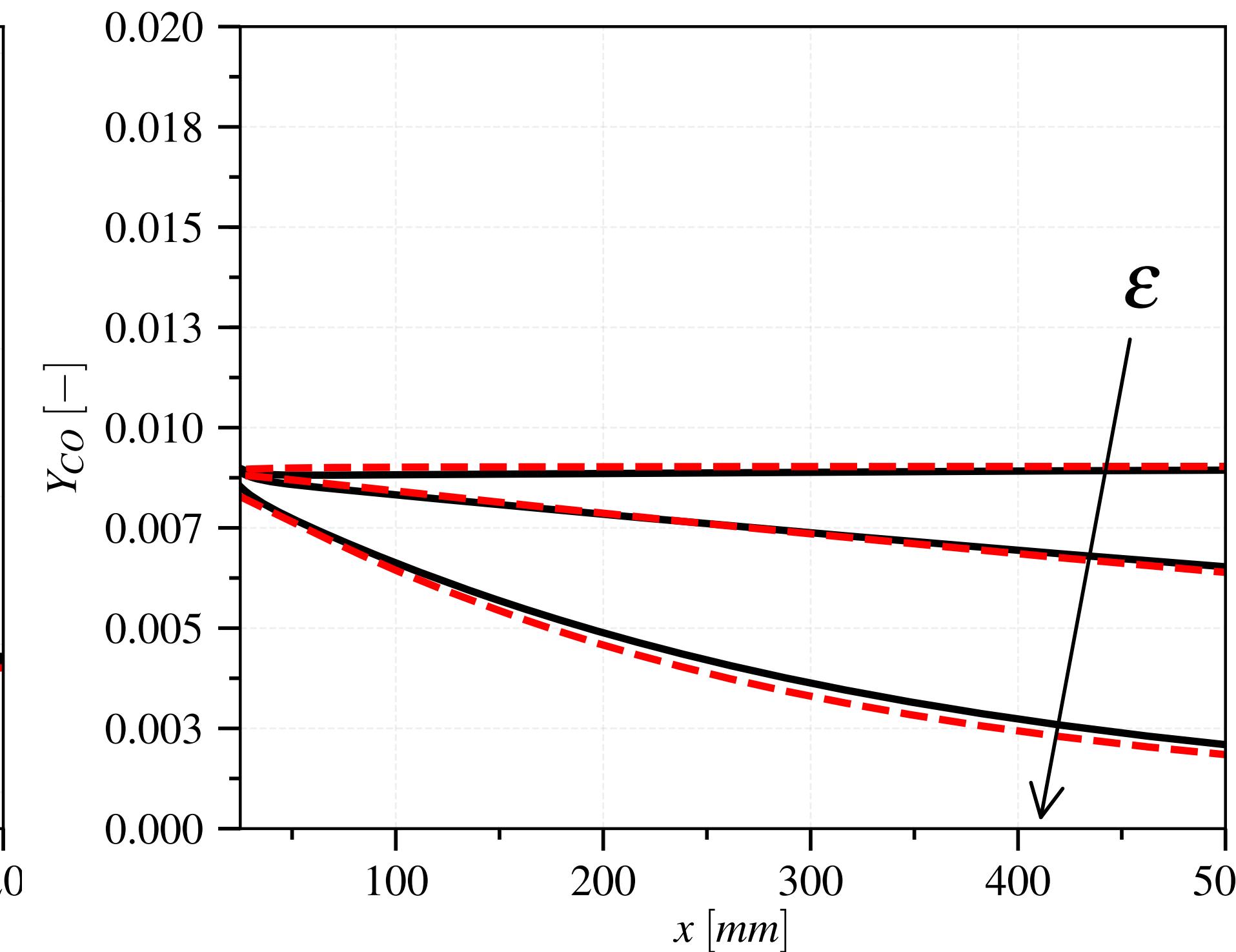
$$u\rho c_p \frac{dT}{dx} - \frac{d}{dx} \left( \lambda \frac{dT}{dx} \right) + \sum_k (\rho Y_k V_k c_{p_k}) \frac{dT}{dx} + \sum_k h_k \dot{\omega}_k + \sigma \epsilon (T^4 - T_0^4) = 0$$

**ID Non-adiabatic**

→ CO mass fraction / Flame Front

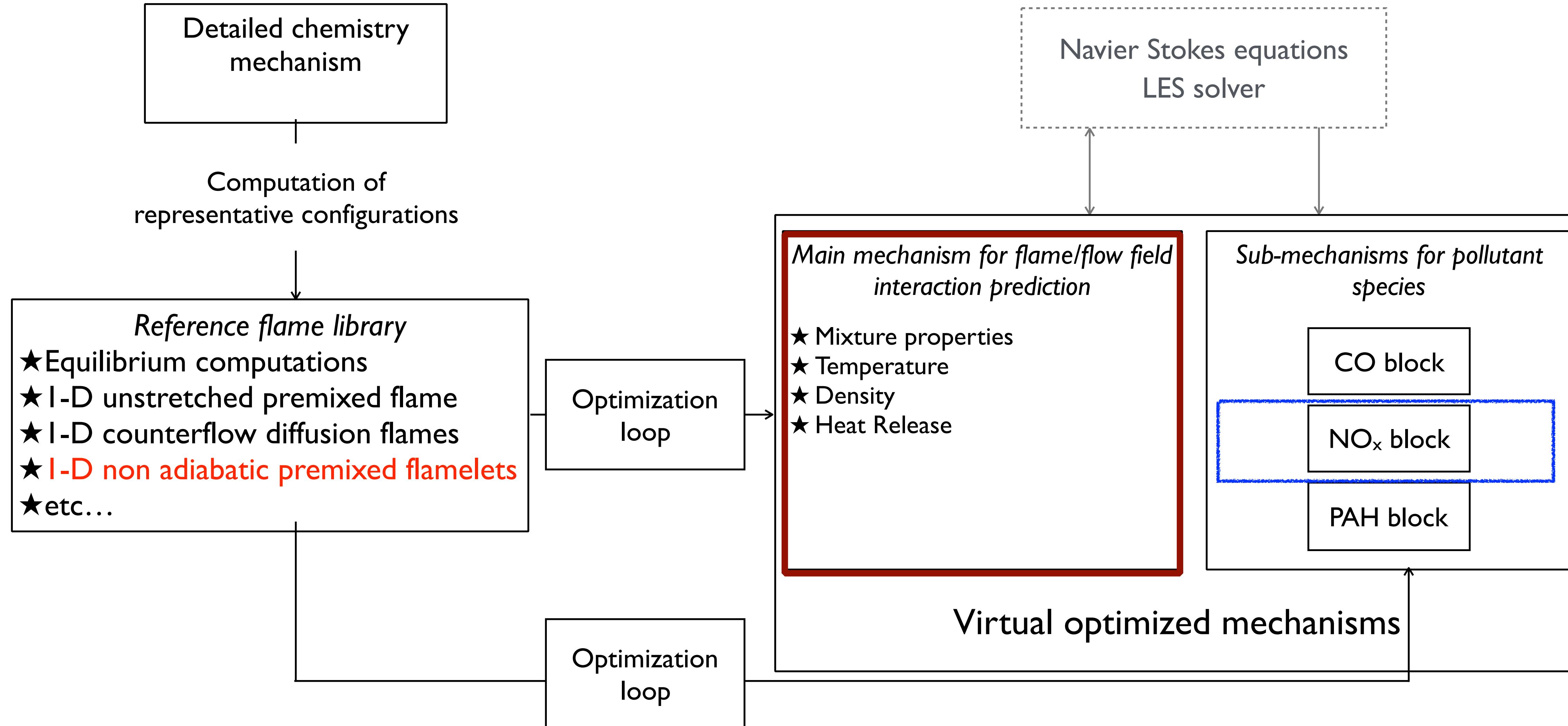


→ CO mass fraction / Post Flame



- CO production well captured in the post-flame region / local equilibrium change

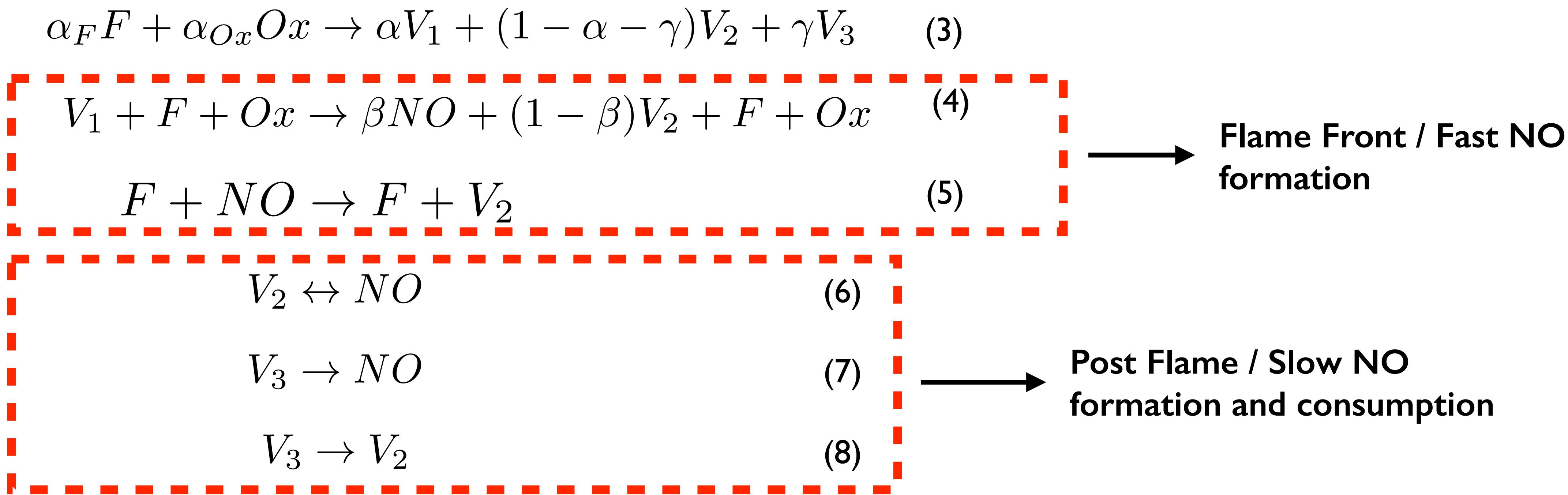
# Virtual optimized mechanisms: pollutants (NO)



# *Virtual sub-mechanism for NO prediction*

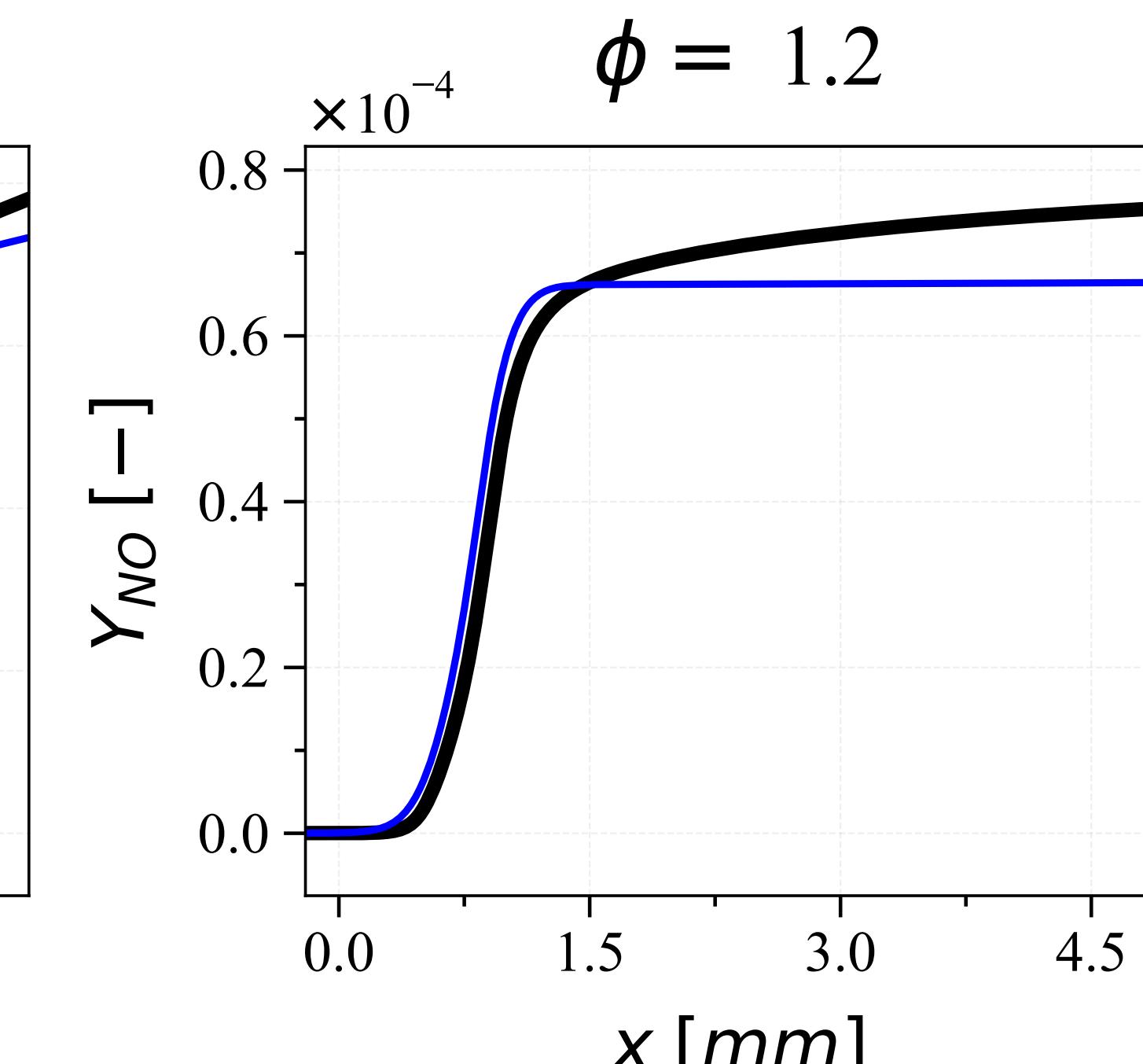
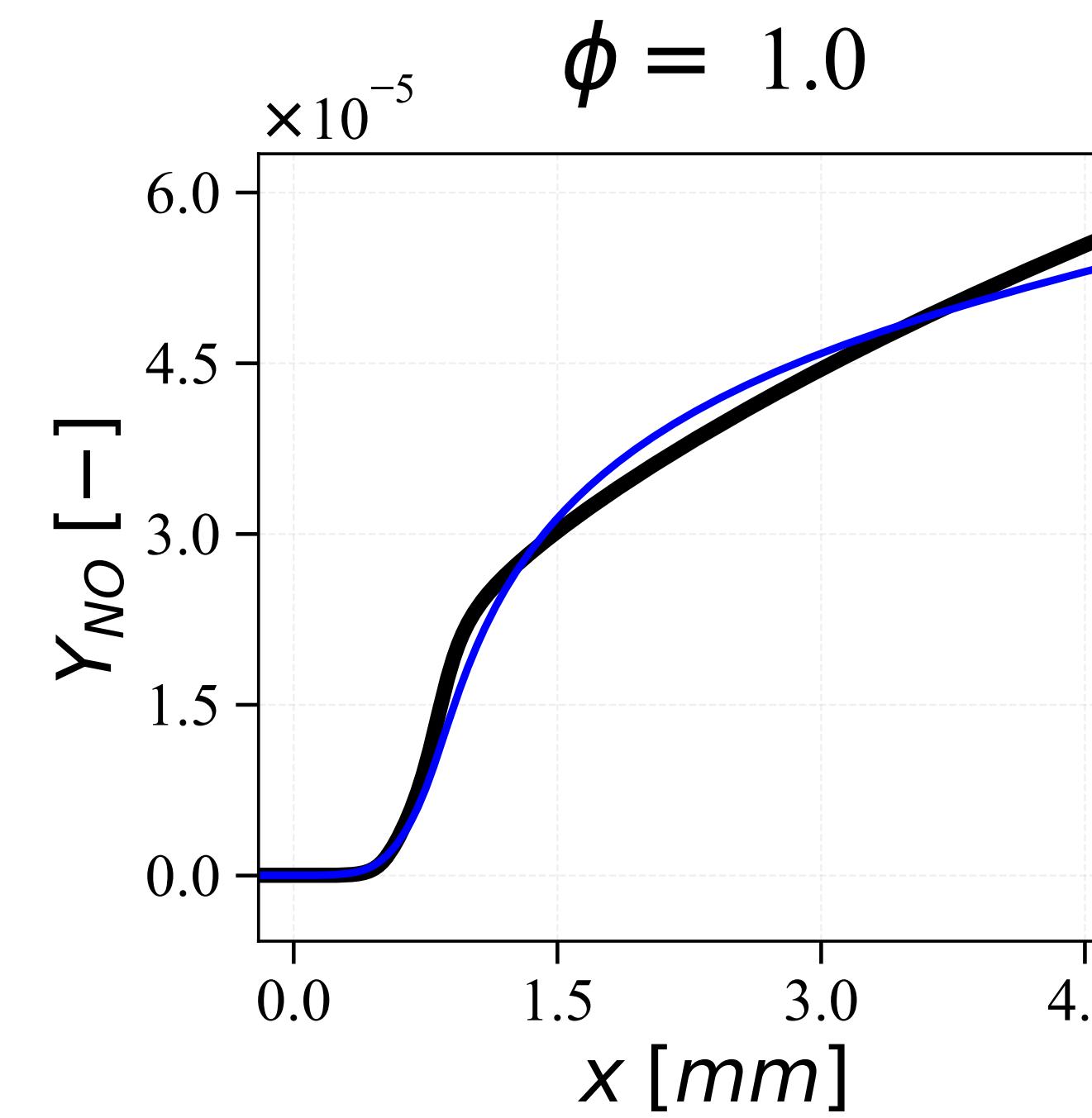
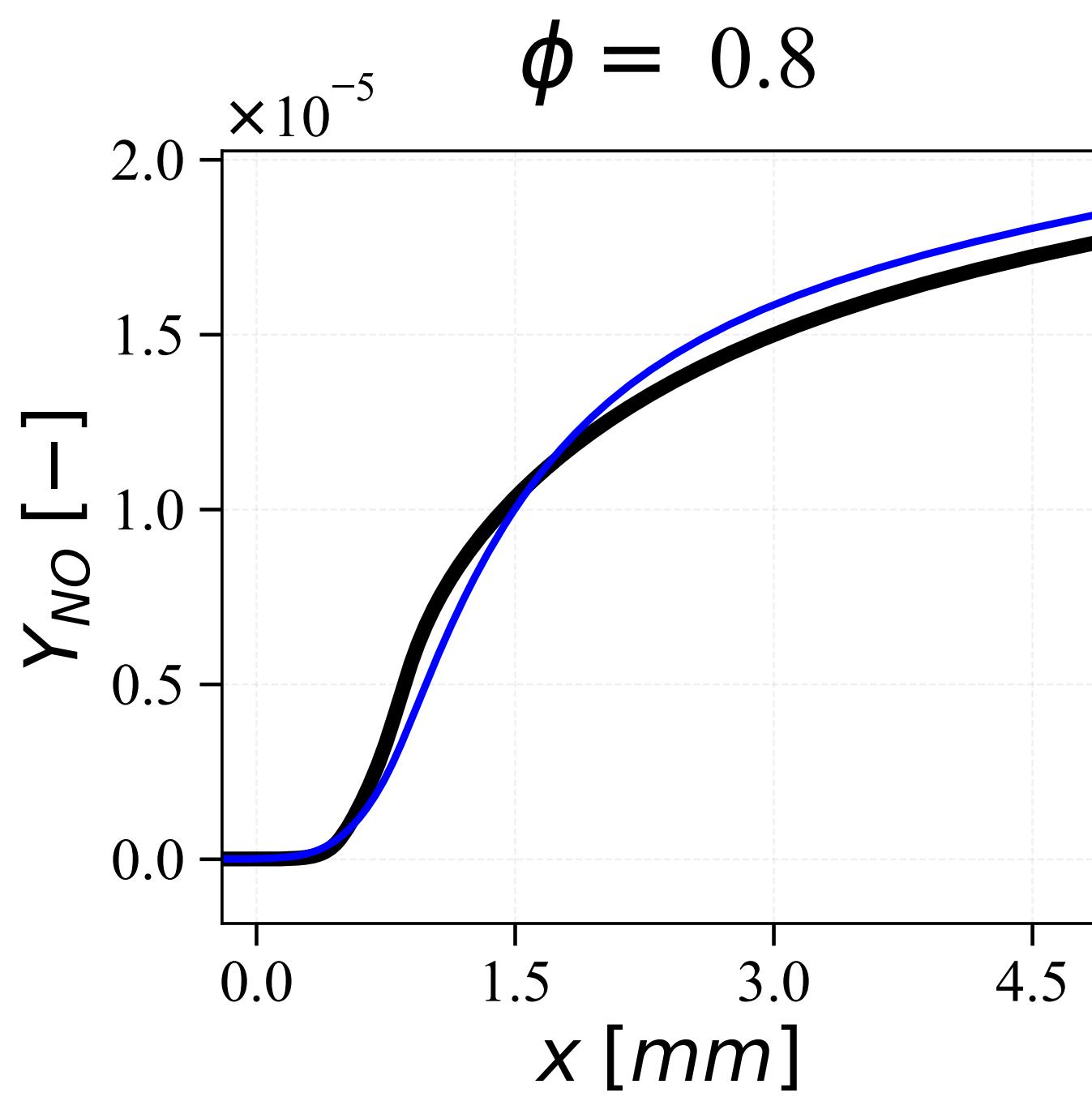
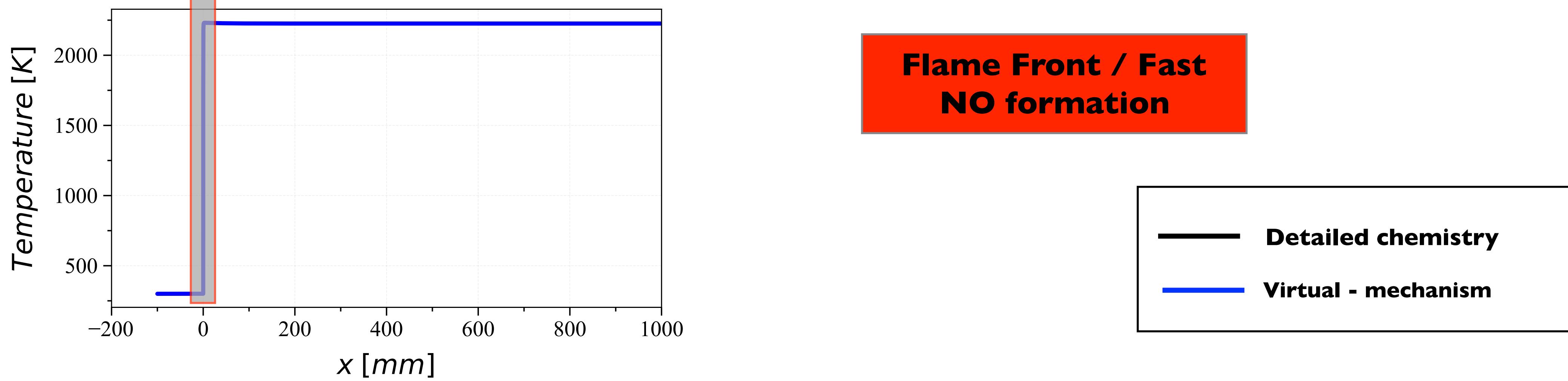
## NO sub-mechanism :

- Challenge : multiple time scale to capture + post-flame reburning effects



# Virtual sub-mechanism for NO prediction

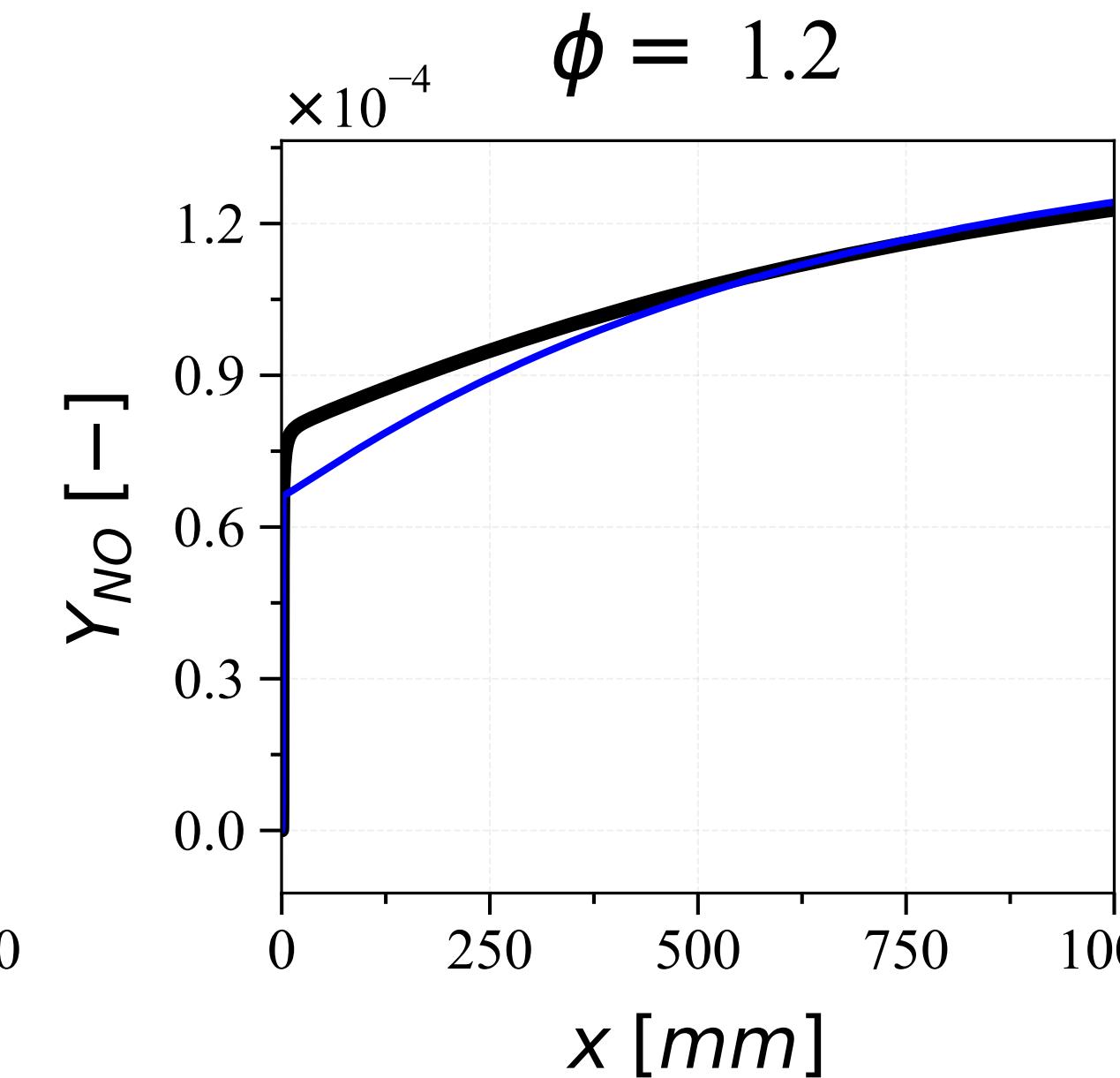
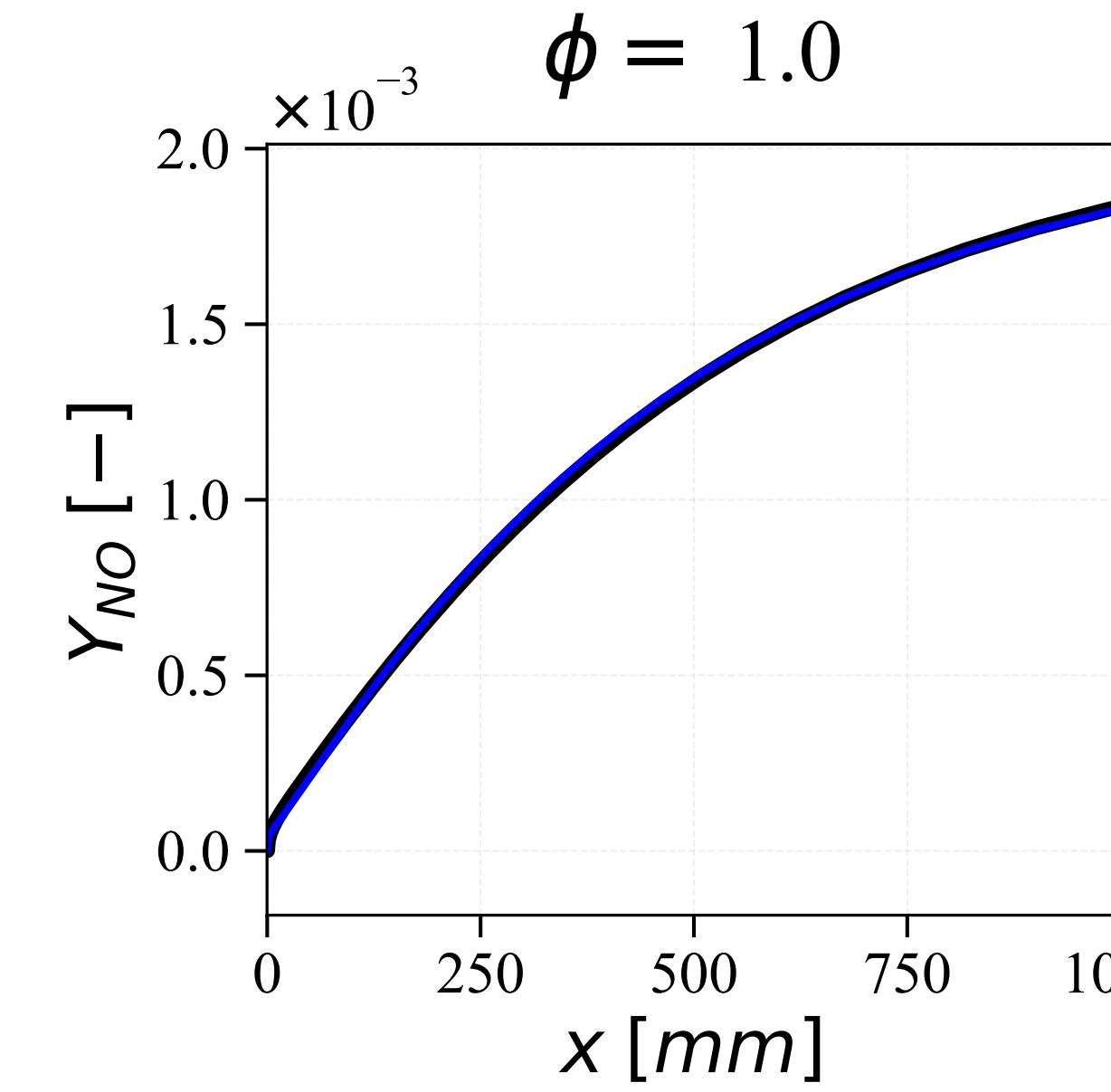
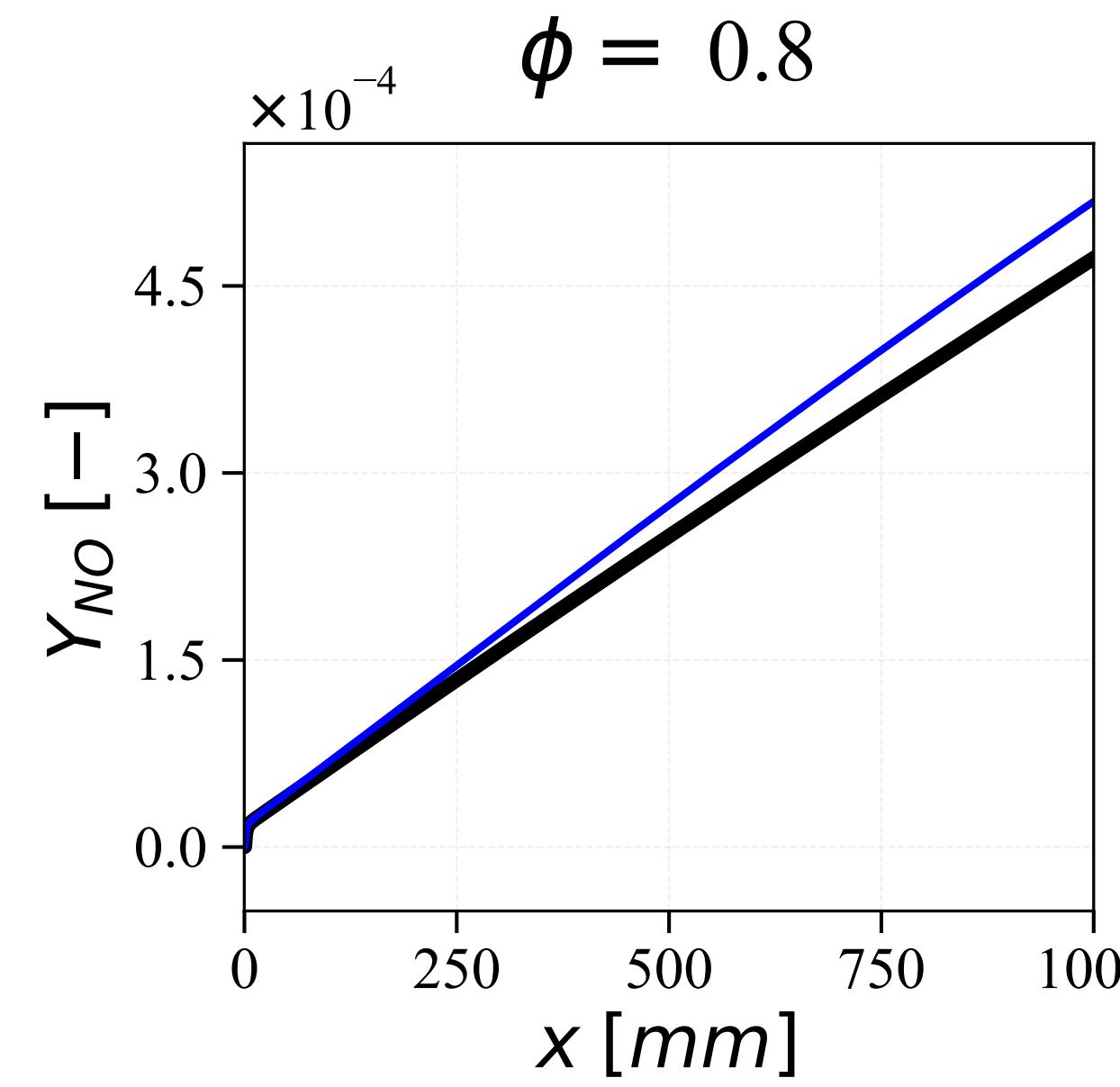
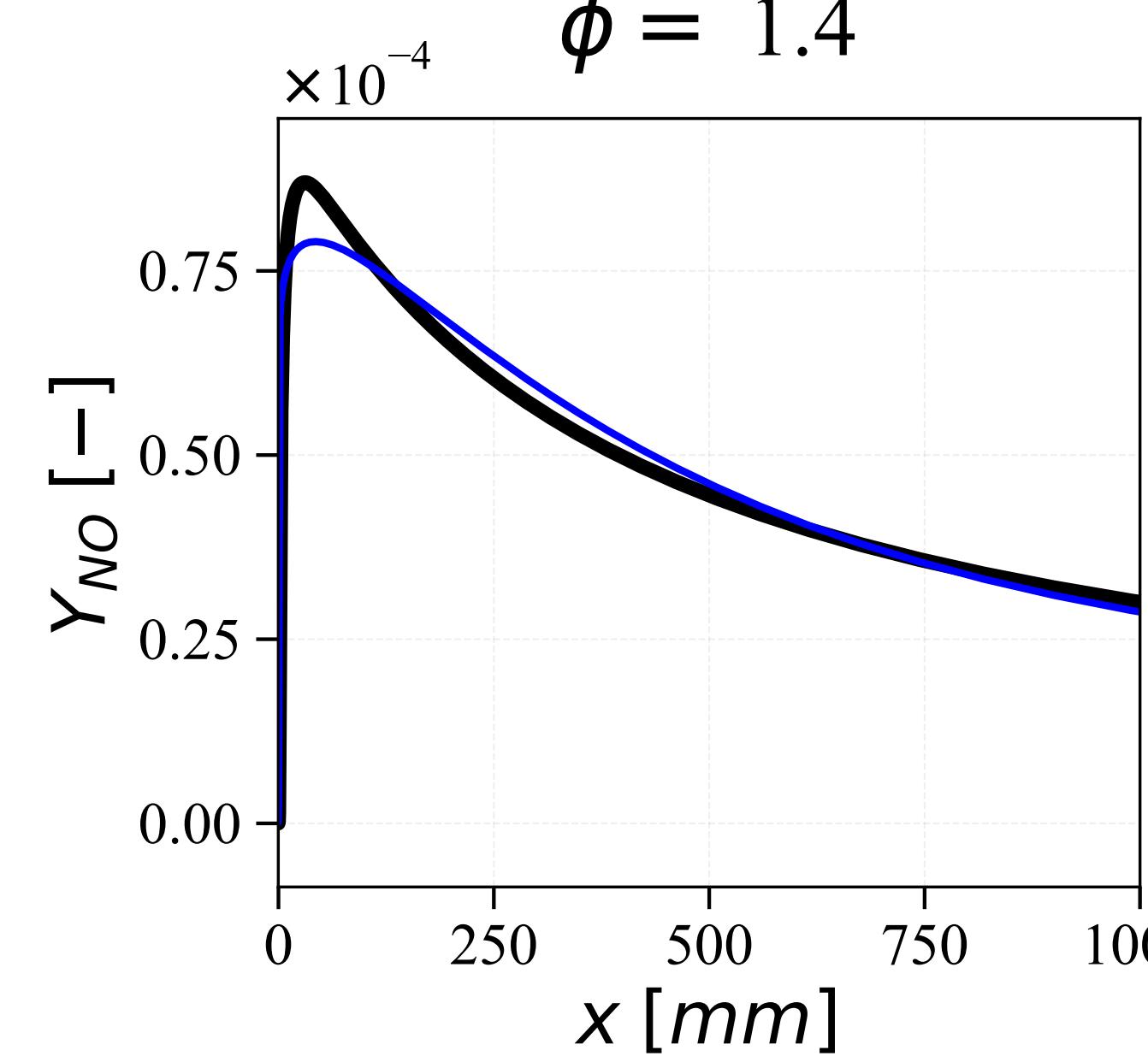
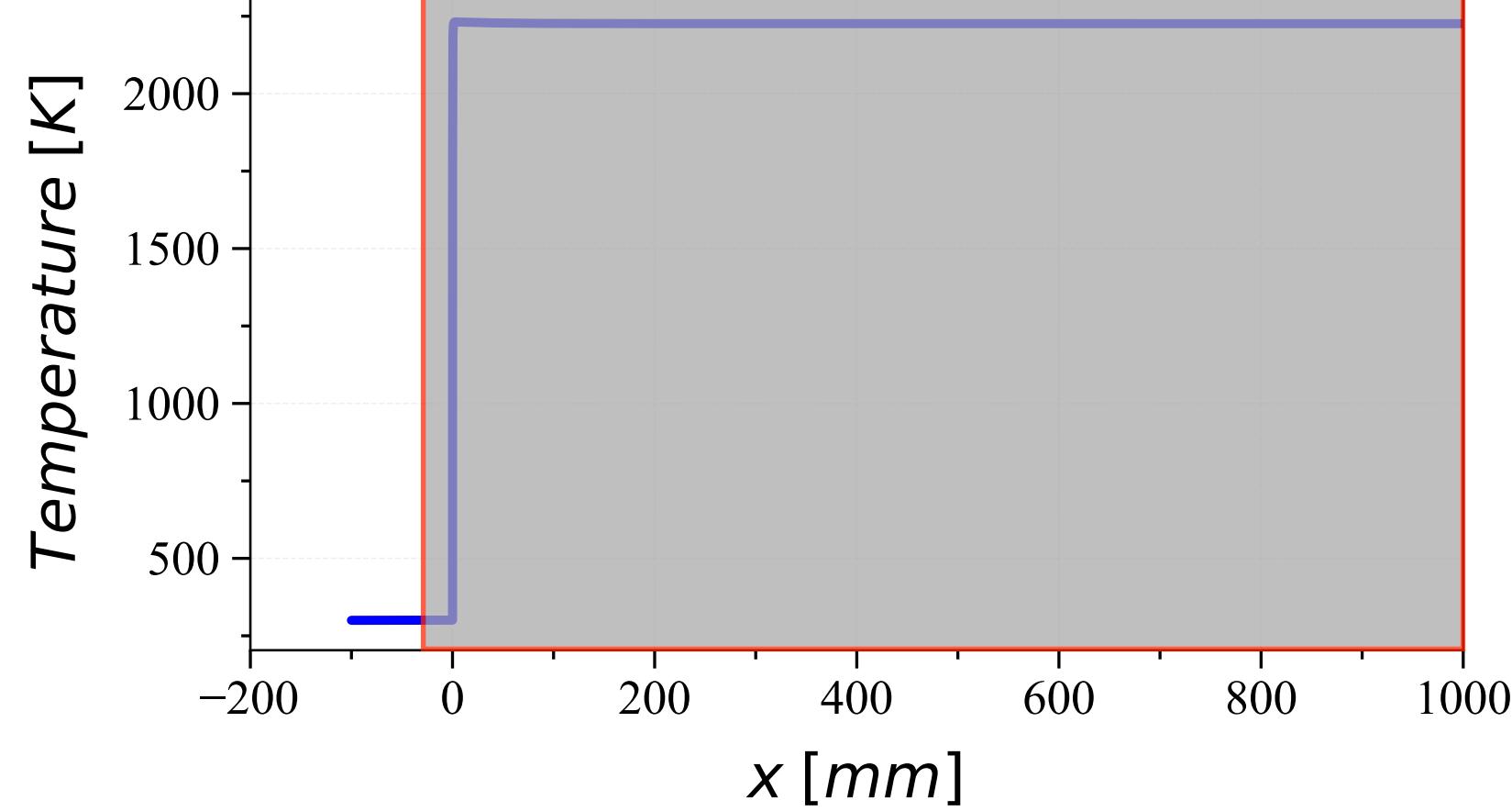
Validation of the results :



# Virtual sub-mechanism for NO prediction

Validation of the results :

**Post Flame / Slow  
NO formation**



**Detailed chemistry**  
**Virtual - mechanism**

# *LES of a non-adiabatic turbulent premixed flame (Preccinsta burner)*

- Test non adiabatic virtual chemistry in a 3-D LES turbulent calculation
  - Turbulent premixed swirled burner (Meier et al., Combust. Flame, 2007)
- Previous numerical works reproduce fairly well flow dynamics as well as the mean flame front position

B. Franzelli, E. Riber, L.Y. Gicquel, T. Poinsot, Combust. Flame 159 (2012) 621–637

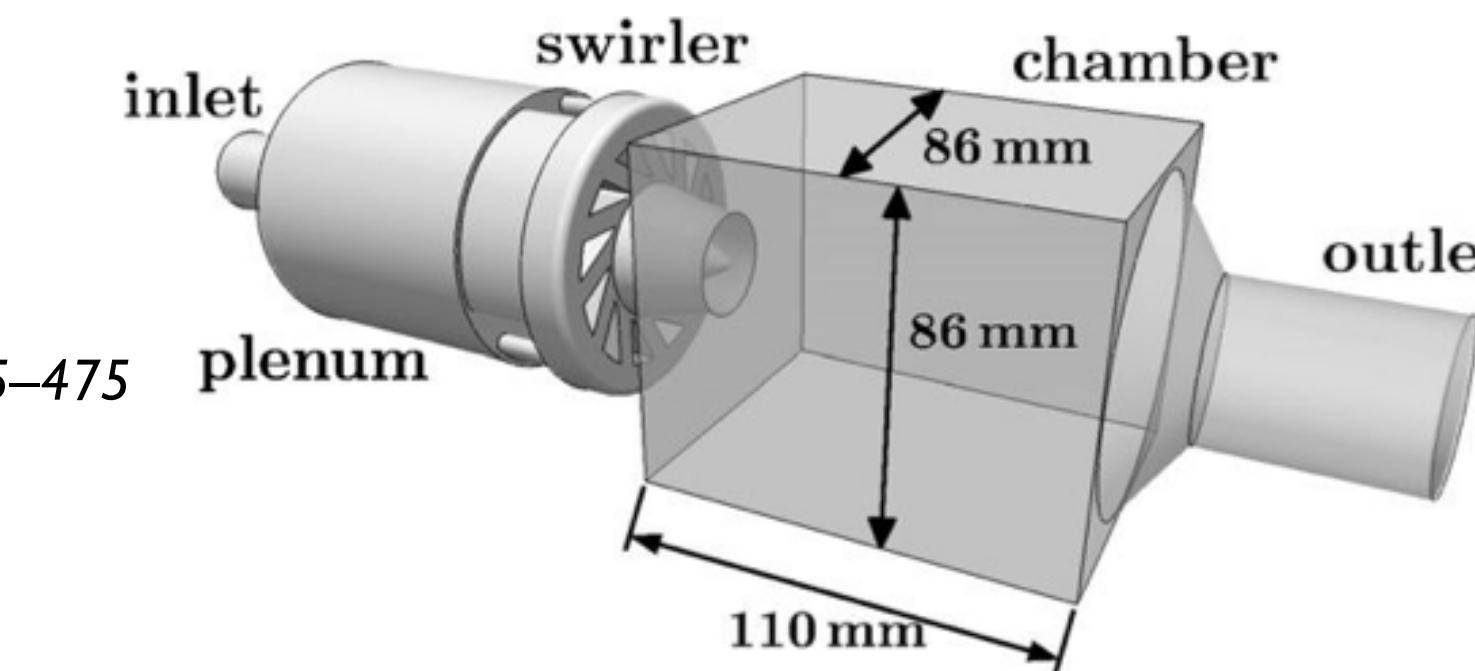
S. Roux, G. Lartigue, T. Poinsot, U. Meier, C. Bérat, Combust. Flame 141 (2005) 40–54

V. Moureau, P. Domingo, L. Vervisch, Combust. Flame 158 (2011) 1340–1357

B. Fiorina, R. Vicquelin, P. Auzillon, N. Darabiha, O. Gicquel, D. Veynante, Combust. Flame 157 (2010) 465–475

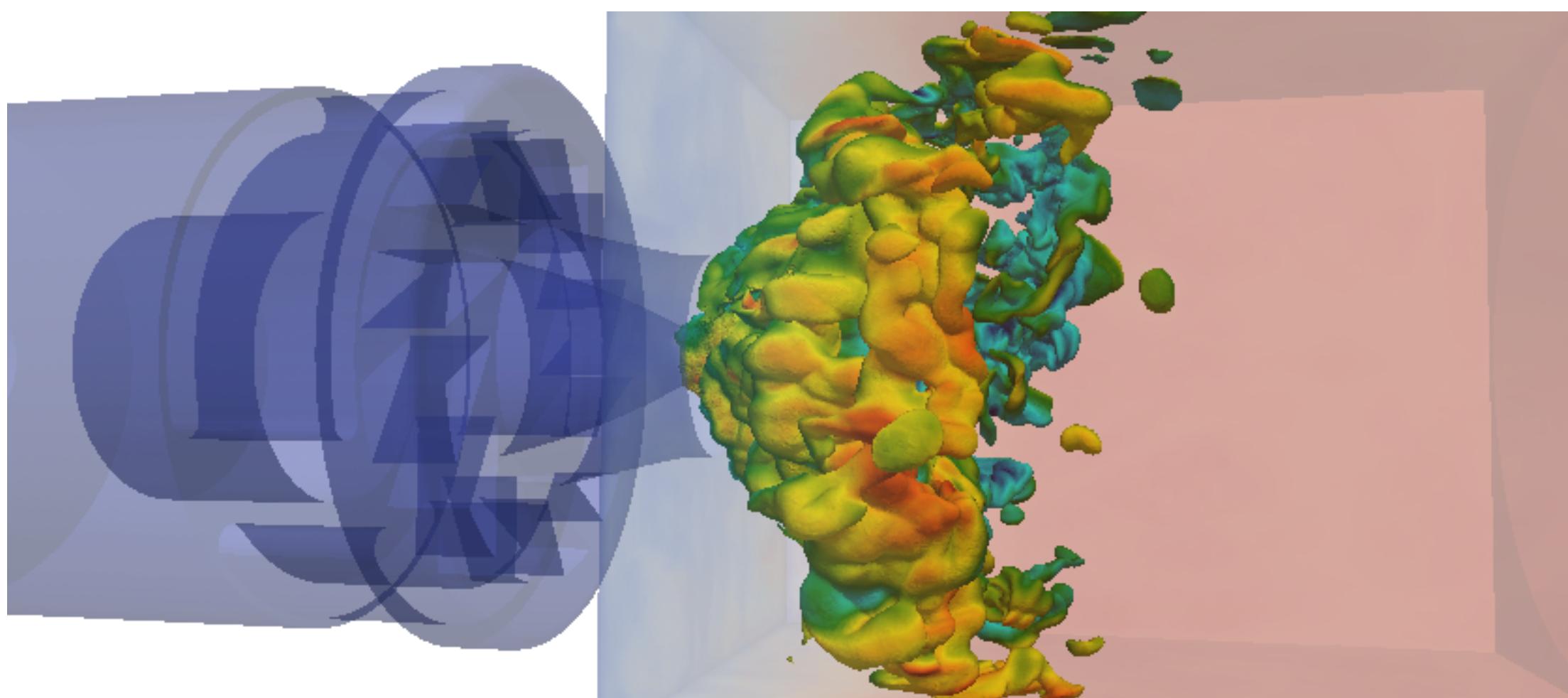
R. Mercier, V. Moureau, D. Veynante, B. Fiorina, Proc. Combust. Inst. 35 (2015) 1359–1366

P. S. Volpiani, T. Schmitt, D. Veynante, Combust. Flame 180 (2017) 124–135



Geometry of the Preccinsta burner  
(Moureau et al., Combust. Flame, 2011)

## Wall heat losses



- But problems for Temperature and CO prediction in the ORZ, attributed to chamber wall heat losses

# *LES of a non-adiabatic turbulent premixed flame (Preccinsta burner)*

## Numerical set-up:

- Wall temperature profile not experimentally provided
- Wall Dirichlet boundary conditions provided by CORIA group (*P.Benard et al., 2018*) by numerical simulation conducted with a reference 17 species skeletal scheme (Sankaran et al. 2007)
- **YALE2 (CORIA V.Moureau et al. 2011)**
  - Combustion chemistry modeled using non adiabatic **virtual mechanisms** (two-step main mechanism and the CO dedicated sub-mechanism )
  - Turbulence/Flame interaction: Thickened Flame model for LES (*Colin et al., 2000*)

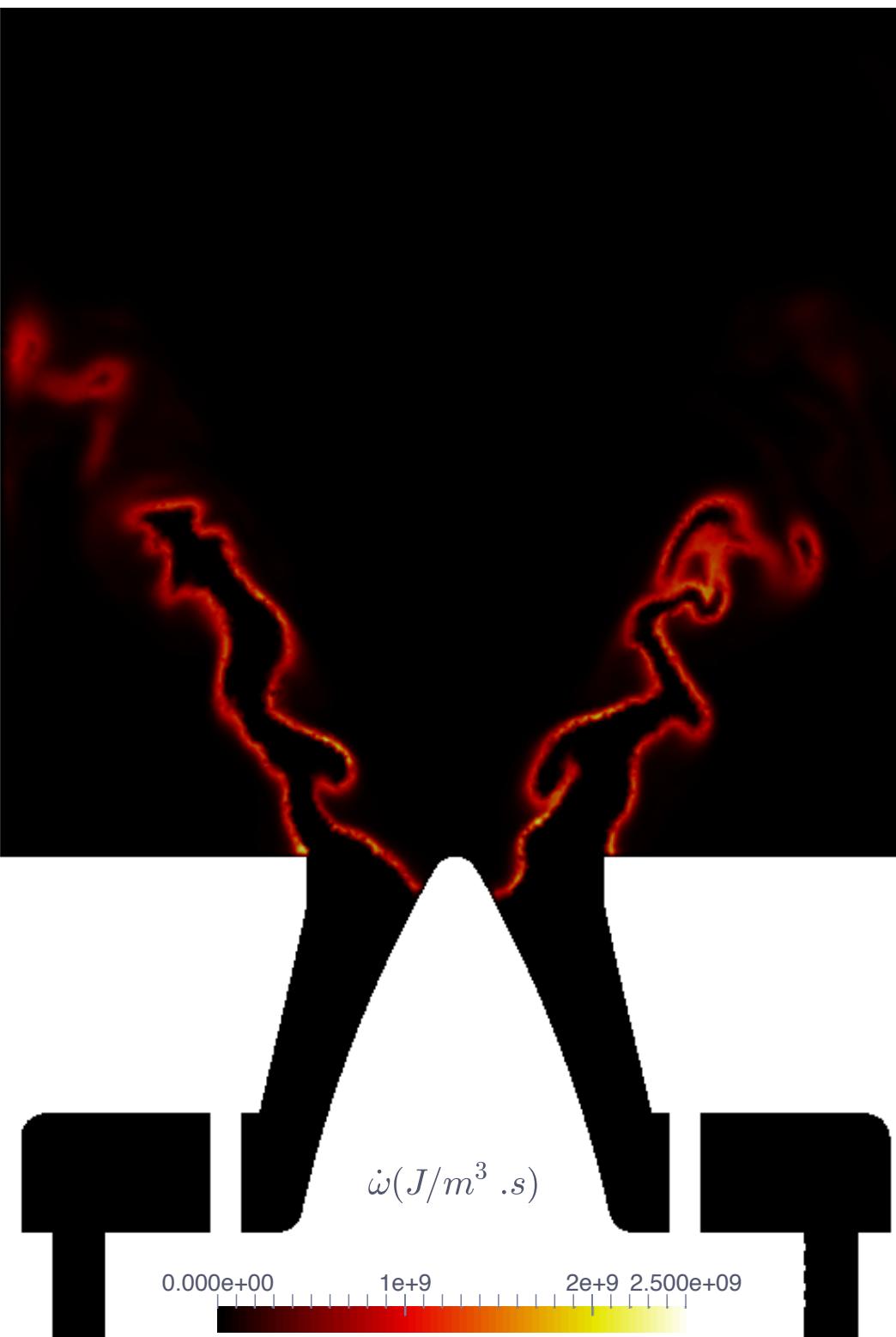
$$\frac{\partial \bar{\rho} \widetilde{Y}_k}{\partial t} + \frac{\partial}{\partial x_i} \left( \bar{\rho} \widetilde{u}_i \widetilde{Y}_k \right) = \frac{\partial}{\partial x_i} \left( \left[ \textcolor{red}{F} \textcolor{blue}{\Xi}_{\Delta} \frac{\mu}{Sc} + (1 - \textcolor{orange}{S}) \frac{\mu_t}{Sc_t} \right] \frac{\partial \widetilde{Y}_k}{\partial x_i} \right) + \frac{\textcolor{blue}{\Xi}_{\Delta}}{\textcolor{red}{F}} \widetilde{\omega}_k$$

- Sub-grid flame wrinkling (*Charlette et al., 2002*) with  $\beta=0.5$ .
- Flame sensor based on virtual species source term. (*Caillet et al. 2017*)

# *LES of a non-adiabatic turbulent premixed flame (Preccinista burner)*

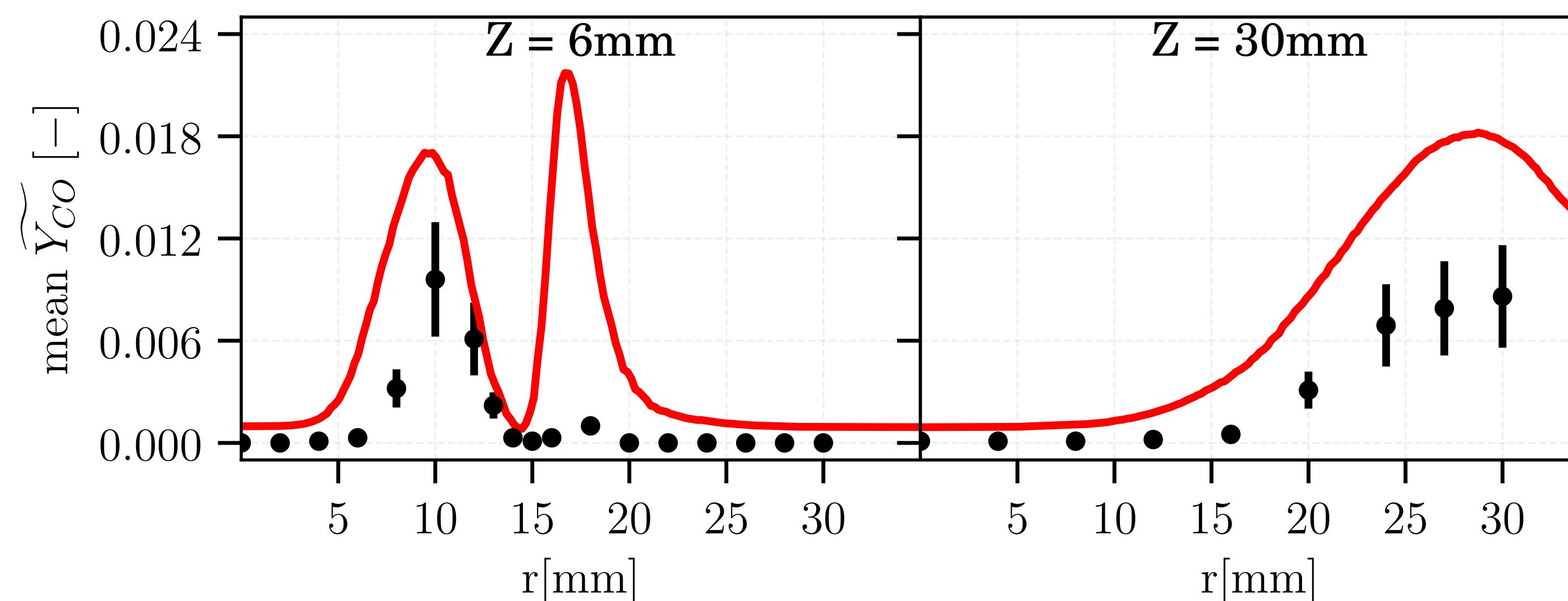
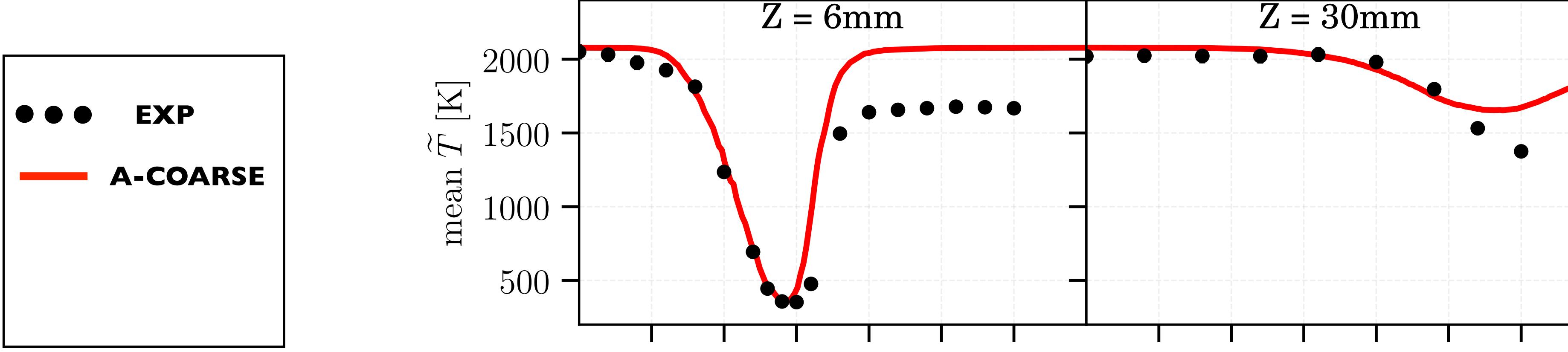
→ Simulation results : Instantaneous Heat release rate

Adiabatic Coarse 2.7M nodes



# *LES of a non-adiabatic turbulent premixed flame (Preccinista burner)*

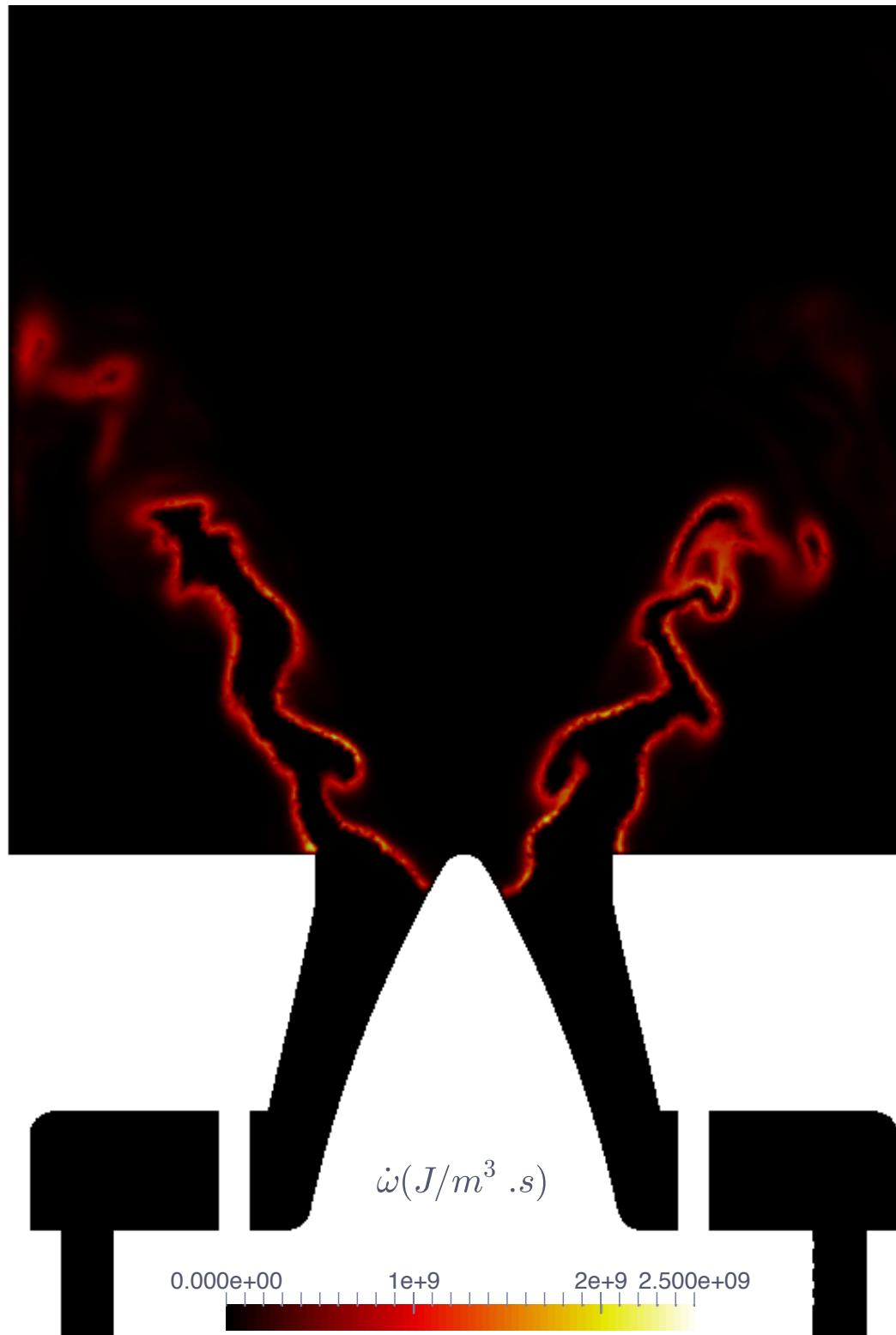
Simulation results : Temperature / CO Reynolds statistics



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→ Simulation results : Instantaneous Heat release rate

Adiabatic Coarse 2.7M nodes

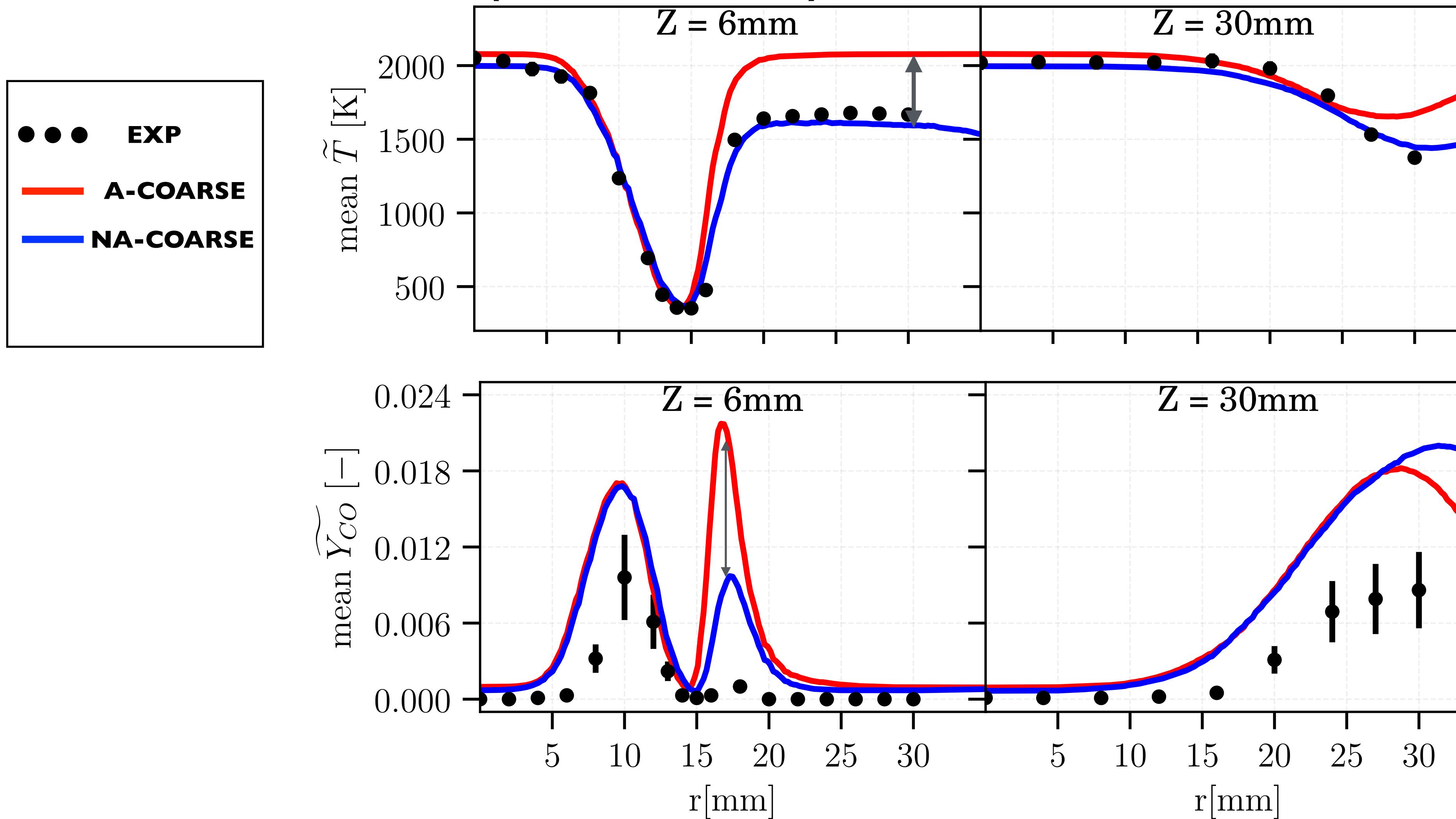


N-Adiabatic Coarse 2.7M nodes



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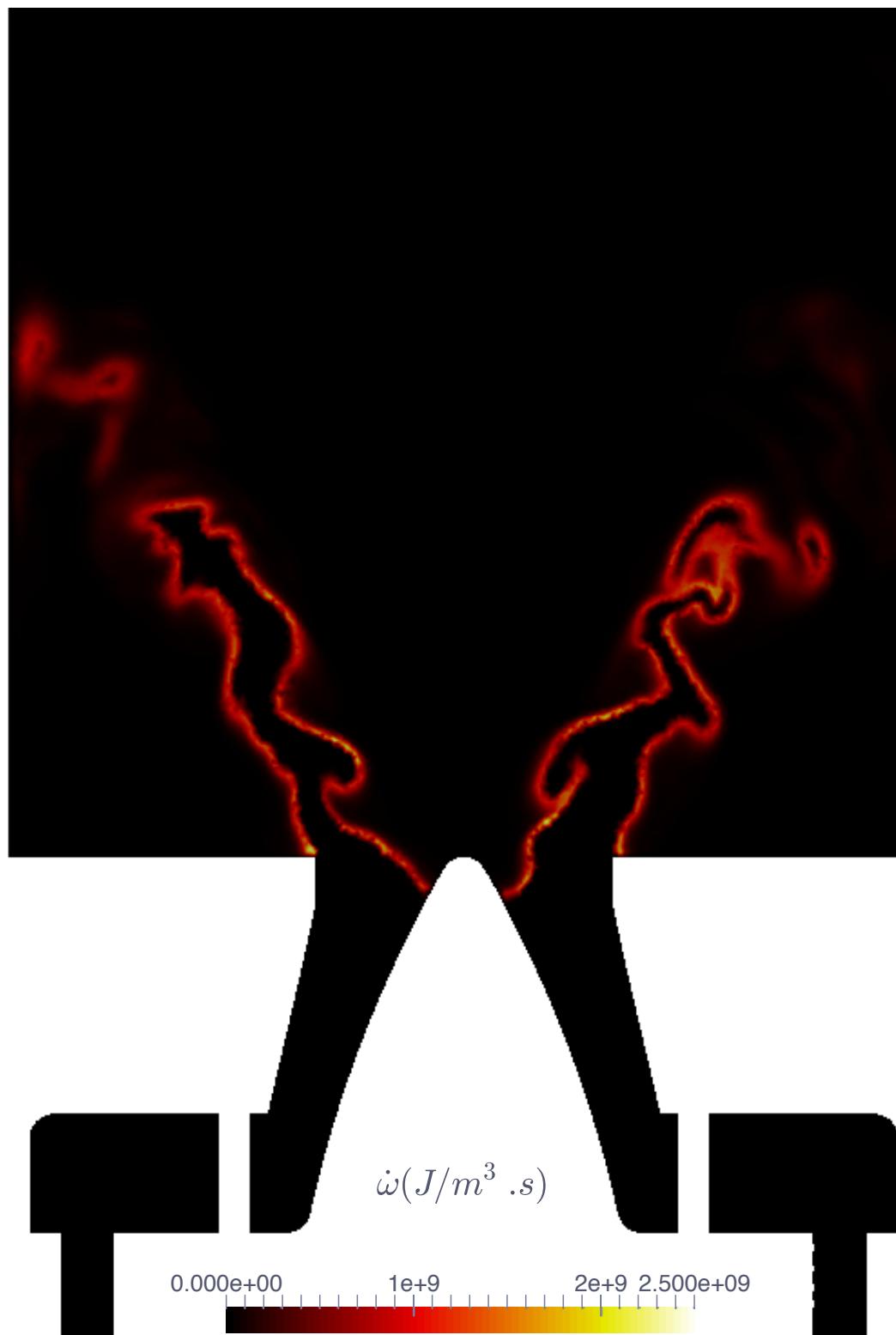
Simulation results : Temperature / CO Reynolds statistics



# *LES of a non-adiabatic turbulent premixed flame (Preccinsta burner)*

→ Simulation results : Instantaneous Heat release rate

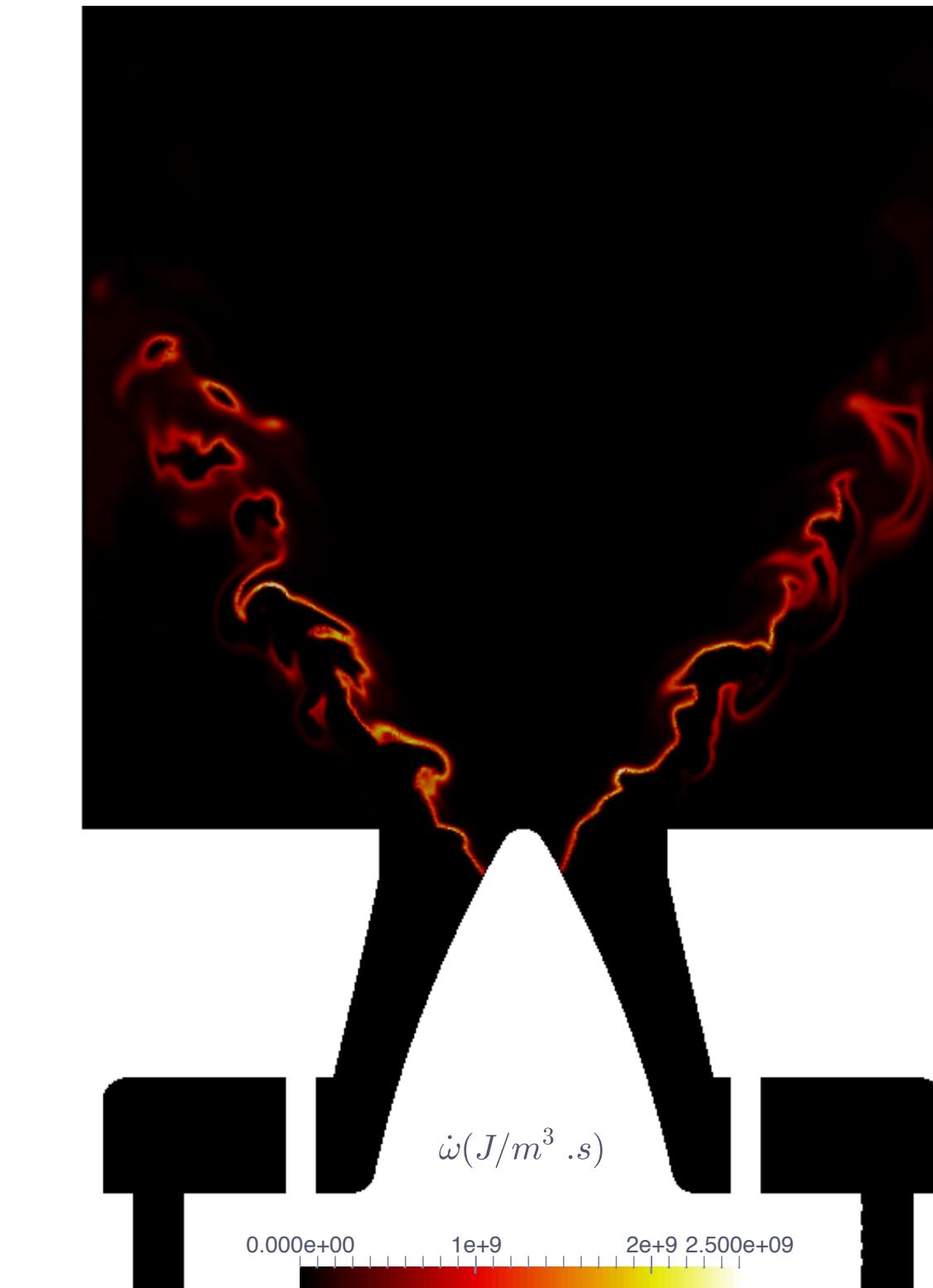
Adiabatic Coarse 2.7M nodes



N-Adiabatic Coarse 2.7M nodes



N-Adiabatic Fine 20.9M nodes

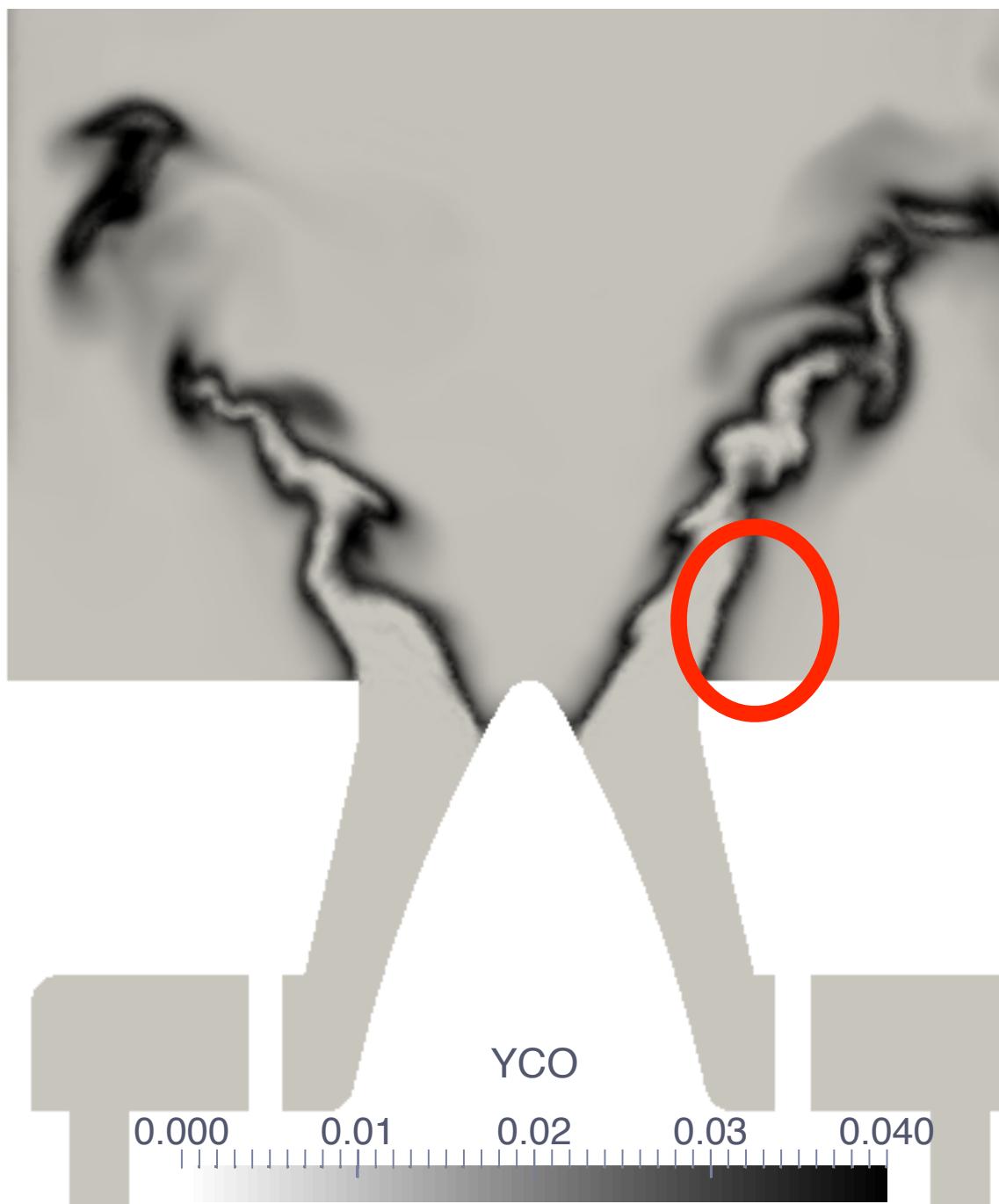


Transition from a M shape flame toward V shape flame due to heat losses and flame resolution improvement

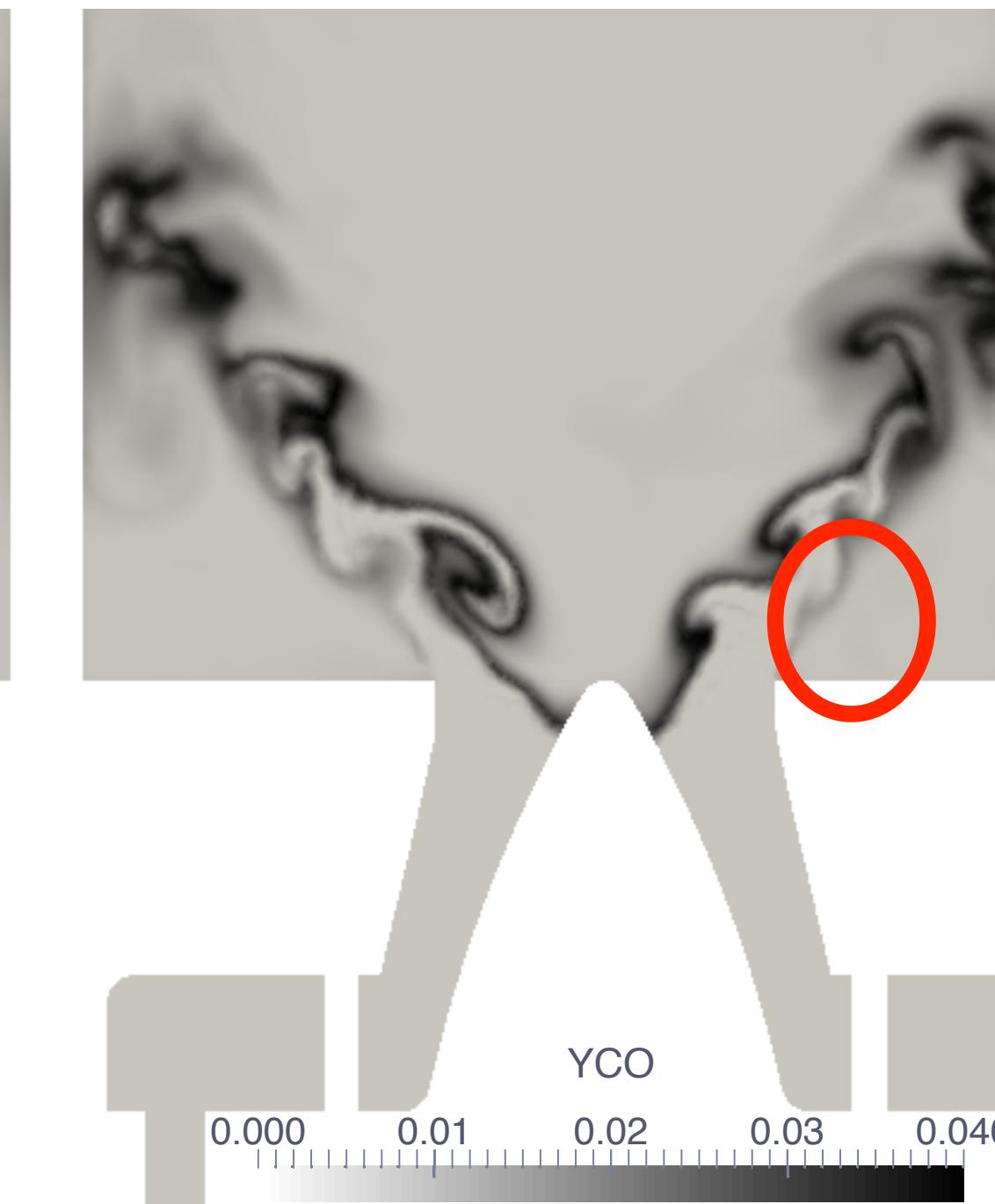
# *LES of a non-adiabatic turbulent premixed flame (Preccinsta burner)*

→ Simulation results : Instantaneous CO filtered mass fraction

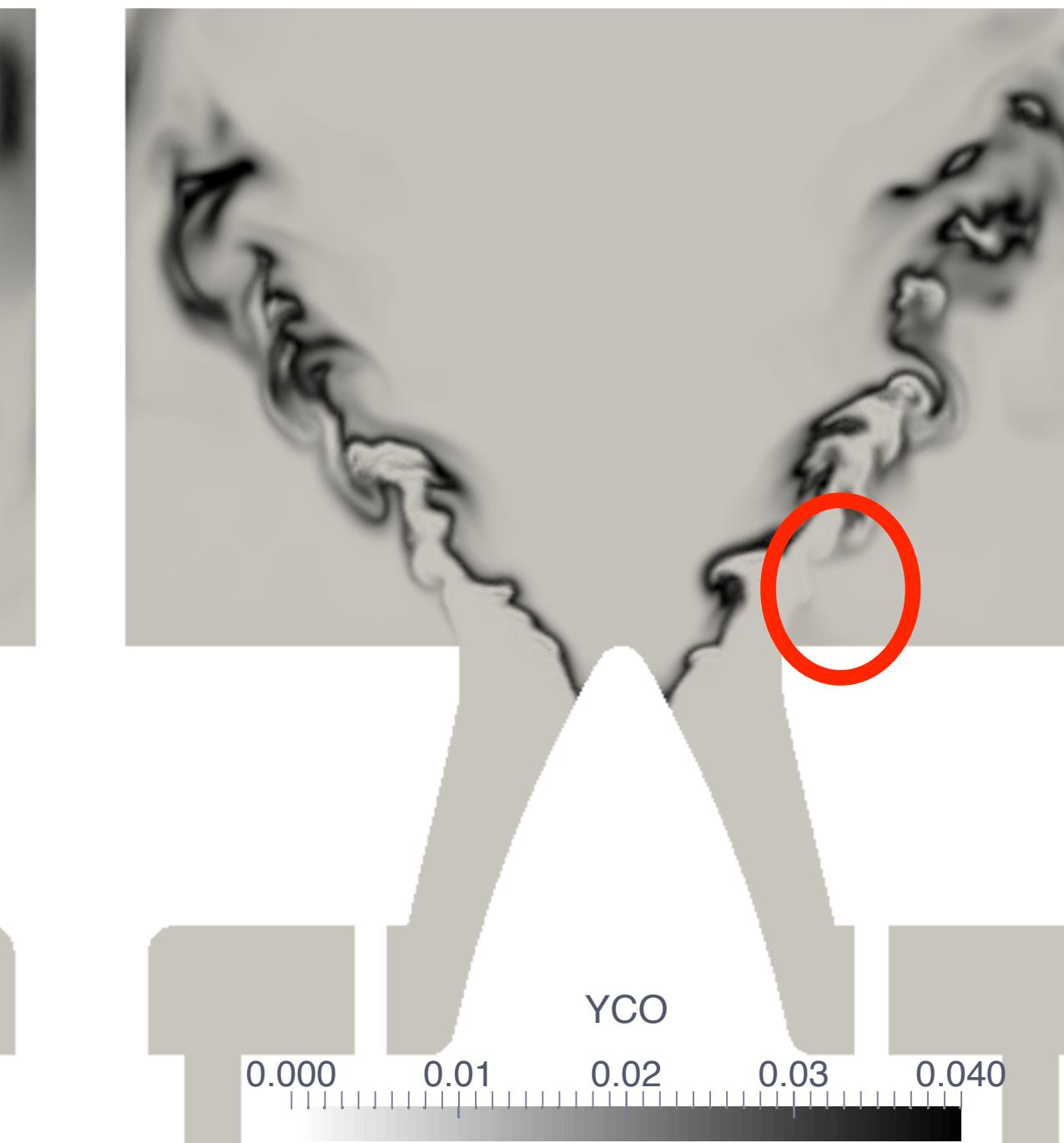
**Adiabatic Coarse 2.7M nodes**



**N-Adiabatic Coarse 2.7M nodes**



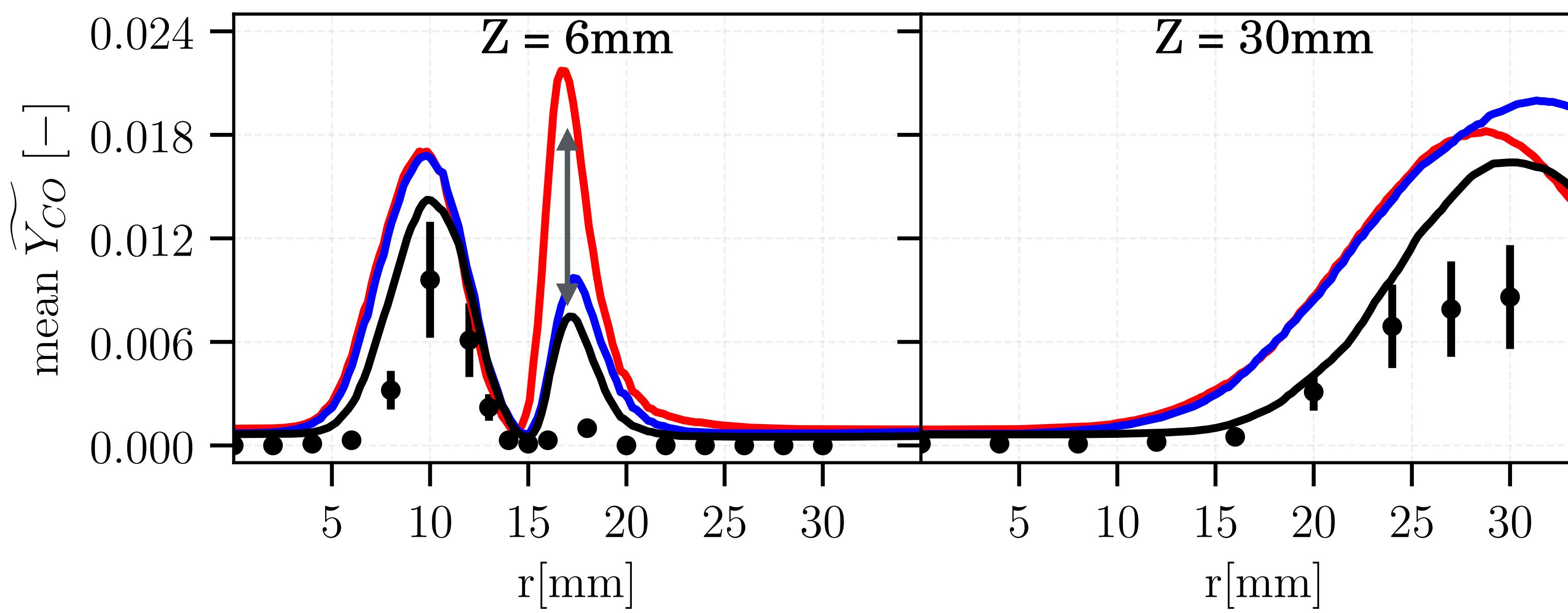
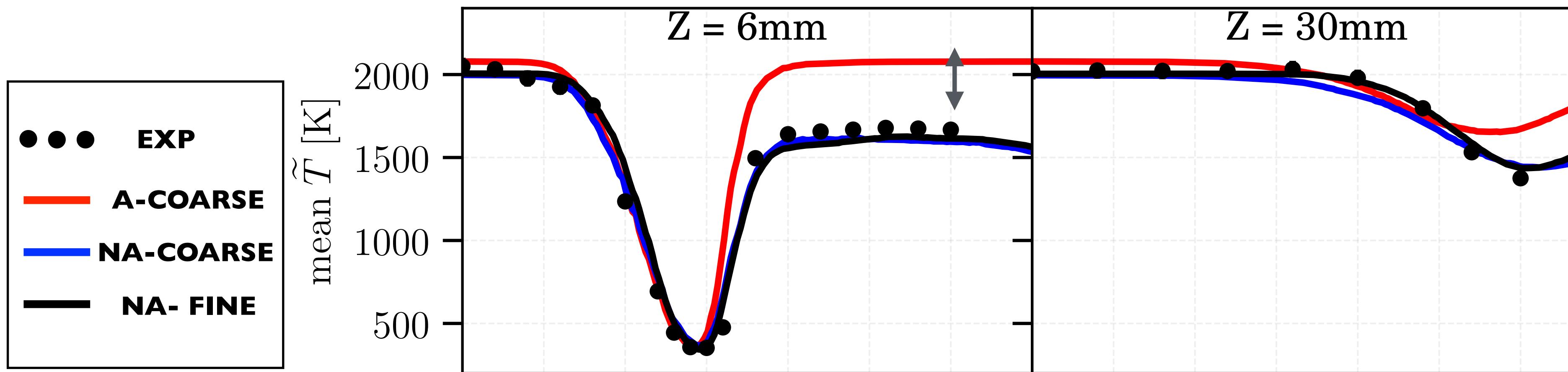
**N-Adiabatic Fine 20.9M nodes**



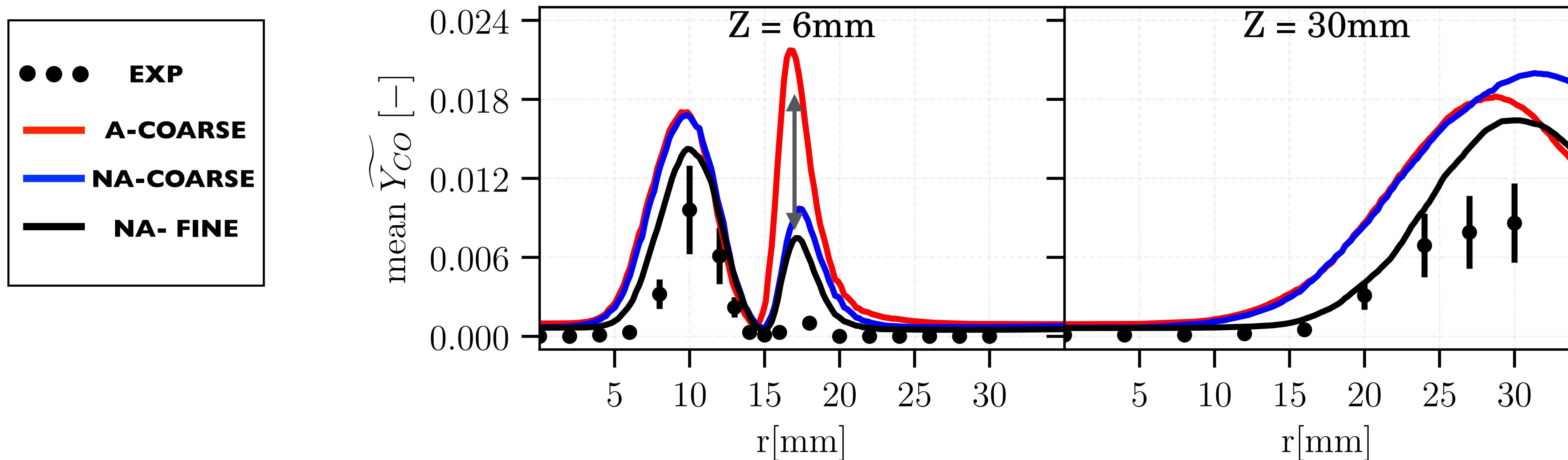
- CO formation impacted by heat losses on the external branch of the flame
- CO is sensitive to mesh resolution.

# *LES of a non-adiabatic turbulent premixed flame (Preccinista burner)*

Simulation results : Temperature / CO Reynolds statistics



# *CO-prediction impacted by the turbulent combustion model (Preccinsta burner)*



- Combined effect of Thickening factor and Efficiency function on intermediate filtered species prediction (for example CO)

Benard, P., Lartigue, G., Moureau, V., and Mercier R., Large-Eddy Simulation of the lean-premixed PRECCINSTA burner with wall heat losses Proceedings of the Combustion Institute, 2018.

R. Mercier, C. Mehl, B. Fiorina and V. Moureau. Filtered Wrinkled Flamelets model for Large-Eddy Simulation of turbulent premixed combustion, Submitted to Combustion and Flame(2018).

## *Conclusion*

- Reduced combustion chemistry model that allows pollutants prediction (CO / NO)
- Virtual chemistry validated in the LES context including heat losses
- Mis-prediction of CO attributed to the turbulent combustion model
- The size of the virtual chemistry mechanisms does not change if the complexity of the system increases:
  - heat losses
  - multiple flame regimes
  - different fuels

# **Thank you for your attention**



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